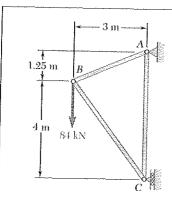
# CHAPTER 6





Using the method of joints, determine the force in each member of the truss shown. State whether each member is in tension or compression.

#### **SOLUTION**

$$AB = \sqrt{3^2 + 1.25^2} = 3.25 \text{ m}$$
  
 $BC = \sqrt{3^2 + 4^2} = 5 \text{ m}$ 

Reactions:

+)
$$\Sigma M_A = 0$$
: (84 kN)(3 m) –  $C(5.25 \text{ m}) = 0$ 

$$+\Sigma F_x=0$$
:  $A_x-C=0$ 

$$A_r = 48 \text{ kN} \longrightarrow$$

$$+ \sum F_y = 0$$
:  $A_y = 84 \text{ kN} = 0$ 

$$A_y = 84 \text{ kN}$$

Joint A:

$$\pm \Sigma F_x = 0$$
:  $48 \text{ kN} - \frac{12}{13} F_{AB} = 0$ 

$$F_{AB} = +52 \text{ kN}$$

$$+1 \Sigma F_y = 0$$
: 84 kN  $-\frac{5}{13}$  (52 kN)  $-F_{AC} = 0$ 

$$F_{AC} = +64.0 \text{ kN}$$

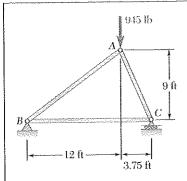
$$F_{AC} = 64.0 \, \text{kN}$$
 T

 $F_{AB} = 52 \text{ kN}$   $T \blacktriangleleft$ 

Joint C:

$$\frac{F_{BC}}{5} = \frac{48 \text{ kN}}{3}$$

$$F_{RC} = 80.0 \text{ kN}$$
 C



Using the method of joints, determine the force in each member of the truss shown. State whether each member is in tension or compression.

#### SOLUTION

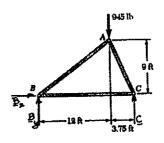
Free body: Entire truss

$$F_x = 0$$
:  $F_x = 0$ :

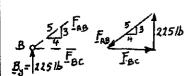
$$C = 720 \text{ lb}$$

$$+ \sum F_v = 0$$
:  $B_v + 720 \text{ lb} - 945 \text{ lb} = 0$ 

$$B_v = 225 \, lb$$



Free body: Joint B:



$$\frac{F_{AB}}{5} = \frac{F_{BC}}{4} = \frac{225 \text{ lb}}{3}$$

$$F_{AB} = 375 \, \text{lb} \quad C \blacktriangleleft$$

$$F_{BC} = 300 \, \text{lb}$$
  $T \blacktriangleleft$ 

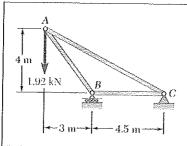
Free body: Joint C:



$$\frac{F_{AC}}{9.75} = \frac{F_{BC}}{3.75} = \frac{720 \text{ lb}}{9}$$

$$F_{BC} = 300 \text{ lb}$$
  $T$  (Checks)

$$F_{AC} = 780 \text{ lb}$$
 C



Using the method of joints, determine the force in each member of the truss shown. State whether each member is in tension or compression.

#### SOLUTION

Free body: Entire truss

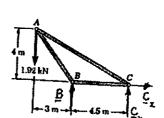
$$+\Sigma F_x = 0$$
:  $C_x = 0$   $C_x = 0$ 

+)
$$\Sigma M_B = 0$$
:  $(1.92 \text{ kN})(3 \text{ m}) + C_y(4.5 \text{ m}) = 0$ 

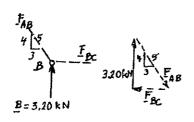
$$C_y = -1.28 \text{ kN}$$
  $C_y = 1.28 \text{ kN}$ 

$$+ \sum F_y = 0$$
:  $B - 1.92 \text{ kN} - 1.28 \text{ kN} = 0$ 

$$B = 3.20 \text{ kN}$$



Free body: Joint B:



$$\frac{F_{AB}}{5} = \frac{F_{BC}}{3} = \frac{3.20 \text{ kN}}{4}$$

$$F_{AB} = 4.00 \,\mathrm{kN}$$
 C

$$F_{RC} = 2.40 \text{ kN}$$
 C

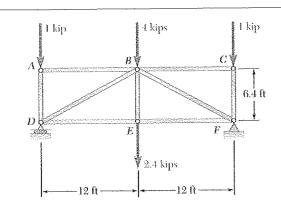
Free body: Joint C:

$$+ \Sigma F_x = 0$$
:  $-\frac{7.5}{8.5} F_{AC} + 2.40 \text{ kN} = 0$ 

$$F_{AC} = +2.72 \text{ kN}$$

$$F_{AC} = +2.72 \text{ kN}$$
  $F_{AC} = 2.72 \text{ kN}$   $T$ 

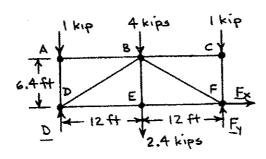
+ 
$$\Sigma F_y = \frac{4}{8.5} (2.72 \text{ kN}) - 1.28 \text{ kN} = 0 \text{ (Checks)}$$



Using the method of joints, determine the force in each member of the truss shown. State whether each member is in tension or compression.

# **SOLUTION**

#### Reactions:



$$+\Sigma M_D = 0$$
:  $F_v(24) - (4 + 2.4)(12) - (1)(24) = 0$ 

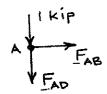
$$\mathbf{F}_{v} = 4.2 \text{ kips}$$

$$\Sigma F_{\rm v} = 0$$
:  $\mathbf{F}_{\rm v} = 0$ 

$$+\int \Sigma F_{\nu} = 0$$
:  $D - (1 + 4 + 1 + 2.4) + 4.2 = 0$ 

$$\mathbf{D} = 4.2 \text{ kips}$$

Joint A:



$$\Sigma F_x = 0: \quad F_{AB} = 0$$

$$F_{AB} = 0$$

$$+ \sum F_y = 0: -1 - F_{AD} = 0$$

$$F_{AD} = -1 \text{ kip}$$

$$F_{AD} = 1.000 \text{ kip} \ C \blacktriangleleft$$

Joint D:

$$+ \sum F_y = 0$$
:  $-1 + 4.2 + \frac{8}{17} F_{BD} = 0$ 

$$F_{BD} = -6.8 \text{ kips}$$

$$F_{RD} = 6.80 \text{ kips}$$
 C

$$\pm \Sigma F_x = 0$$
:  $\frac{15}{17}(-6.8) + F_{DE} = 0$ 

$$F_{DE} = +6 \text{ kips}$$

$$F_{DE} = 6.00 \text{ kips}$$
  $T \blacktriangleleft$ 

# PROBLEM 6.4 (Continued)

Joint E:

$$F_{DE} = 6 \text{ kips} \qquad F_{EF}$$

$$2.4 \text{ kips}$$

$$+ \sum_{BE} F_{BE} = 12.4 \text{ kips}$$

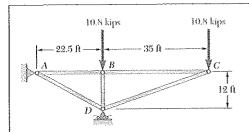
$$F_{BE} = 12.4 \text{ kips}$$

$$+ \sum F_v = 0$$
:  $F_{BE} - 2.4 = 0$ 

$$F_{BE} = +2.4 \text{ kips}$$

 $F_{BE} = 2.40 \text{ kips}$   $T \blacktriangleleft$ 

Truss and loading symmetrical about &



Using the method of joints, determine the force in each member of the truss shown. State whether each member is in tension or compression.

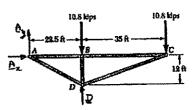
#### **SOLUTION**

Free body: Truss

$$\pm \Sigma F_{\rm r} = 0$$
:  $\mathbf{A}_{\rm r} = 0$ 

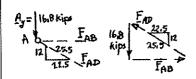
+)
$$\Sigma M_A = 0$$
:  $D(22.5) - (10.8 \text{ kips})(22.5) - (10.8 \text{ kips})(57.5) = 0$ 

 $D = 38.4 \text{ kips } \uparrow$ 



 $\Sigma F_y = 0$ :  $\mathbf{A}_y = 16.8 \text{ kips}$ 

Free body: Joint A:



$$\frac{F_{AB}}{22.5} = \frac{F_{AD}}{25.5} = \frac{16.8 \text{ kips}}{12}$$

 $F_{AB} = 31.5 \text{ kips}$   $T \blacktriangleleft$ 

 $F_{AD} = 35.7 \text{ kips}$   $C \blacktriangleleft$ 

Free body: Joint B:

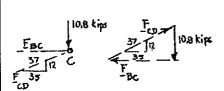
$$\Sigma F_{\rm r} = 0$$
:

 $F_{BC} = 31.5 \text{ kips} \quad T \blacktriangleleft$ 

$$\Sigma F_v = 0$$
:

 $F_{BD} = 10.80 \text{ kips}$  C

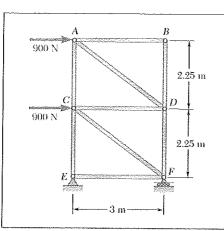
Free body: Joint C:



$$\frac{F_{CD}}{37} = \frac{F_{BC}}{35} = \frac{10.8 \text{ kips}}{12}$$

 $F_{CD} = 33.3 \text{ kips}$  C

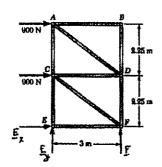
 $F_{BC} = 31.5 \text{ kips}$  T (Checks)



Using the method of joints, determine the force in each member of the truss shown. State whether each member is in tension or compression.

#### **SOLUTION**

Free Body: Truss



+)
$$\Sigma M_E = 0$$
:  $F(3 \text{ m}) - (900 \text{ N})(2.25 \text{ m}) - (900 \text{ N})(4.5 \text{ m}) = 0$ 

$$F = 2025 N$$

$$\pm \Sigma F_x = 0$$
:  $E_x + 900 \text{ N} + 900 \text{ N} = 0$ 

$$E_x = -1800 \text{ N}$$
  $E_x = 1800 \text{ N}$ 

$$+ \sum F_y = 0$$
:  $E_y + 2025 \text{ N} = 0$ 

$$E_y = -2025 \text{ N}$$
  $\mathbf{E}_y = 2025 \text{ N}$ 

We note that AB and BD are zero-force members:

$$F_{AB} = F_{BD} = 0$$

Free body: Joint A:





$$\frac{F_{AC}}{2.25} = \frac{F_{AD}}{3.75} = \frac{900 \text{ N}}{3}$$

$$F_{AC} = 675 \,\mathrm{N}$$
  $T \blacktriangleleft$ 

$$F_{AD} = 1125 \text{ N} \quad C \blacktriangleleft$$

Free body: Joint D:





$$\frac{F_{CD}}{3} = \frac{F_{DE}}{2.23} = \frac{1125 \text{ N}}{3.75}$$

$$F_{CD} = 900 \text{ N}$$
 T

$$F_{DF} = 675 \,\mathrm{N}$$
  $C \blacktriangleleft$ 

Free body: Joint E:

$$\pm \Sigma F_x = 0$$
:  $F_{EF} - 1800 \text{ N} = 0$ 

$$F_{EF} = 1800 \text{ N}$$
  $T \blacktriangleleft$ 

$$+ \sum F_y = 0$$
:  $F_{CE} - 2025 \text{ N} = 0$ 

$$F_{CE} = 2025 \text{ N}$$
 T

# PROBLEM 6.6 (Continued)

Free body: Joint *F*:

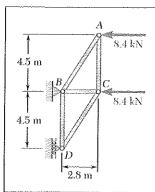


+ 
$$\Sigma F_y = 0$$
:  $\frac{2.25}{3.75} F_{CF} + 2025 \text{ N} - 675 \text{ N} = 0$ 

$$F_{cr} = -2250 \text{ N}$$

$$F_{CF} = -2250 \text{ N}$$
  $F_{CF} = 2250 \text{ N}$   $C \blacktriangleleft$ 

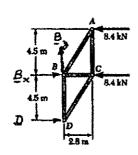
$$\pm \Sigma F_x = -\frac{3}{3.75} (-2250 \text{ N}) - 1800 \text{ N} = 0 \text{ (Checks)}$$



Using the method of joints, determine the force in each member of the truss shown. State whether each member is in tension or compression.

#### SOLUTION

Free body: Truss



$$+ \sum F_y = 0: \quad \mathbf{B}_y = 0$$

+)
$$\Sigma M_B = 0$$
:  $D(4.5 \text{ m}) + (8.4 \text{ kN})(4.5 \text{ m}) = 0$ 

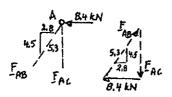
$$D = -8.4 \text{ kN}$$

 $\mathbf{D} = 8.4 \, \mathrm{kN} \, \blacktriangleleft$ 

$$+ \Sigma F_x = 0$$
:  $B_x - 8.4 \text{ kN} - 8.4 \text{ kN} - 8.4 \text{ kN} = 0$ 

$$B_x = +25.2 \text{ kN}$$
  $B_y = 25.2 \text{ kN} \longrightarrow$ 

Free body: Joint A:

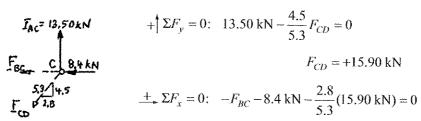


$$\frac{F_{AB}}{5.3} = \frac{F_{AC}}{4.5} = \frac{8.4 \text{ kN}}{2.8}$$

 $F_{AB} = 15.90 \text{ kN}$  C

$$F_{AC} = 13.50 \,\text{kN}$$
 T

Free body: Joint C:



$$+\int \Sigma F_y = 0$$
: 13.50 kN  $-\frac{4.5}{5.3}F_{CD} = 0$ 

$$F_{CD} = +15.90 \text{ kN}$$

 $F_{CD} = 15.90 \text{ kN}$  T

$$\pm \Sigma F_x = 0$$
:  $-F_{BC} - 8.4 \text{ kN} - \frac{2.8}{5.3} (15.90 \text{ kN}) = 0$ 

 $F_{BC} = -16.80 \text{ kN}$   $F_{BC} = 16.80 \text{ kN}$   $C \blacktriangleleft$ 

# **PROBLEM 6.7 (Continued)**

Free body: Joint D:

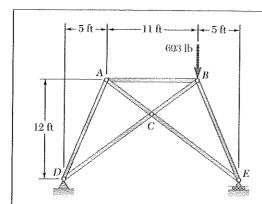
$$\frac{F_{BD}}{4.5} = \frac{8.4 \text{ kN}}{2.8}$$

$$F_{BD} = 13.50 \,\mathrm{kN}$$
  $C$ 

We can also write the proportion

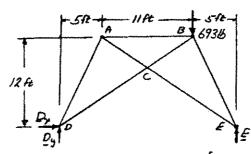
$$\frac{F_{BD}}{4.5} = \frac{15.90 \text{ kN}}{5.3}$$

$$F_{BD} = 13.50 \text{ kN}$$
 C (Checks)



Using the method of joints, determine the force in each member of the truss shown. State whether each member is in tension or compression.

#### **SOLUTION**



$$AD = \sqrt{5^2 + 12^2} = 13 \text{ ft}$$

$$BCD = \sqrt{12^2 + 16^2} = 20 \text{ ft}$$

$$BCD = \sqrt{12^2 + 16^2} = 20 \text{ ft}$$

Reactions:

$$\Sigma F_x = 0$$
:  $D_x = 0$ 

+) 
$$\Sigma M_E = 0$$
:  $D_y(21 \text{ ft}) - (693 \text{ lb})(5 \text{ ft}) = 0$   $\mathbf{D}_y = 165 \text{ lb}$ 

$$+ \sum F_y = 0$$
: 165 lb - 693 lb +  $E = 0$  **E** = 528 lb

Joint D:

$$\frac{+}{2} \Sigma F_x = 0: \quad \frac{5}{13} F_{AD} + \frac{4}{5} F_{DC} = 0 \tag{1}$$

$$+ \sum F_y = 0$$
:  $\frac{12}{13} F_{AD} + \frac{3}{5} F_{DC} + 165 \text{ lb} = 0$  (2)

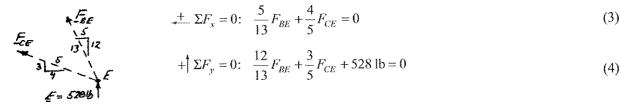
Solving (1) and (2), simultaneously:

$$F_{AD} = -260 \text{ lb}$$
  $F_{AD} = 260 \text{ lb}$   $C \blacktriangleleft$ 

$$F_{DC}$$
 = +125 lb  $F_{DC}$  = 125 lb  $T$ 

# **PROBLEM 6.8 (Continued)**

Joint E:



Solving (3) and (4), simultaneously:

$$F_{BE} = -832 \text{ lb}$$
  $F_{BE} = 832 \text{ lb}$   $C \blacktriangleleft$   $F_{CE} = +400 \text{ lb}$   $F_{CE} = 400 \text{ lb}$   $T \blacktriangleleft$ 

Joint C:

Force polygon is a parallelogram (see Fig. 6.11 p. 209)

$$F_{AC} = 400 \text{ lb}$$
  $T \blacktriangleleft$ 

$$F_{BC} = 125 \, \text{lb} \quad T \blacktriangleleft$$

Joint A:

$$\pm \Sigma F_x = 0$$
:  $\frac{5}{13}(260 \text{ lb}) + \frac{4}{5}(400 \text{ lb}) + F_{AB} = 0$ 

F= 400lb

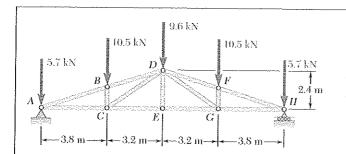
$$F_{AB} = -420 \text{ lb} \qquad F_{AB} = 420 \text{ lb} \qquad C \blacktriangleleft$$

$$F_{AB} = 420 \text{ lb} \qquad C \blacktriangleleft$$

$$F_{AB} = 420 \text{ lb} \qquad C \blacktriangleleft$$

$$F_{AB} = 420 \text{ lb} \qquad C \blacktriangleleft$$

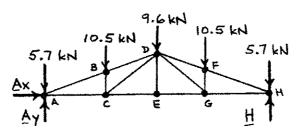
$$0 = 0 \quad \text{(Checks)}$$



Determine the force in each member of the Pratt roof truss shown. State whether each member is in tension or compression.

#### SOLUTION

Free body: Truss



$$\Sigma F_{\rm r} = 0$$
:  $A_{\rm r} = 0$ 

Due to symmetry of truss and load

$$A_y = H = \frac{1}{2}$$
 total load = 21 kN

Free body: Joint A:

$$\frac{F_{AB}}{37} = \frac{F_{AC}}{35} = \frac{15.3 \text{ kN}}{12}$$

$$F_{AB} = 47.175 \text{ kN}$$
  $F_{AC} = 44.625 \text{ kN}$ 

$$F_{AB} = 47.2 \text{ kN} \quad C \blacktriangleleft$$

$$F_{AC} = 44.6 \text{ kN}$$
 T

Free body: Joint B:

From force polygon:

$$F_{BD} = 47.175 \text{ kN}, \quad F_{BC} = 10.5 \text{ kN}$$

$$F_{BC} = 10.50 \, \text{kN} \cdot C \blacktriangleleft$$

$$F_{BD} = 47.2 \text{ kN} \ C \blacktriangleleft$$

# PROBLEM 6.9 (Continued)

$$+ \sum F_y = 0: \quad \frac{3}{5}F_{CD} - 10.5 = 0$$

$$F_{CD} = 17.50 \text{ kN}$$
 T

$$F_{Bc} = 10.5 \text{ kN}$$
 $F_{CE} = \frac{5}{44.625 \text{ kN}}$ 
 $F_{CE} = \frac{5}{44.625 \text{ kN}}$ 
 $F_{CE} = \frac{5}{44.625 \text{ kN}}$ 
 $F_{CE} = 36$ 

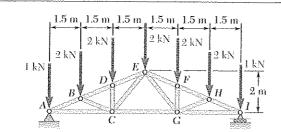
$$\pm \Sigma F_x = 0$$
:  $F_{CE} + \frac{4}{5}(17.50) - 44.625 = 0$ 

$$F_{CE} = 30.625 \text{ kN}$$
  $F_{CE} = 30.6 \text{ kN}$   $T$ 

Free body: Joint *E*:

DE is a zero-force member

Truss and loading symmetrical about &



Determine the force in each member of the fan roof truss shown. State whether each member is in tension or compression.

#### SOLUTION

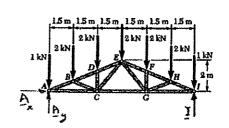
Free body: Truss

$$\Sigma F_x = 0$$
:  $\mathbf{A}_y = 0$ 

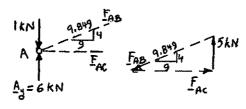
From symmetry of truss and loading:

$$\mathbf{A}_y = \mathbf{I} = \frac{1}{2}$$
 Total load

$$\mathbf{A}_y = \mathbf{I} = 6 \text{ kN}$$



Free body: Joint A:



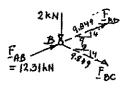
$$\frac{F_{AB}}{9.849} = \frac{F_{AC}}{9} = \frac{5 \text{ kN}}{4}$$

 $F_{AB} = 12.31 \,\text{kN}$  C

 $F_{AC} = 11.25 \text{ kN}$  T

Free body: Joint B:

$$\pm \Sigma F_x = \frac{9}{9.849} (12.31 \text{ kN} + F_{BD} + F_{DC}) = 0$$



or

$$F_{BD} + F_{BC} = -12.31 \,\mathrm{kN}$$

(1)

$$+\int \Sigma F_y = \frac{4}{9.849} (12.31 \text{ kN} + F_{BD} - F_{BC}) - 2 \text{ kN} = 0$$

or

$$F_{BD} - F_{BC} = -7.386 \text{ kN}$$

(2)

Add (1) and (2):

$$2F_{BD} = -19.70 \text{ kN}$$

 $F_{BD} = 9.85 \, \text{kN} \, C \, \blacktriangleleft$ 

Subtract (2) from (1):

$$2F_{BC} = -4.924 \text{ kN}$$

$$F_{BC} = 2.46 \text{ kN} \cdot C \blacktriangleleft$$

# PROBLEM 6.10 (Continued)

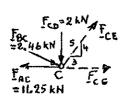
Free body: Joint D:

From force polygon:

$$F_{CD} = 2.00 \text{ kN} \cdot C \blacktriangleleft$$

$$F_{DE} = 9.85 \,\text{kN} \, C \, \blacktriangleleft$$

Free body: Joint C:



$$+1 \Sigma F_y = \frac{4}{5} F_{CE} - \frac{4}{9.849} (2.46 \text{ kN}) - 2 \text{ kN} = 0$$

$$F_{CE} = 3.75 \text{ kN}$$
 T

$$+ \sum F_y = \frac{4}{5}F_{CE} - \frac{4}{9.849}(2.46 \text{ kN}) - 2 \text{ kN} = 0$$

$$+ \sum F_x = 0: \quad F_{CG} + \frac{3}{5}(3.75 \text{ kN}) + \frac{9}{9.849}(2.46 \text{ kN}) - 11.25 \text{ kN} = 0$$

$$F_{CG} = +6.75 \text{ kN}$$

$$F_{CG} = 6.75 \text{ kN}$$
 T

From the symmetry of the truss and loading:

$$F_{EF} = F_{DE}$$

$$F_{EF} = 9.85 \text{ kN}$$
 C

$$F_{EG} = F_{CE}$$

$$F_{FG} = 3.75 \text{ kN}$$
  $T \blacktriangleleft$ 

$$F_{FG} = 2.00 \text{ kN}$$
 C

$$F_{FG} = F_{CD}$$

$$F_{FH} = F_{BD}$$

$$F_{FH} = 9.85 \text{ kN}$$
  $C \blacktriangleleft$ 

$$F_{GH} = F_{BC}$$

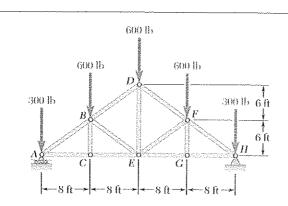
$$F_{GH} = 2.46 \text{ kN} \ C \ \blacktriangleleft$$

$$F_{GI} = F_{AC}$$

$$F_{GI} = 11.25 \, \text{kN} \, T \, \blacktriangleleft$$

$$F_{HI} = F_{AB}$$

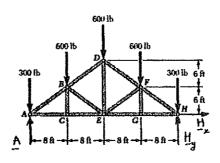
$$F_{HI} = 12.31 \,\text{kN}$$
 C



Determine the force in each member of the Howe roof truss shown. State whether each member is in tension or compression.

#### **SOLUTION**

Free body: Truss



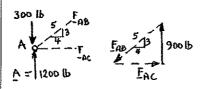
$$\Sigma F_x = 0$$
:  $\mathbf{H}_x = 0$ 

Because of the symmetry of the truss and loading:

$$A = H_y = \frac{1}{2}$$
 Total load

$$A = H_v = 1200 \text{ lb}^{\dagger}$$

Free body: Joint A:



$$\frac{F_{AB}}{5} = \frac{F_{AC}}{4} = \frac{900 \text{ lb}}{3}$$

$$\mathbf{F}_{AB} = 1500 \text{ lb} \quad C \blacktriangleleft$$

$$\mathbf{F}_{4C} = 1200 \, \text{lb} \quad T \blacktriangleleft$$

Free body: Joint C:

BC is a zero-force member

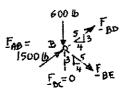
$$\mathbf{F}_{BC} = 0$$

$$\mathbf{F}_{CE} = 1200 \text{ lb} \quad T \blacktriangleleft$$

# **PROBLEM 6.11 (Continued)**

Free body: Joint B:

$$\pm \Sigma F_x = 0$$
:  $\frac{4}{5}F_{BD} + \frac{4}{5}F_{BC} + \frac{4}{5}(1500 \text{ lb}) = 0$ 



or

$$F_{RD} + F_{RE} = -1500 \text{ lb} \tag{1}$$

$$+\int \Sigma F_y = 0$$
:  $\frac{3}{5}F_{BD} - \frac{3}{5}F_{BE} + \frac{3}{5}(1500 \text{ lb}) - 600 \text{ lb} = 0$ 

Ωŧ

$$F_{BD} - F_{BE} = -500 \text{ lb} ag{2}$$

Add Eqs. (1) and (2):

$$2F_{RD} = -2000 \text{ lb}$$

$$F_{RD} = 1000 \, \text{lb} \quad C \blacktriangleleft$$

Subtract (2) from (1):

$$2F_{RE} = -1000 \text{ lb}$$

$$F_{BE} = 500 \text{ lb} \ C \blacktriangleleft$$

Free Body: Joint D:

$$\pm \Sigma F_x = 0$$
:  $\frac{4}{5}(1000 \text{ lb}) + \frac{4}{5}F_{DF} = 0$ 

$$F_{DF} = -1000 \text{ lb}$$

$$F_{DF} = 1000 \text{ lb} \quad C \blacktriangleleft$$

$$+\frac{1}{5}\Sigma F_y = 0$$
:  $\frac{3}{5}(1000 \text{ lb}) - \frac{3}{5}(-1000 \text{ lb}) - 600 \text{ lb} - F_{DE} = 0$ 

$$F_{DE} = +600 \text{ lb}$$

$$F_{DE} = 600 \, \text{lb} \quad T \, \blacktriangleleft$$

Because of the symmetry of the truss and loading, we deduce that

$$F_{EF} = F_{BE}$$

$$F_{FF} = 500 \, \text{lb} \, C \, \blacktriangleleft$$

$$F_{EG} = F_{CE}$$

$$F_{EG} = 1200 \text{ lb}$$
  $T \blacktriangleleft$ 

$$F_{EG} = F_{RC}$$

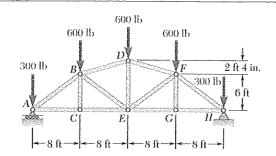
$$F_{FG} = 0$$

$$F_{FH} = F_{AB}$$

$$F_{EH} = 1500 \, \text{lb} \ C \blacktriangleleft$$

$$F_{GH} = F_{AC}$$

$$F_{GH} = 1200 \text{ lb}$$
  $T \blacktriangleleft$ 



Determine the force in each member of the Gambrel roof truss shown. State whether each member is in tension or compression.

#### SOLUTION

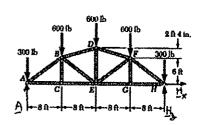
Free body: Truss

$$\Sigma F_x = 0$$
:  $\mathbf{H}_x = 0$ 

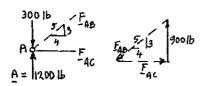
Because of the symmetry of the truss and loading

$$\mathbf{A} = \mathbf{H}_y = \frac{1}{2}$$
 Total load

$$A = H_y = 1200 \text{ lb}^{\dagger}$$



Free body: Joint A:



$$\frac{F_{AB}}{5} = \frac{F_{AC}}{4} = \frac{900 \text{ lb}}{3}$$

 $\mathbf{F}_{AB} = 1500 \, \text{lb} \, C \, \blacktriangleleft$ 

$$\mathbf{F}_{AC} = 1200 \text{ lb}$$
  $T \blacktriangleleft$ 

Free body: Joint C:

BC is a zero-force member

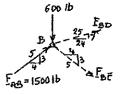
$$\mathbf{F}_{BC} = 0$$

FAC = 1200 16 For

$$F_{CE} = 1200 \text{ lb}$$
  $T \blacktriangleleft$ 

Free body: Joint B:

$$\pm \Sigma F_x = 0$$
:  $\frac{24}{25} F_{BD} + \frac{4}{5} F_{BE} + \frac{4}{5} (1500 \text{ lb}) = 0$ 



(1)

or

$$24F_{BD} + 20F_{BE} = -30,000 \text{ lb}$$

+ 
$$\Sigma F_y = 0$$
:  $\frac{7}{25}F_{BD} - \frac{3}{5}F_{BE} + \frac{3}{5}(1500) - 600 = 0$ 

$$7F_{BD} - 15F_{BE} = -7,500 \text{ lb} \tag{2}$$

# PROBLEM 6.12 (Continued)

Multiply (1) by (3), (2) by 4, and add:

$$100F_{BD} = -120,000 \text{ lb}$$

$$F_{RD} = 1200 \text{ lb} \quad C \blacktriangleleft$$

Multiply (1) by 7, (2) by -24, and add:

$$500F_{RE} = -30,000 \text{ lb}$$

$$F_{BE} = 60.0 \text{ lb}$$
 C

Free body: Joint D:

$$\pm \Sigma F_x = 0$$
:  $\frac{24}{25}(1200 \text{ lb}) + \frac{24}{25}F_{DF} = 0$ 

$$F_{DE} = -1200 \text{ lb}$$

$$F_{DF} = -1200 \text{ lb}$$
  $F_{DF} = 1200 \text{ lb}$   $C \blacktriangleleft$ 

$$+1\Sigma F_y = 0$$
:  $\frac{7}{25}(1200 \text{ lb}) - \frac{7}{25}(-1200 \text{ lb}) - 600 \text{ lb} - F_{DE} = 0$ 

$$F_{DF} = 72.0 \text{ lb}$$

$$F_{DE} = 72.0 \, \text{lb} \, T \, \blacktriangleleft$$

Because of the symmetry of the truss and loading, we deduce that

$$F_{FF} = F_{RE}$$

$$F_{FF} = 60.0 \, \text{lb} \, C \, \blacktriangleleft$$

$$F_{EG} = F_{CE}$$

$$F_{EG} = 1200 \text{ lb}$$
 T

$$F_{FG} = F_{BC}$$

$$F_{FG} = 0$$

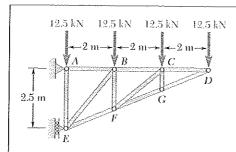
$$F_{FH} = F_{AR}$$

$$F_{FH} = 1500 \, \text{lb} \quad C \blacktriangleleft$$

$$F_{GH} = F_{AC}$$

$$F_{GH} = 1200 \, \text{lb} \ T \blacktriangleleft$$

Note: Compare results with those of Problem 6.9.



Determine the force in each member of the truss shown.

#### SOLUTION

Joint D:

$$\frac{12.5 \text{ kN}}{2.5} = \frac{F_{CD}}{6} = \frac{F_{DG}}{6.5}$$

 $F_{CD} = 30 \text{ kN}$  T

$$F_{DG} = 32.5 \text{ kN}$$
 C

Joint *G*:

$$\Sigma F = 0$$
:  $F_{CG} = 0$ 

...4

For FOR ZAS WI

$$\nearrow \Sigma F = 0$$
:  $F_{FG} = 32.5 \text{ kN}$  C

 $F_{CF} = -19.526 \text{ kN}$ 

-4

<u>Joint *C*</u>:

$$BF = \frac{2}{3}(2.5 \text{ m}) = 1.6667 \text{ m}$$
  $\beta = \angle BCF = \tan^{-1} \frac{BF}{2} = 39.81^{\circ}$ 

F8C C 12.544

$$+ \sum F_y = 0$$
:  $-12.5 \text{ kN} - F_{CF} \sin \beta = 0$ 

$$-12.5 \text{ kN} - F_{CF} \sin 39.81^\circ = 0$$

 $F_{CE} = 19.53 \text{ kN} \cdot C$ 

$$\pm \Sigma F_x = 0$$
: 30 kN  $-F_{RC} - F_{CE} \cos \beta = 0$ 

$$30 \text{ kN} - F_{BC} - (-19.526 \text{ kN}) \cos 39.81^\circ = 0$$

 $F_{RC} = +45.0 \text{ kN}$ 

 $F_{BC} = 45.0 \text{ kN}$   $T \blacktriangleleft$ 

 $\underline{\text{Joint } F}$ :

$$\pm \Sigma F_x = 0$$
:  $-\frac{6}{6.5} F_{EF} - \frac{6}{6.5} (32.5 \text{ kN}) - F_{CF} \cos \beta = 0$ 

F<sub>8</sub> = 32,5 μη

F<sub>6</sub> = 32,5 μη

$$F_{EF} = -32.5 \text{ kN} \sim \left(\frac{6.5}{6}\right) (19.526 \text{ kN}) \cos 39.81^{\circ}$$

$$F_{EF} = -48.75 \text{ kN}$$

 $F_{EF} = 48.8 \,\mathrm{kN} \, C$ 

$$+|\Sigma F_y| = 0$$
:  $F_{BF} - \frac{2.5}{6.5} F_{EF} - \frac{2.5}{6.5} (32.5 \text{ kN}) - (19.526 \text{ kN}) \sin 39.81^\circ = 0$ 

$$F_{BF} - \frac{2.5}{6.5} (-48.75 \text{ kN}) - 12.5 \text{ kN} - 12.5 \text{ kN} = 0$$

$$F_{BF} = +6.25 \text{ kN}$$

 $F_{BF} = 6.25 \,\mathrm{kN}$  T

# **PROBLEM 6.13 (Continued)**

$$\tan \alpha = \frac{2.5 \text{ m}}{2 \text{ m}}; \quad \gamma = 51.34^{\circ}$$

+ 
$$\Sigma F_y = 0$$
: -12.5 kN - 6.25 kN -  $F_{BE} \sin 51.34^\circ = 0$   
 $F_{BE} = -24.0$  kN

$$F_{nc} = -24.0 \text{ kN}$$

$$F_{pr} = 24.0 \text{ kN} \cdot C$$

$$\pm \Sigma F_x = 0$$
: 45.0 kN -  $F_{AB}$  + (24.0 kN) cos 51.34° = 0

$$F_{AB} = +60 \text{ kN}$$

$$F_{AB} = 60.0 \text{ kN}$$
 T

Joint *E*:

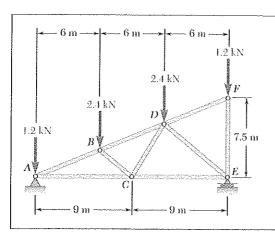
$$\gamma = 51.34^{\circ}$$

$$\gamma = 51.34^{\circ}$$

$$F_{AE} \uparrow \qquad + \Sigma F_{y} = 0: \quad F_{AE} - (24 \text{ kN}) \sin 51.34^{\circ} - (48.75 \text{ kN}) \frac{2.5}{6.5} = 0$$

$$F_{AE} = +37.5 \text{ kN}$$

$$F_{AE} = 37.5 \, \text{kN} \, T$$



Determine the force in each member of the roof truss shown. State whether each member is in tension or compression.

#### **SOLUTION**

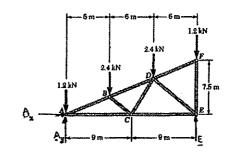
Free body: Truss

$$\Sigma F_{\rm r} = 0$$
:  $\mathbf{A}_{\rm r} = 0$ 

From symmetry of loading:

$$\mathbf{A}_y = \mathbf{E} = \frac{1}{2}$$
 Total load

$$\mathbf{A}_y = \mathbf{E} = 3.6 \text{ kN}$$



We note that DF is a zero-force member and that EF is aligned with the load. Thus

$$F_{DF} = 0$$

$$F_{EF} = 1.2 \text{ kN}$$
 C

Free body: Joint A:

$$\frac{F_{AB}}{13} = \frac{F_{AC}}{12} = \frac{2.4 \text{ kN}}{5}$$

$$F_{AB} = 6.24 \text{ kN}$$
 C

$$F_{AC} = 2.76 \text{ kN}$$
 T

Free body: Joint B:

# PROBLEM 6.14 (Continued)

Multiply (1) by 2.5, (2) by 3, and add:

$$\frac{45}{13}F_{BD} + \frac{45}{13}(6.24 \text{ kN}) - 7.2 \text{ kN} = 0, \quad F_{BD} = -4.16 \text{ kN}, \qquad F_{BD} = 4.16 \text{ kN} \quad C \blacktriangleleft$$

$$F_{BD} = 4.16 \text{ kN}$$
 C

Multiply (1) by 5, (2) by -12, and add:

$$\frac{45}{3.905}F_{BC}$$
 + 28.8 kN = 0,  $F_{BC}$  = -2.50 kN,  $F_{BC}$  = 2.50 kN  $C$  ◀

$$F_{BC} = 2.50 \text{ kN}$$
 C

Free body: Joint C:

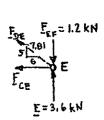
$$+ \sum F_y = 0$$
:  $\frac{5}{5.831} F_{CD} - \frac{2.5}{3.905} (2.50 \text{ kN}) = 0$ 

$$F_{CD} = 1.867 \text{ kN}$$
 T

$$\pm \Sigma F_x = 0$$
:  $F_{CE} - 5.76 \text{ kN} + \frac{3}{3.905} (2.50 \text{ kN}) + \frac{3}{5.831} (1.867 \text{ kN}) = 0$ 

$$F_{CE} = 2.88 \text{ kN} - T$$

Free body: Joint *E*:

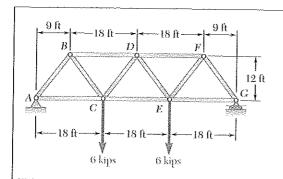


$$+ \int \Sigma F_y = 0$$
:  $\frac{5}{7.81} F_{DE} + 3.6 \text{ kN} - 1.2 \text{ kN} = 0$ 

$$F_{DE} = -3.75 \text{ kN}$$
  $F_{DE} = 3.75 \text{ kN}$   $C$ 

$$+\Sigma F_x = 0$$
:  $-F_{CE} - \frac{6}{7.81}(-3.75 \text{ kN}) = 0$ 

$$F_{CE} = +2.88 \text{ kN}$$
  $F_{CE} = 2.88 \text{ kN}$   $T$  (Checks)



Determine the force in each member of the Warren bridge truss shown. State whether each member is in tension or compression.

## SOLUTION

Free body: Truss

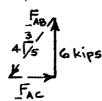
$$\Sigma F_x = 0$$
:  $A_x = 0$ 

Due to symmetry of truss and loading

$$A_y = G = \frac{1}{2}$$
 Total load = 6 kips

Free body: Joint A:

$$\frac{F_{AB}}{5} = \frac{F_{AC}}{3} = \frac{6 \text{ kips}}{4}$$

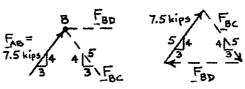


 $F_{AB} = 7.50 \text{ kips}$   $C \blacktriangleleft$ 

$$F_{AC} = 4.50 \text{ kips}$$
  $T \blacktriangleleft$ 

Free body: Joint B:

$$\frac{F_{BC}}{5} = \frac{F_{BD}}{6} = \frac{7.5 \text{ kips}}{5}$$



$$F_{BC} = 7.50 \text{ kips}$$
  $T \blacktriangleleft$ 

 $F_{BD} = 9.00 \text{ kips}$  C

Free body: Joint C:

$$+1\Sigma F_{y} = 0: \frac{4}{5}(7.5) + \frac{4}{5}F_{CD} - 6 = 0$$

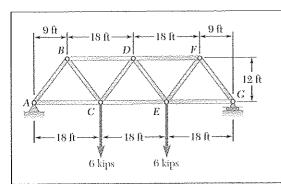
$$+\Sigma F_x = 0$$
:  $F_{CE} - 4.5 - \frac{3}{5}(7.5) = 0$ 

$$+$$
  $F_{CE} = +9 \text{ kips}$ 

 $F_{CD} = 0$ 

 $F_{CE} = 9.00 \text{ kips}$   $T \blacktriangleleft$ 

Truss and loading symmetrical about &

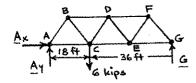


Solve Problem 6.15 assuming that the load applied at E has been removed.

PROBLEM 6.15 Determine the force in each member of the Warren bridge truss shown. State whether each member is in tension or compression.

## SOLUTION

Free body: Truss

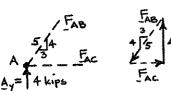


$$\Sigma F_x = 0$$
:  $A_x = 0$ 

+)
$$\Sigma M_G = 0$$
:  $6(36) - A_y(54) = 0$   $A_y = 4 \text{ kips}$ 

$$+ \sum F_y = 0$$
:  $4 - 6 + G = 0$  **G** = 2 kips

Free body: Joint A:



$$\frac{F_{AB}}{5} = \frac{F_{AC}}{3} = \frac{4 \text{ kips}}{4}$$

$$F_{AB} = 5.00 \text{ kips}$$
  $C \blacktriangleleft$ 

$$F_{AC} = 3.00 \text{ kips}$$
  $T \blacktriangleleft$ 

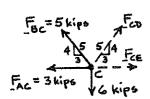
Free body Joint B:

$$\frac{F_{BC}}{5} = \frac{F_{BD}}{6} = \frac{5 \text{ kips}}{5}$$

$$F_{BC} = 5.00 \text{ kips}$$
  $T \blacktriangleleft$ 

$$F_{BD} = 6.00 \text{ kips}$$
  $C \blacktriangleleft$ 

Free body Joint C:



$$+\frac{1}{5}\Sigma M_y = 0$$
:  $\frac{4}{5}(5) + \frac{4}{5}F_{CD} - 6 = 0$ 

$$F_{CD} = 2.50 \text{ kips}$$
  $T \blacktriangleleft$ 

+ 
$$\sum M_y = 0$$
:  $\frac{4}{5}(5) + \frac{4}{5}F_{CD} - 6 = 0$   
+  $\sum F_x = 0$ :  $F_{CE} + \frac{3}{5}(2.5) - \frac{3}{5}(5) - 3 = 0$ 

$$F_{CE} = 4.50 \text{ kips}$$
  $T \blacktriangleleft$ 

# PROBLEM 6.16 (Continued)

Free body: Joint *D*:

$$+ \int \Sigma F_y = 0$$
:  $-\frac{4}{5}(2.5) - \frac{4}{5}F_{DE} = 0$ 

$$F_{DE} = -2.5 \text{ kips}$$

$$F_{DE} = -2.5 \text{ kips}$$
  $F_{DE} = 2.50 \text{ kips}$   $C \blacktriangleleft$ 

$$\pm \Sigma F_x = 0$$
:  $F_{DF} + 6 - \frac{3}{5}(2.5) - \frac{3}{5}(2.5) = 0$ 

$$F_{DF} = -3 \text{ kips}$$

$$F_{DF} = 3.00 \text{ kips} \quad C \blacktriangleleft$$



 $\frac{F_{EF}}{5} = \frac{F_{FG}}{5} = \frac{3 \text{ kips}}{6}$ 

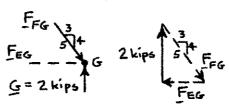
$$F_{EF} = 2.50 \text{ kips}$$
  $T \blacktriangleleft$ 

$$F_{FG} = 2.50 \text{ kips}$$
  $C \blacktriangleleft$ 

Free body: Joint G:

$$\frac{F_{EG}}{3} = \frac{2 \text{ kips}}{4}$$

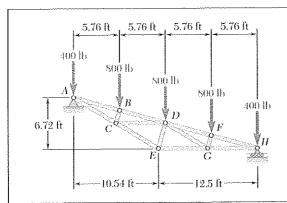
$$F_{EG} = 1.500 \text{ kips}$$
  $T \blacktriangleleft$ 



Also:

$$\frac{F_{FG}}{5} = \frac{2 \text{ kips}}{4}$$

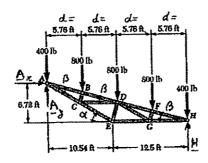
$$F_{FG} = 2.50 \text{ kips}$$
 C (Checks)



Determine the force in member DE and in each of the members located to the left of DE for the inverted Howe roof truss shown. State whether each member is in tension or compression.

#### **SOLUTION**

Free body: Truss:



$$\Sigma F_x = 0$$
:  $\mathbf{A}_x = 0$ 

$$+)\Sigma M_H = 0$$
:  $(400 \text{ lb})(4d) + (800 \text{ lb})(3d) + (800 \text{ lb})(2d) + (800 \text{ lb})d - A_y(4d) = 0$ 

 $A_v = 1600 \text{ lb}$ 

Angles:

$$\tan \alpha = \frac{6.72}{10.54}$$
  $\alpha = 32.52^{\circ}$ 

$$\tan \beta = \frac{6.72}{23.04}$$
  $\beta = 16.26^{\circ}$ 

Free body: Joint A:

$$\frac{F_{AB}}{\sin 57.48^{\circ}} = \frac{F_{AC}}{\sin 106.26^{\circ}} = \frac{1200 \text{ lb}}{\sin 16.26^{\circ}}$$

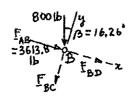
$$F_{AB} = 3613.8 \text{ lb}$$
 C

$$F_{AC} = 4114.3 \text{ lb}$$

$$F_{AB} = 3613.8 \text{ lb}$$
 C  
 $F_{AC} = 4114.3 \text{ lb}$  T  $F_{AB} = 3610 \text{ lb}$  C,  $F_{AC} = 4110 \text{ lb}$  T

# PROBLEM 6.17 (Continued)

Free body: Joint B:



$$+/ \Sigma F_y = 0$$
:  $-F_{BC} - (800 \text{ lb})\cos 16.26^\circ = 0$ 

$$F_{BC} = -768.0 \text{ lb}$$
  $F_{BC} = 768 \text{ lb}$   $C \blacktriangleleft$ 

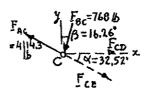
$$+/^{4} \Sigma F_{y} = 0$$
:  $-F_{BC} - (800 \text{ lb}) \cos 16.26^{\circ} = 0$ 

$$F_{BC} = -768.0 \text{ lb} \quad F_{BC}$$
 $+/^{4} \Sigma F_{x} = 0$ :  $F_{BD} + 3613.8 \text{ lb} + (800 \text{ lb}) \sin 16.26^{\circ} = 0$ 

$$F_{BD} = -3837.8 \text{ lb}$$

$$F_{RD} = 3840 \, \text{lb} \ C \blacktriangleleft$$

Free body: Joint C:



$$+ \int \Sigma F_y = 0$$
:  $-F_{CE} \sin 32.52^\circ + (4114.3 \text{ lb}) \sin 32.52^\circ - (768 \text{ lb}) \cos 16.26^\circ = 0$ 

$$F_{CE} = 2742.9 \text{ lb}$$

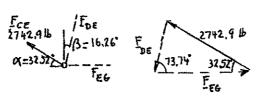
$$F_{CF} = 2740 \, \text{lb} \quad T \quad \blacktriangleleft$$

$$\pm \Sigma F_x = 0$$
:  $F_{CD} - (4114.3 \text{ lb}) \cos 32.52^\circ + (2742.9 \text{ lb}) \cos 32.52^\circ - (768 \text{ lb}) \sin 16.26^\circ = 0$ 

$$F_{CD} = 1371.4 \text{ lb}$$

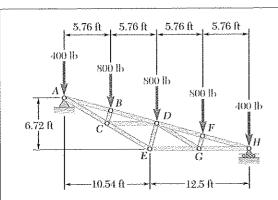
 $F_{CD} = 13711b$  T

Free body: Joint *E*:



$$\frac{F_{DE}}{\sin 32.52^{\circ}} = \frac{2742.9 \text{ lb}}{\sin 73.74^{\circ}}$$

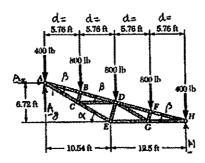
 $F_{DE} = 1536 \, \text{lb} \ C \blacktriangleleft$ 



Determine the force in each of the members located to the right of DE for the inverted Howe roof truss shown. State whether each member is in tension or compression.

#### SOLUTION

Free body: Truss



+) 
$$\Sigma \mathbf{M}_A = 0$$
:  $H(4d) - (800 \text{ lb})d - (800 \text{ lb})(2d) - (800 \text{ lb})(3d) - (400 \text{ lb})(4d) = 0$ 

H = 1600 lb

Angles:

$$\tan \alpha = \frac{6.72}{10.54}$$
  $\alpha = 32.52^{\circ}$   
 $\tan \beta = \frac{6.72}{23.04}$   $\beta = 16.26^{\circ}$ 

$$\tan \beta = \frac{6.72}{23.04}$$
  $\beta = 16.26^{\circ}$ 

Free body: Joint H:

$$F_{GH} = (1200 \text{ lb}) \cot 16.26^{\circ}$$

$$F_{GH} = 4114.3 \text{ lb}$$
 T

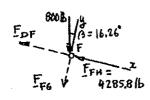
$$F_{GH} = 4110 \, \text{lb} \quad T \blacktriangleleft$$

$$F_{FH} = \frac{1200 \text{ lb}}{\sin 16.26^{\circ}} = 4285.8 \text{ lb}$$

$$F_{FH} = 4290 \text{ lb} \quad C \blacktriangleleft$$

# PROBLEM 6.18 (Continued)

#### Free body Joint F:



+/ 
$$\Sigma F_y = 0$$
:  $-F_{FG} - (800 \text{ lb}) \cos 16.26^\circ = 0$   
 $F_{FG} = -768.0 \text{ lb}$ 

$$F_{FG} = -768.0 \text{ lb}$$

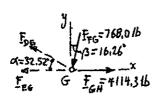
 $F_{FG} = 768 \, \text{lb} \ C \blacktriangleleft$ 

$$^+\Sigma F_x = 0$$
:  $-F_{DF} - 4285.8 \text{ lb} + (800 \text{ lb}) \sin 16.26^\circ = 0$ 

$$F_{DF} = -4061.8 \text{ lb}$$

 $F_{DF} = 4060 \, \text{lb} \quad C \blacktriangleleft$ 

#### Free body: Joint G:



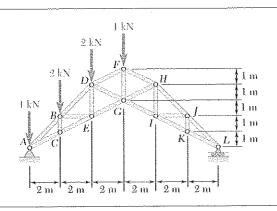
+ 
$$\Sigma F_y = 0$$
:  $F_{DG} \sin 32.52^{\circ} - (768.0 \text{ lb}) \cos 16.26^{\circ} = 0$   
 $F_{DG} = +1371.4 \text{ lb}$ 

 $F_{DG} = 1371 \, \text{lb} \ T \blacktriangleleft$ 

$$\pm \Sigma F_x = 0$$
:  $-F_{EG} + 4114.3 \text{ lb} - (768.0 \text{ lb}) \sin 16.26^\circ - (1371.4 \text{ lb}) \cos 32.52^\circ = 0$ 

$$F_{EG} = +2742.9 \text{ lb}$$

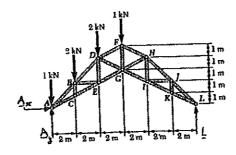
 $F_{EG} = 2740 \text{ lb}$   $T \blacktriangleleft$ 



Determine the force in each of the members located to the left of FG for the scissors roof truss shown. State whether each member is in tension or compression.

#### SOLUTION

Free Body: Truss



$$\Sigma F_v = 0$$
:  $\mathbf{A}_x = 0$ 

$$+)\Sigma M_L = 0$$
:  $(1 \text{ kN})(12 \text{ m}) + (2 \text{ kN})(10 \text{ m}) + (2 \text{ kN})(8 \text{ m}) + (1 \text{ kN})(6 \text{ m}) - A_y(12 \text{ m}) = 0$ 

$$A_y = 4.50 \text{ kN}$$

We note that BC is a zero-force member:

$$F_{BC} = 0$$

Also:

$$F_{CE} = F_{AC} \tag{1}$$

Free body: Joint A:

$$\pm \Sigma F_x = 0$$
:  $\frac{1}{\sqrt{2}} F_{AB} + \frac{2}{\sqrt{5}} F_{AC} = 0$  (2)

+ 
$$\sum F_y = 0$$
:  $\frac{1}{\sqrt{2}} F_{AB} + \frac{1}{\sqrt{5}} F_{AC} + 3.50 \text{ kN} = 0$  (3)

Multiply (3) by -2 and add (2):

$$-\frac{1}{\sqrt{2}}F_{AB} - 7 \text{ kN} = 0$$
  $F_{AB} = 9.90 \text{ kN}$   $C \blacktriangleleft$ 

# PROBLEM 6.19 (Continued)

Subtract (3) from (2):

$$\frac{1}{\sqrt{5}}F_{AC} - 3.50 \text{ kN} = 0$$
  $F_{AC} = 7.826 \text{ kN}$ 

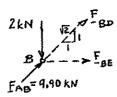
$$F_{AC} = 7.83 \text{ kN}$$
 T

From (1):

$$F_{CE} = F_{AC} = 7.826 \text{ kN}$$

$$F_{CE} = 7.83 \text{ kN}$$
 T

Free body: Joint B:



$$+ \sum F_{y} = 0: \quad \frac{1}{\sqrt{2}} F_{BD} + \frac{1}{\sqrt{2}} (9.90 \text{ kN}) - 2 \text{ kN} = 0$$

$$- \sum F_{BE} = -2$$

$$F_{BD} = -7.071 \,\text{kN}$$

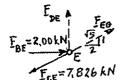
$$F_{BD} = -7.071 \,\text{kN}$$
  $F_{BD} = 7.07 \,\text{kN}$   $C$ 

$$\pm \Sigma F_x = 0$$
:  $F_{BE} + \frac{1}{\sqrt{2}}(9.90 - 7.071)$ kN = 0

$$F_{BE} = -2.000 \text{ kN}$$

$$F_{BE} = -2.000 \text{ kN}$$
  $F_{BE} = 2.00 \text{ kN}$   $C$ 

Free body: Joint E: 
$$\pm \Sigma F_x = 0$$
:  $\frac{2}{\sqrt{5}} (F_{EG} - 7.826 \text{ kN}) + 2.00 \text{ kN} = 0$ 



$$F_{EG} = 5.590 \text{ kN}$$

$$F_{EG} = 5.59 \text{ kN}$$
  $T \blacktriangleleft$ 

$$F_{EG} = 5$$

$$F_{EG} = 5$$

$$+ \sum F_{y} = 0: \quad F_{DE} - \frac{1}{\sqrt{5}} (7.826 - 5.590) \text{kN} = 0$$

$$F_{DE} = 1.0$$

$$F_{DE} = 1.000 \text{ kN}$$

$$F_{DE} = 1.000 \, \text{kN}$$
 T

Free body: Joint D:

$$\pm \Sigma F_x = 0$$
:  $\frac{2}{\sqrt{5}} (F_{DF} + F_{DG}) + \frac{1}{\sqrt{2}} (7.071 \text{ kN})$ 

or

$$F_{DF} + F_{DG} = -5.590 \text{ kN}$$

$$(4)$$

+ 
$$\Sigma F_y = 0$$
:  $\frac{1}{\sqrt{5}} (F_{DF} - F_{DG}) + \frac{1}{\sqrt{2}} (7.071 \text{ kN}) = 2 \text{ kN} - 1 \text{ kN} = 0$ 

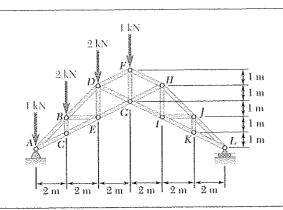
$$F_{DE} - F_{DG} = -4.472$$

$$2F_{DF} = -10.062 \text{ kN}$$

$$F_{DF} = 5.03 \text{ kN}$$
  $C \blacktriangleleft$ 

$$2F_{DG} = -1.1180 \text{ kN}$$

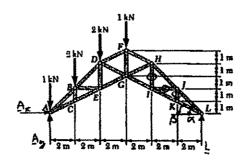
$$F_{DG} = 0.559 \, \text{kN} \cdot C \blacktriangleleft$$



Determine the force in member FG and in each of the members located to the right of FG for the scissors roof truss shown. State whether each member is in tension or compression.

#### **SOLUTION**

Free body: Truss



+)
$$\Sigma M_A = 0$$
:  $L(12 \text{ m}) - (2 \text{ kN})(2 \text{ m}) - (2 \text{ kN})(4 \text{ m}) - (1 \text{ kN})(6 \text{ m}) = 0$ 

L = 1.500 kN

Angles:

$$\tan \alpha = 1$$
  $\alpha = 45^{\circ}$ 

$$\tan \beta = \frac{1}{2}$$
  $\beta = 26.57^{\circ}$ 

#### Zero-force members:

Examining successively joints K, J, and I, we note that the following members to the right of FG are zero-force members: JK, IJ, and HI.

Thus:

$$F_{HI} = F_{IJ} = F_{JK} = 0$$

We also note that

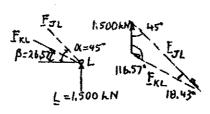
$$F_{GI} = F_{IK} = F_{KL} \tag{1}$$

and

$$F_{HJ} = F_{JL} \tag{2}$$

### PROBLEM 6.20 (Continued)

### Free body: Joint L:



$$\frac{F_{JL}}{\sin 116.57^{\circ}} = \frac{F_{KL}}{\sin 45^{\circ}} = \frac{1.500 \text{ kN}}{\sin 18.43^{\circ}}$$

$$F_{JL} = 4.2436 \text{ kN}$$

$$F_{JL} = 4.24 \text{ kN}$$
  $C \blacktriangleleft$ 

$$F_{KL} = 3.35 \text{ kN}$$
  $T \blacktriangleleft$ 

From Eq. (1):

$$F_{GI} = F_{IK} = F_{KL}$$

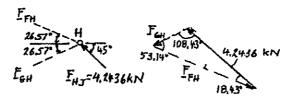
$$F_{GI} = F_{IK} = 3.35 \text{ kN}$$
 T

From Eq. (2):

$$F_{IIJ} = F_{JL} = 4.2436 \text{ kN}$$

$$F_{HJ} = 4.24 \text{ kN}$$
 T

Free body: Joint H:



$$\frac{F_{FH}}{\sin 108.43^{\circ}} = \frac{F_{GH}}{\sin 18.43^{\circ}} = \frac{4.2436}{\sin 53.14^{\circ}}$$

$$F_{FH} = 5.03 \text{ kN} \ C \blacktriangleleft$$

$$F_{GH} = 1.677 \text{ kN}$$
 T

Free body: Joint F:

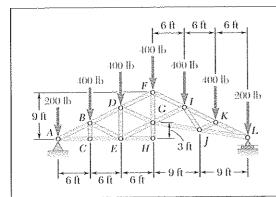
$$+ \Sigma F_x = 0$$
:  $-F_{DF} \cos 26.57^\circ - (5.03 \text{ kN}) \cos 26.57^\circ = 0$ 

$$F_{DF} = -5.03 \text{ kN}$$

+ 
$$\Sigma F_y = 0$$
:  $-F_{FG} - 1 \text{ kN} + (5.03 \text{ kN}) \sin 26.57^\circ - (-5.03 \text{ kN}) \sin 26.57^\circ = 0$ 

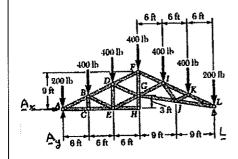
$$F_{FG} = 3.500 \text{ kN}$$

$$F_{FG} = 3.50 \text{ kN}$$
  $T \blacktriangleleft$ 



Determine the force in each of the members located to the left of line FGH for the studio roof truss shown. State whether each member is in tension or compression.

#### SOLUTION



Free body: Truss

$$\Sigma F_x = 0$$
:  $\mathbf{A}_x = 0$ 

Because of symmetry of loading:

$$A_y = L = \frac{1}{2}$$
 Total load

$$A_v = L = 1200 \text{ lb}$$

Zero-Force Members. Examining joints C and H, we conclude that BC, EH, and GH are zero-force members. Thus

$$\mathbf{F}_{RC} = \mathbf{F}_{EH} = 0$$

$$F_{CE} = F_{AC} \tag{1}$$

$$\frac{F_{AB}}{\sqrt{5}} = \frac{F_{AC}}{2} = \frac{1000 \text{ lb}}{1}$$

$$F_{AB} = 2236 \text{ lb}$$
 C

$$F_{AB} = 2240 \, \text{lb} \quad C \blacktriangleleft$$

$$F_{AC} = 2000 \text{ lb} \quad T \blacktriangleleft$$

$$F_{CE} = 2000 \text{ lb}$$
 T

From Ed. (1).

$$+ \Sigma F_x = 0$$
:  $\frac{2}{\sqrt{5}} F_{BD} + \frac{2}{\sqrt{5}} F_{BE} + \frac{2}{\sqrt{5}} (2236 \text{ lb}) = 0$ 

or 
$$F_{BD} + F_{BE} = -2236 \text{ lb}$$
 (2)

+ 
$$\sum F_y = 0$$
:  $\frac{1}{\sqrt{5}} F_{BD} - \frac{1}{\sqrt{5}} F_{BE} + \frac{1}{\sqrt{5}} (2236 \text{ lb}) - 400 \text{ lb} = 0$ 

or 
$$F_{BD} - F_{BE} = -1342 \text{ lb}$$
 (3)

### PROBLEM 6.21 (Continued)

$$2F_{BD} = -3578 \text{ lb}$$

$$F_{RD} = 1789 \, \text{lb} \ C \blacktriangleleft$$

$$2F_{BE} = -894 \text{ lb}$$

$$F_{BE} = 447 \text{ lb} \quad C \blacktriangleleft$$

Free body: Joint E

$$+ \Sigma F_x = 0$$
:  $\frac{2}{\sqrt{5}} F_{EG} + \frac{2}{\sqrt{5}} (447 \text{ lb}) - 2000 \text{ lb} = 0$ 

$$F_{EG} = 1789 \text{ lb}$$
  $T \blacktriangleleft$ 

+ 
$$\sum F_y = 0$$
:  $F_{DE} + \frac{1}{\sqrt{5}} (1789 \text{ lb}) - \frac{1}{\sqrt{5}} (447 \text{ lb}) = 0$ 

$$F_{DE} = -600 \, \text{lb}$$

$$F_{DE} = -600 \text{ lb}$$
  $F_{DE} = 600 \text{ lb}$   $C \blacktriangleleft$ 

Free body: Joint D

$$\pm \Sigma F_x = 0$$
:  $\frac{2}{\sqrt{5}} F_{DF} + \frac{2}{\sqrt{5}} F_{DG} + \frac{2}{\sqrt{5}} (1789 \text{ lb}) = 0$ 

or

$$F_{DF} + F_{DG} = -1789 \text{ lb} ag{4}$$

+ 
$$\Sigma F_y = 0$$
:  $\frac{1}{\sqrt{5}} F_{DF} - \frac{1}{\sqrt{5}} F_{DG} + \frac{1}{\sqrt{5}} (1789 \text{ lb})$   
+ 600 lb - 400 lb = 0

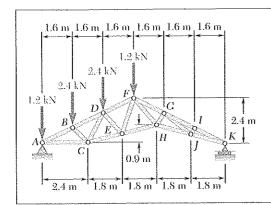
$$F_{DF} - F_{DG} = -2236 \text{ lb}$$
 (5)

$$2F_{DF} = -4025 \text{ lb}$$

$$F_{DF} = 2010 \, \text{lb} \ C \blacktriangleleft$$

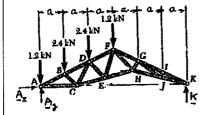
$$2F_{\rm oc} = 447 \, \text{lb}$$

$$2F_{DG} = 447 \text{ lb}$$
  $F_{DG} = 224 \text{ lb}$   $T \blacktriangleleft$ 



Determine the force in each of the members connecting joints A through F of the vaulted roof truss shown. State whether each member is in tension or compression.

#### SOLUTION



Free body: Truss

$$\Sigma F_x = 0: \quad \mathbf{A}_x = 0$$
+ ) \( \Sigma M\_K = 0: \quad (1.2 \text{ kN})6a + (2.4 \text{ kN})5a + (2.4 \text{ kN})4a + (1.2 \text{ kN})3a \\
- A\_v (6a) = 0 \quad \mathbf{A}\_v = 5.40 \text{ kN} \)

Free body: Joint A

$$\frac{F_{AB}}{F_{AC}} = \frac{F_{AC}}{2} = \frac{4.20 \text{ kN}}{1}$$

$$F_{AB} = 9.3915 \text{ kN}$$

$$\sqrt{5}$$
 2 1  
 $F_{AB}$  = 9.3915 kN  $F_{AB}$  = 9.39 kN C ◀

$$F_{AC} = 8.40 \text{ kN}$$
 T

Free body: Joint B

$$\pm \sum F_x = 0: \quad \frac{2}{\sqrt{5}} F_{BD} + \frac{1}{\sqrt{2}} F_{BC} + \frac{2}{\sqrt{5}} (9.3915) = 0 \tag{1}$$

$$+ \sum F_y = 0: \frac{1}{\sqrt{5}} F_{BD} - \frac{1}{\sqrt{2}} F_{BC} + \frac{1}{\sqrt{5}} (9.3915) - 2.4 = 0$$
 (2)

2,4kN FBD FBD FAB 1 1 4 FBC

Add (1) and (2):

$$\frac{3}{\sqrt{5}}F_{BD} + \frac{3}{\sqrt{5}}(9.3915 \,\mathrm{kN}) - 2.4 \,\mathrm{kN} = 0$$

$$F_{BD} = -7.6026 \text{ kN}$$

$$F_{BD} = 7.60 \,\mathrm{kN}$$
 C

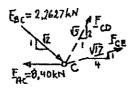
Multiply (2) by –2 and add (1):

$$\frac{3}{\sqrt{2}}F_B + 4.8 \text{ kN} = 0$$

$$F_{BC} = -2.2627 \text{ kN}$$

$$F_{BC} = 2.26 \text{ kN}$$
 C

### PROBLEM 6.23 (Continued)



Free body: Joint C

$$+ \Sigma F_x = 0: \quad \frac{1}{\sqrt{5}} F_{CD} + \frac{4}{\sqrt{17}} F_{CE} + \frac{1}{\sqrt{2}} (2.2627) - 8.40 = 0$$
 (3)

$$+\frac{1}{2}\Sigma F_y = 0: \quad \frac{2}{\sqrt{5}}F_{CD} + \frac{1}{\sqrt{17}}F_{CE} - \frac{1}{\sqrt{2}}(2.2627) = 0$$
 (4)

Multiply (4) by -4 and add (1):

$$-\frac{7}{\sqrt{5}}F_{CD} + \frac{5}{\sqrt{2}}(2.2627) - 8.40 = 0$$

$$F_{CD} = -0.1278 \text{ kN}$$

 $F_{CD} = 0.128 \, \text{kN} \cdot C \blacktriangleleft$ 

Multiply (1) by 2 and subtract (2):

$$\frac{7}{\sqrt{17}}F_{CE} + \frac{3}{\sqrt{2}}(2.2627) - 2(8.40) = 0$$

$$F_{CE} = 7.068 \text{ kN}$$

 $F_{CE} = 7.07 \text{ kN}$  T

Free body: Joint D

$$\frac{+}{\sqrt{5}} \Sigma F_x = 0: \quad \frac{2}{\sqrt{5}} F_{DF} + \frac{1}{1.524} F_{DE} + \frac{2}{\sqrt{5}} (7.6026) + \frac{1}{\sqrt{5}} (0.1278) = 0$$
(5)

$$+ \int \Sigma F_y = 0: \quad \frac{1}{\sqrt{5}} F_{DF} - \frac{1.15}{1.524} F_{DE} + \frac{1}{\sqrt{5}} (7.6026)$$

$$+\frac{2}{\sqrt{5}}(0.1278) - 2.4 = 0 \tag{6}$$

Multiply (5) by 1.15 and add (6):

$$\frac{3.30}{\sqrt{5}}F_{DF} + \frac{3.30}{\sqrt{5}}(7.6026) + \frac{3.15}{\sqrt{5}}(0.1278) - 2.4 = 0$$

$$F_{DF} = -6.098 \, \text{kN}$$

$$F_{DE} = 6.10 \,\text{kN} \cdot C \, \blacktriangleleft$$

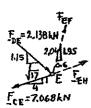
Multiply (6) by -2 and add (5):

$$\frac{3.30}{1.524}F_{DE} - \frac{3}{\sqrt{5}}(0.1278) + 4.8 = 0$$

$$F_{DE} = -2.138 \, \text{kN}$$

$$F_{DE} = 2.14 \, \text{kN} \cdot C \blacktriangleleft$$

### PROBLEM 6.23 (Continued)



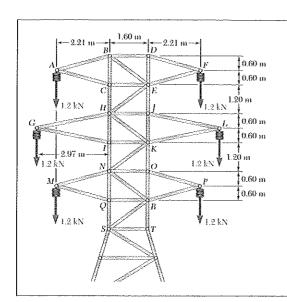
Free body: Joint E

$$+ \Sigma F_x = 0: \quad \frac{0.6}{2.04} F_{EF} + \frac{4}{\sqrt{17}} (F_{EH} - F_{CE}) + \frac{1}{1.524} (2.138) = 0$$
 (7)

+ 
$$\Sigma F_y = 0$$
:  $\frac{1.95}{2.04} F_{EF} + \frac{1}{\sqrt{17}} (F_{EH} - F_{CE}) - \frac{1.15}{1.524} (2.138) = 0$  (8)

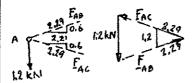
Multiply (8) by 4 and subtract (7):

$$\frac{7.2}{2.04}F_{EF} - 7.856 \text{ kN} = 0 F_{EF} = 2.23 \text{ kN} \quad T \blacktriangleleft$$



The portion of truss shown represents the upper part of a power transmission line tower. For the given loading, determine the force in each of the members located above *HJ*. State whether each member is in tension or compression.

### **SOLUTION**

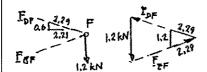


Free body: Joint A

$$\frac{F_{AB}}{2.29} = \frac{F_{AC}}{2.29} = \frac{1.2 \text{ kN}}{1.2}$$

$$F_{AB} = 2.29 \text{ kN}$$
  $T \blacktriangleleft$ 

$$F_{AC} = 2.29 \text{ kN} \cdot C \blacktriangleleft$$

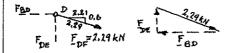


Free body: Joint F

$$\frac{F_{DF}}{2.29} = \frac{F_{EF}}{2.29} = \frac{1.2 \text{ kN}}{2.1}$$

$$F_{DF} = 2.29 \text{ kN}$$
 T

$$F_{EF} = 2.29 \text{ kN}$$
 C



Free body: Joint D

$$\frac{F_{BD}}{2.21} = \frac{F_{DE}}{0.6} = \frac{2.29 \text{ kN}}{2.29}$$

$$F_{BD} = 2.21 \,\mathrm{kN}$$
  $T$ 

$$F_{DE} = 0.600 \text{ kN}$$
 C

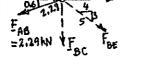
Free body: Joint B

$$\pm \Sigma F_x = 0$$
:  $\frac{4}{5}F_{BE} + 2.21 \text{ kN} - \frac{2.21}{2.29} (2.29 \text{ kN}) = 0$ 

$$F_{BE} = 0$$

+ 
$$\Sigma F_y = 0$$
:  $-F_{BC} - \frac{3}{5}(0) - \frac{0.6}{2.29}(2.29 \text{ kN}) = 0$ 

$$F_{BC} = -0.600 \text{ kN}$$
  $F_{BC} = 0.600 \text{ kN}$   $C \blacktriangleleft$ 



### PROBLEM 6.24 (Continued)

Free body: Joint C

$$F_{CE} = -2.21 \text{ kN} \qquad F_{CE} = 2.21 \text{ kN} \qquad C \blacktriangleleft$$

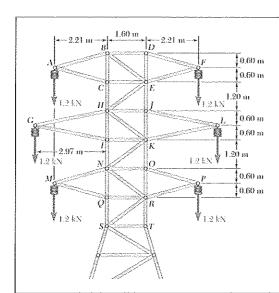
$$F_{CH} = -1.200 \text{ kN} \qquad F_{CH} = 1.200 \text{ kN} \qquad C \blacktriangleleft$$

Free body: Joint E

± Σ
$$F_x = 0$$
: 2.21 kN -  $\frac{2.21}{2.29}$ (2.29 kN) -  $\frac{4}{5}F_{EH} = 0$   
+  $\frac{1}{5}ΣF_y = 0$ : - $F_{EI}$  - 0.600 kN -  $\frac{0.6}{5}$ (2.29 kN) - 0 = 0

+ 
$$\Sigma F_y = 0$$
:  $-F_{EJ} - 0.600 \text{ kN} - \frac{0.6}{2.29} (2.29 \text{ kN}) - 0 = 0$ 

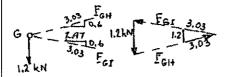
$$F_{EJ} = -1.200 \text{ kN}$$
  $F_{EJ} = 1.200 \text{ kN}$   $C \blacktriangleleft$ 



For the tower and loading of Problem 6.24 and knowing that  $F_{CH} = F_{EJ} = 1.2$  kN C and  $F_{EH} = 0$ , determine the force in member HJ and in each of the members located between HJ and NO. State whether each member is in tension or compression.

**PROBLEM 6.24** The portion of truss shown represents the upper part of a power transmission line tower. For the given loading, determine the force in each of the members located above *HJ*. State whether each member is in tension or compression.

### **SOLUTION**



Free body: Joint G

$$\frac{F_{GH}}{3.03} = \frac{F_{GI}}{3.03} = \frac{1.2 \text{ kN}}{1.2}$$

$$F_{GH} = 3.03 \text{ kN}$$
 T

$$F_{GI} = 3.03 \, \text{kN} \cdot C \blacktriangleleft$$

Free body: Joint L

$$\frac{F_{JL}}{3.03} = \frac{F_{KL}}{3.03} = \frac{1.2 \text{ kN}}{1.2}$$

$$F_{JL} = 3.03 \text{ kN}$$
  $T \blacktriangleleft$ 

$$F_{KL} = 3.03 \text{ kN}$$
 C

Free body: Joint J

$$+ \Sigma F_x = 0$$
:  $-F_{HJ} + \frac{2.97}{3.03} (3.03 \text{ kN}) = 0$ 

$$F_{IJJ} = 2.97 \text{ kN}$$
  $T$ 

$$+ \int F_y = 0$$
:  $-F_{JK} - 1.2 \text{ kN} - \frac{0.6}{3.03} (3.03 \text{ kN}) = 0$ 

$$F_{JK} = -1.800 \text{ kN}$$
  $F_{JK} = 1.800 \text{ kN}$   $C$ 

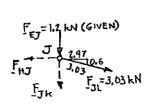
Free body: Joint H

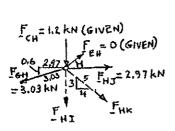
$$+ \Sigma F_x = 0$$
:  $\frac{4}{5} F_{HK} + 2.97 \text{ kN} - \frac{2.97}{3.03} (3.03 \text{ kN}) = 0$ 

$$F_{HK} = 0$$

+ 
$$\Sigma F_y = 0$$
:  $-F_{HI} - 1.2 \text{ kN} - \frac{0.6}{3.03} (3.03) \text{ kN} - \frac{3}{5} (0) = 0$ 

$$F_{HI} = -1.800 \text{ kN}$$
  $F_{III} = 1.800 \text{ kN}$   $C \blacktriangleleft$ 





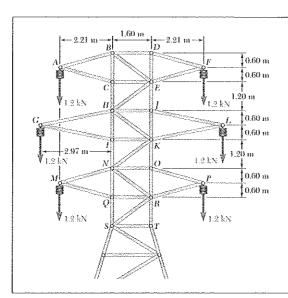
### PROBLEM 6.25 (Continued)

Free body: Joint I

Free body: Joint K

+ 
$$\Sigma F_y = 0$$
:  $-F_{KD} - \frac{0.6}{3.03} (3.03 \text{ kN}) - 1.800 \text{ kN} - \frac{3}{5} (0) = 0$   
 $F_{KD} = -2.40 \text{ kN}$   $F_{KD} = 2.40 \text{ kN}$   $C \blacktriangleleft$ 

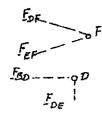
Fix 2.97 km | K 2.97 | K 2.97



Solve Problem 6.24 assuming that the cables hanging from the right side of the tower have fallen to the ground.

PROBLEM 6.24 The portion of truss shown represents the upper part of a power transmission line tower. For the given loading, determine the force in each of the members located above HJ. State whether each member is in tension or compression.

### SOLUTION



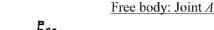
Zero-Force Members.

Considering joint F, we note that DF and EF are zero-force members:

$$F_{DF} = F_{FF} = 0$$

Considering next joint D, we note that BD and DE are zero-force members:

$$F_{RD} = F_{DE} = 0$$





$$\frac{F_{AB}}{2.29} = \frac{F_{AC}}{2.29} = \frac{1.2 \text{ kN}}{1.2}$$
  $F_{AB} = 2.29 \text{ kN}$   $T \blacktriangleleft$ 

$$F_{AC} = 2.29 \text{ kN}$$
 C

Free body: Joint B

$$+ \Sigma F_x = 0: \quad \frac{4}{5} F_{BE} - \frac{2.21}{2.29} (2.29 \text{ kN}) = 0$$

$$F_{BE} = 2.7625 \text{ kN}$$

$$F_{BE} = 2.7625 \text{ kN}$$
  $F_{BE} = 2.76 \text{ kN}$   $T \blacktriangleleft$ 

+ 
$$\Sigma F_y = 0$$
:  $-F_{BC} - \frac{0.6}{2.29} (2.29 \text{ kN}) - \frac{3}{5} (2.7625 \text{ kN}) = 0$ 

$$F_{BC} = -2.2575 \text{ kN}$$
  $F_{BC} = 2.26 \text{ kN}$   $C \blacktriangleleft$ 



### PROBLEM 6.26 (Continued)

Free body: Joint C

$$\pm \Sigma F_x = 0$$
:  $F_{CE} + \frac{2.21}{2.29} (2.29 \text{ kN}) = 0$ 

$$F_{CE} = 2.21 \, \text{kN} \cdot C \, \blacktriangleleft$$

+ 
$$\Sigma F_y = 0$$
:  $-F_{CH} - 2.2575 \text{ kN} - \frac{0.6}{2.29} (2.29 \text{ kN}) = 0$ 

$$F_{CH} = -2.8575 \,\text{kN}$$
  $F_{CH} = 2.86 \,\text{kN}$   $C$ 

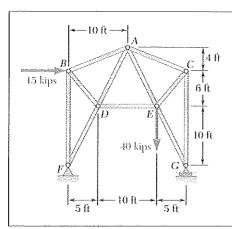
Free body: joint E

$$+\Sigma F_x = 0$$
:  $-\frac{4}{5}F_{EH} - \frac{4}{5}(2.7625 \text{ kN}) + 2.21 \text{ kN} = 0$ 

 $F_{EH} = 0$ 

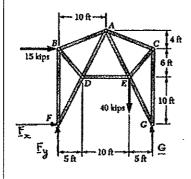
+ 
$$\sum F_y = 0$$
:  $-F_{EJ} + \frac{3}{5}(2.7625 \text{ kN}) - \frac{3}{5}(0) = 0$ 

$$F_{EJ} = +1.6575 \text{ kN}$$
  $F_{EJ} = 1.658 \text{ kN}$   $T$ 



Determine the force in each member of the truss shown. State whether each member is in tension or compression.

### **SOLUTION**



### Free body: Truss

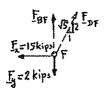
$$F_{x} = 0: \quad G(20 \text{ ft}) - (15 \text{ kips})(16 \text{ ft}) - (40 \text{ kips})(15 \text{ ft}) = 0$$

$$G = 42 \text{ kips} \uparrow$$

$$F_{x} = 0: \quad F_{x} + 15 \text{ kips} = 0$$

$$F_{x} = 15 \text{ kips} \leftarrow$$

$$F_{x} = 0: \quad F_{y} - 40 \text{ kips} + 42 \text{ kips} = 0$$



$$\pm \sum F_x = 0: \quad \frac{1}{\sqrt{5}} F_{DF} - 15 \text{ kips} = 0$$

$$F_{DF} = 33.54 \text{ kips}$$
  $F_{DF} = 33.5 \text{ kips}$   $T \blacktriangleleft + \sum_{y=0}^{4} \sum_{y=0}^{4} F_{BF} = 0$ 

$$F_{BF} = -28.00 \text{ kips}$$
  $F_{BF} = 28.0 \text{ kips}$   $C \blacktriangleleft$ 

 $\mathbf{F}_v = 2 \text{ kips}$ 

### Free body: Joint B

$$\pm \Sigma F_x = 0$$
:  $\frac{5}{\sqrt{29}} F_{AB} + \frac{5}{\sqrt{61}} F_{BD} + 15 \text{ kips} = 0$  (1)

$$+\frac{1}{2}\Sigma F_y = 0: \quad \frac{2}{\sqrt{29}}F_{AB} - \frac{6}{\sqrt{61}}F_{BD} + 28 \text{ kips} = 0$$
 (2)

### PROBLEM 6.27 (Continued)

Multiply (1) by 6, (2) by 5, and add:

$$\frac{40}{\sqrt{29}}F_{AB} + 230 \text{ kips} = 0$$

$$F_{AR} = -30.96 \text{ kip}$$

$$F_{AB} = -30.96 \text{ kips}$$
  $F_{AB} = 31.0 \text{ kips}$   $C \blacktriangleleft$ 

Multiply (1) by 2, (2) by -5, and add:

$$\frac{40}{\sqrt{61}}F_{BD} - 110 \text{ kips} = 0$$

$$F_{mo} = 21.48 \text{ kips}$$

$$F_{BD} = 21.48 \text{ kips}$$
  $F_{BD} = 21.5 \text{ kips}$   $T \blacktriangleleft$ 

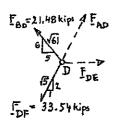
Free body: joint D

+ 
$$\Sigma F_y = 0$$
:  $\frac{2}{\sqrt{5}} F_{AD} - \frac{2}{\sqrt{5}} (33.54) + \frac{6}{\sqrt{61}} (21.48) = 0$ 

$$F_{4D} = 15.09 \text{ kips} \ T \blacktriangleleft$$

$$\pm \Sigma F_x = 0$$
:  $F_{DE} + \frac{1}{\sqrt{5}} (15.09 - 33.54) - \frac{5}{\sqrt{61}} (21.48) = 0$ 

$$F_{DE} = 22.0 \text{ kips}$$
  $T \blacktriangleleft$ 



Free body: joint A

$$+ \sum F_x = 0: \quad \frac{5}{\sqrt{29}} F_{AC} + \frac{1}{\sqrt{5}} F_{AE} + \frac{5}{\sqrt{29}} (30.36) - \frac{1}{\sqrt{5}} (15.09) = 0$$
 (3)

+ 
$$\Sigma F_y = 0$$
:  $-\frac{2}{\sqrt{29}} F_{AC} - \frac{2}{\sqrt{5}} F_{AE} + \frac{2}{\sqrt{29}} (30.96) - \frac{2}{\sqrt{5}} (15.09) = 0$  (4)

Multiply (3) by 2 and add (4):

$$\frac{8}{\sqrt{29}}F_{AC} + \frac{12}{\sqrt{29}}(30.96) - \frac{4}{\sqrt{5}}(15.09) = 0$$

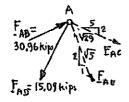
$$F_{AC} = -28.27 \text{ kips},$$

$$F_{AC} = 28.3 \text{ kips}$$
 C

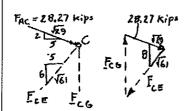
Multiply (3) by 2 (4) by 5 and add:

$$-\frac{8}{\sqrt{5}}F_{AE} + \frac{20}{\sqrt{29}}(30.96) - \frac{12}{\sqrt{5}}(15.09) = 0$$

$$F_{AE} = 9.50 \text{ kips}$$
  $T \blacktriangleleft$ 



## PROBLEM 6.27 (Continued)



Free body: Joint C

From force triangle

$$\frac{F_{CE}}{\sqrt{61}} = \frac{F_{CG}}{8} = \frac{28.27 \text{ kips}}{\sqrt{29}}$$

$$F_{CE} = 41.0 \text{ kips}$$
  $T \blacktriangleleft$ 

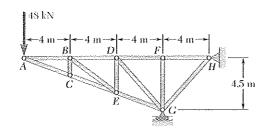
$$F_{CG} = 42.0 \text{ kips}$$
 C

Free body: Joint G

$$^{\perp}$$
  $\Sigma F_x = 0$ :

$$F_{\rm EC} = 0$$

+ 
$$\Sigma F_y = 0$$
: 42 kips – 42 kips = 0 (Checks)



Determine the force in each member of the truss shown. State whether each member is in tension or compression.

### **SOLUTION**

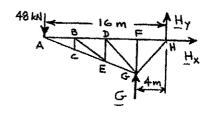
Free body: Truss

$$\Sigma F_{\rm x} = 0$$
:  $\mathbf{H}_{\rm x} = 0$ 

+)
$$\Sigma M_H = 0$$
:  $48(16) - G(4) = 0$  **G** = 192 kN

$$+\int \Sigma F_{\nu} = 0$$
:  $192 - 48 + H_{\nu} = 0$ 

$$H_y = -144 \text{ kN}$$
  $H_y = 144 \text{ kN}$ 



#### Zero-Force Members:

Examining successively joints C, B, E, D, and F, we note that the following are zero-force members: BC, BE, DE, DG, FG

Thus,

$$F_{BC} = F_{BE} = F_{DE} = F_{DG} = F_{FG} = 0$$

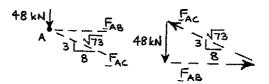
Also note:

$$F_{AB} = F_{BD} = F_{DF} = F_{FH} \tag{1}$$

$$F_{AC} = F_{CE} = F_{EG} \tag{2}$$

Free body: Joint A:

$$\frac{F_{AB}}{8} = \frac{F_{AC}}{\sqrt{73}} = \frac{48 \text{ kN}}{3}$$



$$F_{AB} = 128 \text{ kN}$$

$$F_{AB} = 128.0 \,\text{kN}$$
  $T$ 

$$F_{AC} = 136.704 \text{ kN}$$

$$F_{AC} = 136.7 \text{ kN} \cdot C \blacktriangleleft$$

$$F_{BD} = F_{DF} = F_{EH} = 128.0 \text{ kN}$$
 T

$$F_{CF} = F_{EG} = 136.7 \,\text{kN}$$
 C

### **PROBLEM 6.28 (Continued)**

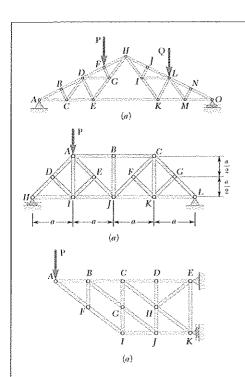
$$\frac{F_{GH}}{\sqrt{145}} = \frac{144 \text{ kN}}{9}$$

$$F_{CH} = 192.7 \text{ kN} \quad C \blacktriangleleft$$

Also

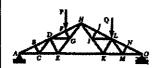
$$\frac{F_{FH}}{8} = \frac{144 \text{ kN}}{9}$$

$$F_{FH} = 128.0 \text{ kN}$$
 T (Checks)



Determine whether the trusses of Problems 6.31a, 6.32a, and 6.33a are simple trusses.

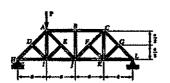
### **SOLUTION**



#### Truss of Problem 6.31a

Starting with triangle ABC and adding two members at a time, we obtain joints D, E, G, F, and H, but cannot go further thus, this truss

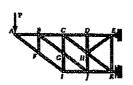
is not a simple truss ◀



### Truss of Problem 6.32a

Starting with triangle HDI and adding two members at a time, we obtain successively joints A, E, J, and B, but cannot go further. Thus, this truss

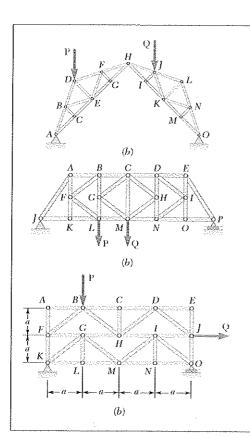
is not a simple truss ◀



### Truss of Problem 6.33a

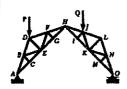
Starting with triangle EHK and adding two members at a time, we obtain successively joints D, J, C, G, I, B, F, and A, thus completing the truss. Therefore, this truss

is a simple truss ◀



Determine whether the trusses of Problems 6.31b, 6.32b, and 6.33b are simple trusses.

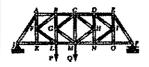
#### SOLUTION



### Truss of Problem 6.31b.

Starting with triangle ABC and adding two members at a time, we obtain successively joints E, D, F, G, and H, but cannot go further. Thus, this truss

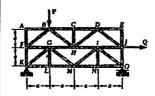
is not a simple truss ◀



#### Truss of Problem 6.32b.

Starting with triangle CGH and adding two members at a time, we obtain successively joints B, L, F, A, K, J, then H, D, N, I, E, O, and P, thus completing the truss.

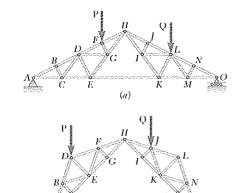
Therefore, this truss is a simple truss ◀



#### Truss of Problem 6.33b.

Starting with triangle GLM and adding two members at a time, we obtain joints K and I but cannot continue, starting instead with triangle BCH, we obtain joint D but cannot continue, thus, this truss

is not a simple truss ◀



For the given loading, determine the zero-force members in each of the two trusses shown.

#### **SOLUTION**

Truss (a)

FB: Joint B: 
$$F_{BC} = 0$$

$$FB$$
: Joint  $C$ :  $F_{CD} = 0$ 

FB: Joint J: 
$$F_{IJ} = 0$$

*FB*: Joint *I*: 
$$F_{IL} = 0$$

FB: Joint N: 
$$F_{MN} = 0$$

*FB*: Joint *M*: 
$$F_{LM} = 0$$

The zero-force members, therefore, are

Truss (b)

*FB*: Joint *C*: 
$$F_{BC} = 0$$

FB: Joint B: 
$$F_{BE} = 0$$

FB: Joint G: 
$$F_{FG} = 0$$

FB: Joint F: 
$$F_{FF} = 0$$

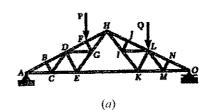
FB: Joint E: 
$$F_{DE} = 0$$

FB: Joint I: 
$$F_{IJ} = 0$$

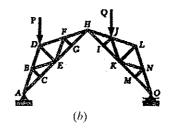
FB: Joint M:  $F_{MN} = 0$ 

FB: Joint N: 
$$F_{KN} = 0$$

The zero-force members, therefore, are



 $BC,CD,IJ,IL,LM,MN \blacktriangleleft$ 



 $BC, BE, DE, EF, FG, IJ, KN, MN \blacktriangleleft$ 

### PROBLEM 6.35\* (Continued)

Substituting for  $F_{CA}$ ,  $F_{CB}$ ,  $F_{CD}$ , and equating to zero the coefficients of i, j, k:

i: 
$$-\frac{24}{26}F_{AC} - \frac{24}{25}(F_{BC} + F_{CD}) = 0$$
 (1)

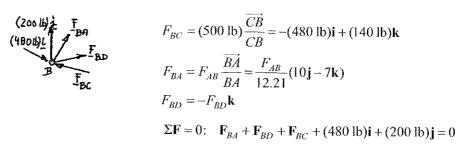
**j**: 
$$\frac{10}{26}F_{AC} - 400 \text{ lb} = 0$$
 
$$F_{AC} = 1040 \text{ lb} \quad T \blacktriangleleft$$

**k**: 
$$\frac{7}{25}(F_{BC} - F_{CD}) = 0$$
  $F_{CD} = F_{BC}$ 

Substitute for  $F_{AC}$  and  $F_{CD}$  in Eq. (1):

$$-\frac{24}{26}(10.40 \text{ lb}) - \frac{24}{25}(2 F_{BC}) = 0 \quad F_{BC} = -500 \text{ lb} \qquad F_{BC} = F_{CD} = 500 \text{ lb} \quad C \blacktriangleleft$$

Free body: B



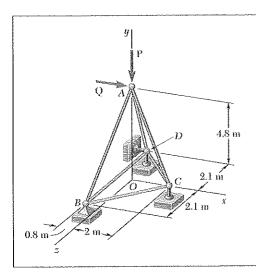
Substituting for  $\mathbf{F}_{BA}$ ,  $\mathbf{F}_{BD}$ ,  $\mathbf{F}_{BC}$  and equating to zero the coefficients of  $\mathbf{j}$  and  $\mathbf{k}$ :

j: 
$$\frac{10}{12.21}F_{AB} + 200 \text{ lb} = 0$$
  $F_{AB} = -244.2 \text{ lb}$   $F_{AB} = 244 \text{ lb}$   $C \blacktriangleleft$ 

**k**: 
$$-\frac{7}{12.21}F_{AB} - F_{BD} + 140 \text{ lb} = 0$$

$$F_{BD} = -\frac{7}{12.21}(-244.2 \text{ lb}) + 140 \text{ lb} = +280 \text{ lb}$$
  $F_{BD} = 280 \text{ lb}$   $T \blacktriangleleft$ 

From symmetry: 
$$F_{AD} = F_{AB}$$
  $F_{AD} = 244 \text{ lb } C \blacktriangleleft$ 



### **PROBLEM 6.36\***

The truss shown consists of six members and is supported by a ball and socket at B, a short link at C, and two short links at D. Determine the force in each of the members for  $P = (-2184 \text{ N})\mathbf{j}$  and Q = 0.

### SOLUTION

Free body: Truss

From symmetry:

$$D_x = B_x$$
 and  $D_y = B_y$   
 $\Sigma F_x = 0$ :  $2B_y = 0$ 

$$B_x = D_x = 0$$

$$\Sigma F_z = 0$$
:  $B_z = 0$ 

$$\Sigma M_{cz} = 0$$
:  $-2B_{y}(2.8 \text{ m}) + (2184 \text{ N})(2 \text{ m}) = 0$ 

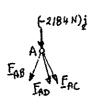
$$B_y = 780 \text{ N}$$

Thus

 $\mathbf{B} = (780 \text{ N})j \triangleleft$ 

=-(2184 M)

Free body: A



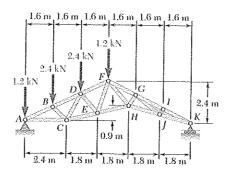
$$F_{AB} = F_{AB} \frac{\overrightarrow{AB}}{AB} = \frac{F_{AB}}{5.30} (-0.8\mathbf{i} - 4.8\mathbf{j} + 2.1\mathbf{k})$$

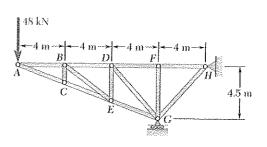
$$F_{AC} = F_{AC} \frac{\overrightarrow{AC}}{AC} = \frac{F_{AC}}{5.20} (2\mathbf{i} - 4.8\mathbf{j})$$

$$F_{AD} = F_{AD} \frac{\overrightarrow{AD}}{AD} = \frac{F_{AD}}{5.30} (+0.8\mathbf{i} - 4.8\mathbf{j} - 2.1\mathbf{k})$$

$$\Sigma \mathbf{F} = 0$$
:  $\mathbf{F}_{AB} + \mathbf{F}_{AC} + \mathbf{F}_{AD} - (2184 \text{ N})\mathbf{j} = 0$ 

Determine the zero-force members in the truss of (a) Problem 6.23, (b) Problem 6.28.





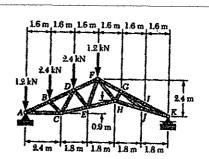
### SOLUTION

### (a) Truss of Problem 6.23

FB: Joint I:  $F_{IJ} = 0$ 

*FB*: Joint *J*:  $F_{GJ} = 0$ 

FB: Joint G:  $F_{GH} = 0$ 



GH,GJ,IJ

The zero-force members, therefore, are

### (b) Truss of Problem 6.28

FB: Joint C:  $F_{BC} = 0$ 

FB: Joint B:  $F_{BE} = 0$ 

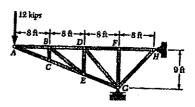
FB: Joint E:  $F_{DE} = 0$ 

FB: Joint D:  $F_{DG} = 0$ 

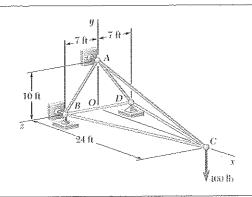
FB: Joint F:  $F_{FG} = 0$ 

The zero-force members, therefore, are

RC RE DE



 $BC, BE, DE, DG, FG \blacktriangleleft$ 



### **PROBLEM 6.35\***

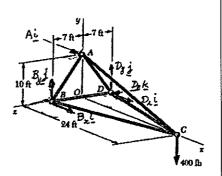
The truss shown consists of six members and is supported by a short link at A, two short links at B, and a ball and socket at D. Determine the force in each of the members for the given loading.

### SOLUTION

Free body: Truss

From symmetry:

$$D_x = B_x$$
 and  $D_y = B_y$   
 $\Sigma M_z = 0$ :  $-A(10 \text{ ft}) - (400 \text{ lb})(24 \text{ ft}) = 0$   
 $A = -960 \text{ lb}$   
 $\Sigma F_x = 0$ :  $B_x + D_x + A = 0$   
 $2B_x - 960 \text{ lb} = 0$ ,  $B_x = 480 \text{ lb}$   
 $\Sigma F_y = 0$ :  $B_y + D_y - 400 \text{ lb} = 0$   
 $2B_y = 400 \text{ lb}$   
 $B_y = +200 \text{ lb}$ 



Thus

 $\mathbf{B} = (480 \text{ lb})\mathbf{i} + (200 \text{ lb})\mathbf{j} \le$ 

Free body: C

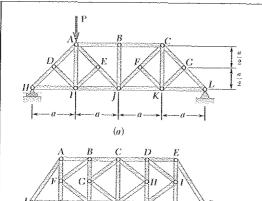


$$F_{CA} = F_{AC} \frac{\overline{CA}}{CA} = \frac{F_{AC}}{26} (-24\mathbf{i} + 10\mathbf{j})$$

$$F_{CB} = F_{BC} \frac{\overline{CB}}{CB} = \frac{F_{BC}}{25} (-24\mathbf{i} + 7\mathbf{k})$$

$$F_{CD} = F_{CD} \frac{\overline{CD}}{CD} = \frac{F_{CD}}{25} (-24\mathbf{i} - 7\mathbf{k})$$

$$\Sigma \mathbf{F} = 0: \quad \mathbf{F}_{CA} + \mathbf{F}_{CB} + \mathbf{F}_{CD} - (400 \text{ lb}) \mathbf{j} = 0$$



**(b)** 

### PROBLEM 6.32

For the given loading, determine the zero-force members in each of the two trusses shown.

### **SOLUTION**

Truss (a)

FB: Joint B: 
$$F_{BJ} = 0$$

FB: Joint D: 
$$F_{DI} = 0$$

FB: Joint E: 
$$F_{EI} = 0$$

FB: Joint I: 
$$F_{AI} = 0$$

*FB*: Joint 
$$F: F_{FK} = 0$$

FB: Joint G: 
$$F_{GK} = 0$$

FB: Joint K: 
$$F_{CK} = 0$$

The zero-force members, therefore, are

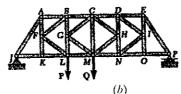
 $AI,BJ,CK,DI,EI,FK,GK \blacktriangleleft$ 

(a)

 $\underline{\text{Truss}}(b)$ 

*FB*: Joint *K*: 
$$F_{FK} = 0$$

$$FB$$
: Joint  $O$ :  $F_{IO} = 0$ 

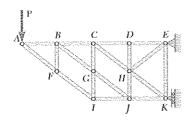


The zero-force members, therefore, are

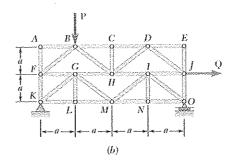
All other members are either in tension or compression.

FK and IO

For the given loading, determine the zero-force members in each of the two trusses shown.



(a)



### **SOLUTION**

Truss (a)

FB: Joint F: 
$$F_{BF} = 0$$

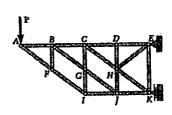
FB: Joint B: 
$$F_{RG} = 0$$

FB: Joint G: 
$$F_{GJ} = 0$$

FB: Joint D: 
$$F_{DH} = 0$$

*FB*: Joint *J*: 
$$F_{HJ} = 0$$

FB: Joint H: 
$$F_{EH} = 0$$



(a)

The zero-force members, therefore, are

Truss(b)

FB: Joint A: 
$$F_{AB} = F_{AF} = 0$$

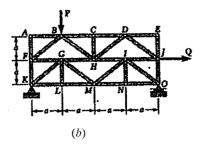
FB: Joint C: 
$$F_{CH} = 0$$

FB: Joint E: 
$$F_{DE} = F_{EJ} = 0$$

FB: Joint L: 
$$F_{GL} = 0$$

FB: Joint N: 
$$F_{IN} = 0$$

 $BF, BG, DH, EH, GJ, HJ \blacktriangleleft$ 



 $AB, AF, CH, DE, EJ, GL, IN \blacktriangleleft$ 

The zero-force members, therefore, are

### PROBLEM 6.36\* (Continued)

Substituting for  $\mathbf{F}_{AB}$ ,  $\mathbf{F}_{AC}$ ,  $\mathbf{F}_{AD}$ , and equating to zero the coefficients of  $\mathbf{i}$ ,  $\mathbf{j}$ ,  $\mathbf{k}$ :

i: 
$$-\frac{0.8}{5.30}(F_{AB} + F_{AD}) + \frac{2}{5.20}F_{AC} = 0$$
 (1)

j: 
$$-\frac{4.8}{5.30}(F_{AB} + F_{AD}) - \frac{4.8}{5.20}F_{AC} - 2184 \text{ N} = 0$$
 (2)

**k**: 
$$\frac{2.1}{5.30}(F_{AB} - F_{AD}) = 0$$
  $F_{AD} = F_{AB}$ 

Multiply (1) by -6 and add (2):

$$-\left(\frac{16.8}{5.20}\right)F_{AC} - 2184 \text{ N} = 0, \quad F_{AC} = -676 \text{ N}$$
  $F_{AC} = 676 \text{ N}$   $C \blacktriangleleft$ 

Substitute for  $F_{AC}$  and  $F_{AD}$  in (1):

$$-\left(\frac{0.8}{5.30}\right)2F_{AB} + \left(\frac{2}{5.20}\right)(-676 \text{ N}) = 0, \quad F_{AB} = -861.25 \text{ N} \qquad F_{AB} = F_{AD} = 861 \text{ N} \quad C \blacktriangleleft$$

Free body: B

$$\mathbf{F}_{AB} = (861.25 \text{ N}) \frac{\overrightarrow{AB}}{AB} = -(130 \text{ N})\mathbf{i} - (780 \text{ N})\mathbf{j} + (341.25 \text{ N})\mathbf{k}$$

$$\mathbf{F}_{BC} = F_{BC} \left( \frac{2.8\mathbf{i} - 2.1\mathbf{k}}{3.5} \right) = F_{BC} (0.8\mathbf{i} - 0.6\mathbf{k})$$

$$\mathbf{F}_{BD} = -F_{BD}\mathbf{k}$$

$$\Sigma \mathbf{F} = 0: \quad \mathbf{F}_{AB} + \mathbf{F}_{BC} + \mathbf{F}_{BD} + (780 \text{ N})\mathbf{j} = 0$$

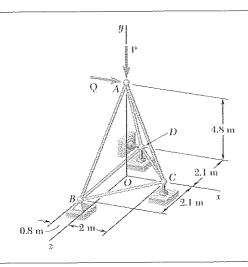
Substituting for  $\mathbf{F}_{AB}$ ,  $\mathbf{F}_{BC}$ ,  $\mathbf{F}_{BD}$  and equating to zero the coefficients of  $\mathbf{i}$  and  $\mathbf{k}$ :

i: 
$$-130 \text{ N} + 0.8 F_{BC} = 0$$
  $F_{BC} = +162.5 \text{ N}$   $F_{BC} = 162.5 \text{ N}$   $T \blacktriangleleft$ 

k: 
$$341.25 \text{ N} - 0.6 F_{BC} - F_{BD} = 0$$

$$F_{BD} = 341.25 - 0.6(162.5) = +243.75 \text{ N}$$
  $F_{RD} = 244 \text{ N}$  T

From symmetry: 
$$F_{CD} = F_{BC}$$
  $F_{CD} = 162.5 \text{ N}$   $T$ 



### **PROBLEM 6.37\***

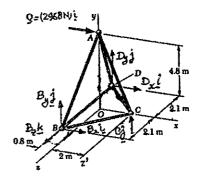
The truss shown consists of six members and is supported by a ball and socket at B, a short link at C, and two short links at D. Determine the force in each of the members for P = 0 and Q = (2968 N)i.

### SOLUTION

### Free body: Truss

From symmetry:

$$D_x = B_x$$
 and  $D_y = B_y$   
 $\Sigma F_x = 0$ :  $2B_x + 2968 \text{ N} = 0$   
 $B_x = D_x = -1484 \text{ N}$   
 $\Sigma M_{cz'} = 0$ :  $-2B_y (2.8 \text{ m}) - (2968 \text{ N})(4.8 \text{ m}) = 0$ 

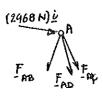


$$B_y = -2544 \text{ N}$$

Thus

 $\mathbf{B} = -(1484 \text{ N})\mathbf{i} - (2544 \text{ N})\mathbf{j} \triangleleft$ 

Free body: A



$$F_{AB} = F_{AB} \frac{\overline{AB}}{AB}$$

$$= \frac{F_{AB}}{5.30} (-0.8\mathbf{i} - 4.8\mathbf{j} + 2.1\mathbf{k})$$

$$F_{AC} = F_{AC} \frac{\overline{AC}}{AC} = \frac{F_{AC}}{5.20} (2\mathbf{i} - 4.8\mathbf{j})$$

$$F_{AD} = F_{AD} \frac{\overline{AD}}{AD}$$

$$= \frac{F_{AD}}{5.30} (-0.8\mathbf{i} - 4.8\mathbf{j} - 2.1\mathbf{k})$$

$$\Sigma \mathbf{F} = 0: \quad \mathbf{F}_{AB} + \mathbf{F}_{AC} + \mathbf{F}_{AD} + (2968 \text{ N})\mathbf{i} = 0$$

### PROBLEM 6.37\* (Continued)

Substituting for  $\mathbf{F}_{AB}$ ,  $\mathbf{F}_{AC}$ ,  $\mathbf{F}_{AD}$ , and equating to zero the coefficients of  $\mathbf{i}$ ,  $\mathbf{j}$ ,  $\mathbf{k}$ :

i: 
$$-\frac{0.8}{5.30}(F_{AB} + F_{AD}) + \frac{2}{5.20}F_{AC} + 2968 \text{ N} = 0$$
 (1)

$$\mathbf{j}: \qquad -\frac{4.8}{5.30}(F_{AB} + F_{AD}) - \frac{4.8}{5.20}F_{AC} = 0 \tag{2}$$

**k**: 
$$\frac{2.1}{5.30}(F_{AB} - F_{AD}) = 0$$
  $F_{AD} = F_{AB}$ 

Multiply (1) by -6 and add (2):

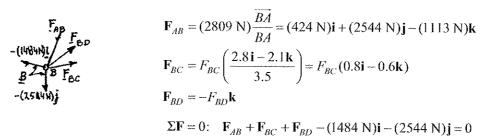
$$-\left(\frac{16.8}{5.20}\right)F_{AC} - 6(2968 \text{ N}) = 0, \quad F_{AC} = -5512 \text{ N}$$
  $F_{AC} = 5510 \text{ N} \quad C \blacktriangleleft$ 

Substitute for  $F_{AC}$  and  $F_{AD}$  in (2):

$$-\left(\frac{4.8}{5.30}\right)2F_{AB} - \left(\frac{4.8}{5.20}\right)(-5512 \text{ N}) = 0, \quad F_{AB} = +2809 \text{ N}$$

 $F_{AB} = F_{AD} = 2810 \text{ N}$  T

Free body: B



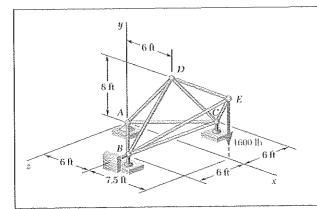
Substituting for  $\mathbf{F}_{AB}$ ,  $\mathbf{F}_{BC}$ ,  $\mathbf{F}_{BD}$  and equating to zero the coefficients of  $\mathbf{i}$  and  $\mathbf{k}$ :

i: 
$$+24 \text{ N} + 0.8 F_{BC} - 1484 \text{ N} = 0$$
,  $F_{BC} = +1325 \text{ N}$   $F_{BC} = 1325 \text{ N}$   $T \blacktriangleleft$ 

**k**: 
$$-1113 \text{ N} - 0.6F_{BC} - F_{BD} = 0$$

$$F_{BD} = -1113 \text{ N} - 0.6(1325 \text{ N}) = -1908 \text{ N}, \qquad F_{BD} = 1908 \text{ N} \quad C \blacktriangleleft$$

From symmetry: 
$$F_{CD} = F_{BC}$$
  $F_{CD} = 1325 \text{ N} \quad T \blacktriangleleft$ 



#### **PROBLEM 6.38\***

The truss shown consists of nine members and is supported by a ball and socket at A, two short links at B, and a short link at C. Determine the force in each of the members for the given loading.

### SOLUTION

Free body: Truss

From symmetry:

$$A_{z} = B_{z} = 0$$

$$\Sigma F_{\rm v} = 0$$
:  $A_{\rm v} = 0$ 

$$\Sigma M_{BC} = 0$$
:  $A_v(6 \text{ ft}) + (1600 \text{ lb})(7.5 \text{ ft}) = 0$ 

$$A_v = -2000 \text{ lb}$$

 $A = -(2000 \text{ lb})j \le$ 

From symmetry:

$$B_y = C$$

$$\Sigma F_{y} = 0$$
:  $2B_{y} - 2000 \text{ lb} - 1600 \text{ lb} = 0$ 

$$B_v = 1800 \text{ lb}$$

 $\mathbf{B} = (1800 \text{ lb})\mathbf{j} \triangleleft$ 

Free body: A

$$\Sigma \mathbf{F} = 0$$
:  $\mathbf{F}_{AB} + \mathbf{F}_{AC} + \mathbf{F}_{AD} - (2000 \text{ lb}) \mathbf{j} = 0$ 

$$F_{AB} \frac{\mathbf{i} + \mathbf{k}}{\sqrt{2}} + F_{AC} \frac{\mathbf{i} - \mathbf{k}}{\sqrt{2}} + F_{AD} (0.6\mathbf{i} + 0.8\mathbf{j}) - (2000 \text{ lb})\mathbf{j} = 0$$

A FAC

Factoring i, j, k and equating their coefficient to zero:

$$\frac{1}{\sqrt{2}}F_{AB} + \frac{1}{\sqrt{2}}F_{AC} + 0.6F_{AD} = 0 \tag{1}$$

$$0.8F_{AD} - 2000 \text{ lb} = 0$$

$$F_{4D} = 2500 \text{ lb}$$
 T

$$\frac{1}{\sqrt{2}}F_{AB} - \frac{1}{\sqrt{2}}F_{AC} = 0$$
  $F_{AC} = F_{AB}$ 

### PROBLEM 6.38\* (Continued)

Substitute for  $F_{AD}$  and  $F_{AC}$  into (1):

$$\frac{2}{\sqrt{2}}F_{AB} + 0.6(2500 \text{ lb}) = 0, \quad \mathbf{F}_{AB} = -1060.7 \text{ lb}, \qquad F_{AB} = F_{AC} = 1061 \text{ lb} \quad C \blacktriangleleft$$

$$F_{AB} = F_{AC} = 1061 \,\text{lb} \quad C \blacktriangleleft$$

Free body: B

$$\mathbf{F}_{BA} = F_{AB} \frac{\overrightarrow{BA}}{BA} = +(1060.7 \text{ lb}) \frac{\mathbf{i} + \mathbf{k}}{\sqrt{2}} = (750 \text{ lb})(\mathbf{i} + \mathbf{k})$$

$$\mathbf{F}_{BC} = -F_{BC}\mathbf{k}$$

B=(1800 L) 1

$$\mathbf{F}_{BD} = F_{BD}(0.8\mathbf{j} - 0.6\mathbf{k})$$

$$\mathbf{F}_{BE} = F_{BE} \frac{\overline{BE}}{BE} = \frac{F_{BE}}{12.5} (7.5\mathbf{i} + 8\mathbf{j} - 6\mathbf{k})$$

$$\Sigma \mathbf{F} = 0$$
:  $\mathbf{F}_{BA} + \mathbf{F}_{BC} + \mathbf{F}_{BD} + \mathbf{F}_{BE} + (1800 \text{ lb})\mathbf{j} = 0$ 

Substituting for  $\mathbf{F}_{BA}$ ,  $\mathbf{F}_{BC}$ ,  $\mathbf{F}_{BD}$  +  $\mathbf{F}_{BE}$  and equate to zero the coefficients of  $\mathbf{i}$ ,  $\mathbf{j}$ ,  $\mathbf{k}$ :

i: 
$$750 \text{ lb} + \left(\frac{7.5}{12.5}\right) F_{BE} = 0$$
,  $F_{BE} = -1250 \text{ lb}$ 

$$F_{BE} = 1250 \text{ lb}$$
  $C \blacktriangleleft$ 

j: 
$$0.8 F_{BD} + \left(\frac{8}{12.5}\right) (-1250 \text{ lb}) + 1800 \text{ lb} = 0$$

$$F_{BD} = 1250 \, \text{lb} \quad C \blacktriangleleft$$

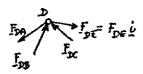
**k**: 
$$750 \text{ lb} - F_{BC} - 0.6(-1250 \text{ lb}) - \frac{6}{12.5}(-1250 \text{ lb}) = 0$$

$$F_{RC} = 2100 \, \text{lb} \quad T \blacktriangleleft$$

From symmetry:

$$F_{BD} = F_{CD} = 1250 \text{ lb}$$
 C

Free body: D



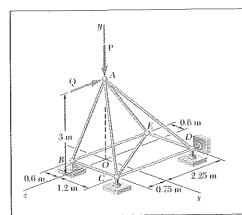
$$\Sigma \mathbf{F} = 0$$
:  $\mathbf{F}_{DA} + \mathbf{F}_{DB} + \mathbf{F}_{DC} + \mathbf{F}_{DE} \mathbf{i} = 0$ 

We now substitute for  $\mathbf{F}_{DA}$ ,  $\mathbf{F}_{DB}$ ,  $\mathbf{F}_{DC}$  and equate to zero the coefficient of  $\mathbf{i}$ . Only  $\mathbf{F}_{DA}$  contains  $\mathbf{i}$  and its coefficient is

$$-0.6 F_{4D} = -0.6(2500 \text{ lb}) = -1500 \text{ lb}$$

i: 
$$-1500 \text{ lb} + F_{DE} = 0$$

$$F_{DE} = 1500 \text{ lb}$$
  $T \blacktriangleleft$ 



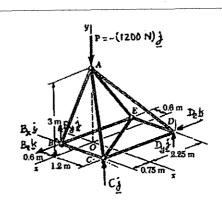
### **PROBLEM 6.39\***

The truss shown consists of nine members and is supported by a ball and socket at B, a short link at C, and two short links at D. (a) Check that this truss is a simple truss, that it is completely constrained, and that the reactions at its supports are statically determinate. (b) Determine the force in each member for P = (-1200 N)j and Q = 0.

### **SOLUTION**

Free body: Truss

$$\begin{split} \mathbf{\Sigma M}_{B} &= 0 \colon \quad 1.8 \mathbf{i} \times C \mathbf{j} + (1.8 \mathbf{i} - 3 \mathbf{k}) \times (D_{y} \mathbf{j} + D \mathbf{k}) \\ &+ (0.6 \mathbf{i} - 0.75 \mathbf{k}) \times (-1200 \mathbf{j}) = 0 \\ &- 1.8 \ C \mathbf{k} + 1.8 \ D_{y} \mathbf{k} - 1.8 \ D_{z} \mathbf{j} \\ &+ 3 D_{y} \mathbf{i} - 720 \mathbf{k} - 900 \mathbf{i} = 0 \end{split}$$



Equate to zero the coefficients of i, j, k:

i: 
$$3D_y - 900 = 0$$
,  $D_y = 300 \text{ N}$ 

$$\mathbf{j}: \qquad D_z = 0,$$

$$\mathbf{D} = (300 \text{ N})\mathbf{j} \le$$

k: 
$$1.8C + 1.8(300) - 720 = 0$$

$$\mathbf{C} = (100 \text{ N})\mathbf{i} \triangleleft$$

$$\Sigma \mathbf{F} = 0$$
:  $\mathbf{B} + 300\mathbf{j} + 100\mathbf{j} - 1200\mathbf{j} = 0$ 

$$B = (800 \text{ N})i \triangleleft$$

Free body: B



$$\Sigma \mathbf{F} = 0$$
:  $\mathbf{F}_{BA} + \mathbf{F}_{BC} + \mathbf{F}_{BE} + (800 \text{ N})\mathbf{j} = 0$ , with

$$\mathbf{F}_{BA} = \mathbf{F}_{AB} \frac{\overrightarrow{BA}}{BA} = \frac{F_{AB}}{3.15} (0.6\mathbf{i} + 3\mathbf{j} - 0.75\mathbf{k})$$

$$\mathbf{F}_{BC} = F_{BC}\mathbf{i} \qquad \qquad \mathbf{F}_{BE} = -F_{BE}\mathbf{k}$$

Substitute and equate to zero the coefficient of j, i, k:

j: 
$$\left(\frac{3}{3.315}\right)F_{AB} + 800 \text{ N} = 0$$
,  $F_{AB} = -840 \text{ N}$ ,

$$F_{AB} = 840 \text{ N}$$
  $C \blacktriangleleft$ 

i: 
$$\left(\frac{0.6}{3.15}\right)(-840 \text{ N}) + F_{BC} = 0$$

$$F_{BC} = 160.0 \,\mathrm{N} \quad T \blacktriangleleft$$

k: 
$$\left(-\frac{0.75}{3.15}\right)(-840 \text{ N}) - F_{BE} = 0$$

$$F_{BE} = 200 \text{ N}$$
  $T \blacktriangleleft$ 

### PROBLEM 6.39\* (Continued)

Free body C:

$$\Sigma \mathbf{F} = 0$$
:  $\mathbf{F}_{CA} + \mathbf{F}_{CB} + \mathbf{F}_{CD} + \mathbf{F}_{CE} + (100 \text{ N})\mathbf{j} = 0$ , with

$$\mathbf{F}_{CA} = F_{AC} \frac{\overline{CA}}{CA} = \frac{F_{AC}}{3.317} (-1.2\mathbf{i} + 3\mathbf{j} - 0.75\mathbf{k})$$

$$F_{CB} = -(160 \,\mathrm{N}) \,\mathrm{i}$$

$$\mathbf{F}_{CD} = -F_{CD}\mathbf{k}$$
  $\mathbf{F}_{CE} = \mathbf{F}_{CE}\frac{\overline{CE}}{CE} = \frac{F_{CE}}{3.499}(-1.8\mathbf{i} - 3\mathbf{k})$ 

Substitute and equate to zero the coefficient of j, i, k:

j: 
$$\left(\frac{3}{3.317}\right)F_{AC} + 100 \text{ N} = 0, \quad F_{AC} = -110.57 \text{ N}$$

$$F_{AC} = 110.6 \text{ N} \cdot C \blacktriangleleft$$

i: 
$$-\frac{1.2}{3.317}(-110.57) - 160 - \frac{1.8}{3.499}F_{CE} = 0$$
,  $F_{CE} = -233.3$ 

$$F_{CE} = 233 \text{ N} \cdot C \blacktriangleleft$$

**k**: 
$$-\frac{0.75}{3.317}(-110.57) - F_{CD} - \frac{3}{3.499}(-233.3) = 0$$

$$F_{CD} = 225 \text{ N} \cdot T$$

Free body: D



$$\Sigma \mathbf{F} = 0$$
:  $\mathbf{F}_{DA} + \mathbf{F}_{DC} + \mathbf{F}_{DE} + (300 \text{ N})\mathbf{j} = 0$ , with

$$\mathbf{F}_{DA} = F_{AD} \frac{\overrightarrow{DA}}{DA} = \frac{F_{AD}}{3.937} (-1.2\mathbf{i} + 3\mathbf{j} + 2.25\mathbf{k})$$

$$\mathbf{F}_{DC} = F_{CD}\mathbf{k} = (225 \text{ N})\mathbf{k}$$
  $F_{DE} = -F_{DE}\mathbf{i}$ 

Substitute and equate to zero the coefficient of j, i, k:

j: 
$$\left(\frac{3}{3.937}\right)F_{AD} + 300 \text{ N} = 0, \qquad F_{AD} = -393.7 \text{ N},$$

$$F_{AD} = -393.7 \text{ N}$$

$$F_{AD} = 394 \text{ N} \quad C \blacktriangleleft$$

i: 
$$\left(-\frac{1.2}{3.937}\right)(-393.7 \text{ N}) - F_{DE} = 0$$

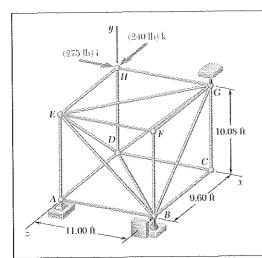
$$F_{DE} = 1200 \text{ N}$$
  $T \blacktriangleleft$ 

**k**: 
$$\left(\frac{2.25}{3.937}\right)(-393.7 \text{ N}) + 225 \text{ N} = 0$$
 (Checks)

Free body: E

Member AE is the only member at E which does not lie in the xz plane. Therefore, it is a zero-force member.





### **PROBLEM 6.41\***

The truss shown consists of 18 members and is supported by a ball and socket at A, two short links at B, and one short link at G. (a) Check that this truss is a simple truss, that it is completely constrained, and that the reactions at its supports are statically determinate. (b) For the given loading, determine the force in each of the six members joined at E.

#### **SOLUTION**

#### (a) Check simple truss.

- (1) start with tetrahedron BEFG
- (2) Add members BD, ED, GD joining at D.
- (3) Add members BA, DA, EA joining at A.
- (4) Add members DH, EH, GH joining at H.
- (5) Add members BC, DC, GC joining at C.

Truss has been completed: It is a simple truss

Free body: Truss

#### Check constraints and reactions:

Six unknown reactions-ok-however supports at A and B constrain truss to rotate about AB and support at G prevents such a rotation. Thus,

Truss is completely constrained and reactions are statically determinate

#### Determination of Reactions:

$$\Sigma \mathbf{M}_{A} = 0: \quad 11\mathbf{i} \times (B_{y}\mathbf{j} + B_{z}\mathbf{k}) + (11\mathbf{i} - 9.6\mathbf{k}) \times G\mathbf{j}$$

$$+ (10.08\mathbf{j} - 9.6\mathbf{k}) \times (275\mathbf{i} + 240\mathbf{k}) = 0$$

$$11B_{y}\mathbf{k} - 11B_{y}\mathbf{j} + 11G\mathbf{k} + 9.6G\mathbf{i} - (10.08)(275)\mathbf{k}$$

$$+ (10.08)(240)\mathbf{i} - (9.6)(275)\mathbf{j} = 0$$

Equate to zero the coefficient of i, j, k:

e coefficient of i, j, k:  
i: 
$$9.6G + (10.08)(240) = 0$$
  $G = -252$  lb

$$G = (-252 \text{ lb})j \triangleleft$$

j: 
$$-11B_z - (9.6)(275) = 0$$
  $B_z = -240 \text{ lb}$ 

**k**: 
$$11B_y + 11(-252) - (10.08)(275) = 0$$
,  $B_y = 504 \text{ lb}$  **B** =  $(504 \text{ lb})\mathbf{j} - (240 \text{ lb})\mathbf{k}$ 

$$\mathbf{B} = (504 \text{ lb})\mathbf{j} - (240 \text{ lb})\mathbf{k} \le$$

# PROBLEM 6.41\* (Continued)

$$\Sigma$$
F = 0: **A** + (504 lb)**j** - (240 lb)**k** - (252 lb)**j**  
+ (275 lb)**i** + (240 lb)**k** = 0

$$A = -(275 \text{ lb})\mathbf{i} - (252 \text{ lb})\mathbf{j} \triangleleft$$

### Zero-force members

The determination of these members will facilitate our solution.

$$\Sigma F_x = 0$$
,  $\Sigma F_y = 0$ ,  $\Sigma F_z = 0$ 

Yields 
$$F_{BC} = F_{CD} = F_{CG} = 0 \triangleleft$$

$$\Sigma F_x = 0$$
,  $\Sigma F_y = 0$ ,  $\Sigma F_z = 0$ 

Yields 
$$F_{BF} = F_{EF} = F_{FG} = 0$$

$$A_z = 0$$
, writing  $\Sigma F_z = 0$ 

Yields 
$$F_{AD} = 0 \triangleleft$$

$$\Sigma F_v = 0$$

Yields 
$$F_{DH} = 0 \triangleleft$$

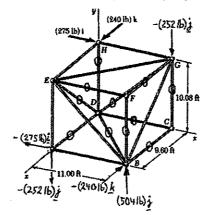
<u>FB</u>: <u>D</u>: Since  $F_{AD} = F_{CD} = F_{DH} = 0$ , we need consider only members DB, DE, and DG.

Since  $F_{DE}$  is the only force not contained in plane BDG, it must be zero. Simple reasonings show that the other two forces are also zero.



$$F_{BD} = F_{DE} = F_{DG} = 0 < 1$$

The results obtained for the reactions at the supports and for the zero-force members are shown on the figure below. Zero-force members are indicated by a zero ("0").



# (b) Force in each of the members joined at E

We already found that

$$F_{DE} = F_{EF} = 0$$

$$\Sigma F_y = 0$$

Yields 
$$F_{AE} = 252 \text{ lb}$$
  $T \blacktriangleleft$ 

$$\Sigma F_{*} = 0$$

Yields 
$$F_{EH} = 240 \text{ lb}$$
  $C \blacktriangleleft$ 

### PROBLEM 6.41\* (Continued)

Free body: E

$$\Sigma \mathbf{F} = 0$$
:  $\mathbf{F}_{EB} + \mathbf{F}_{EG} + (240 \text{ lb})\mathbf{k} - (252 \text{ lb})\mathbf{j} = 0$ 

$$\frac{F_{BE}}{14.92}(11\mathbf{i} - 10.08\mathbf{j}) + \frac{F_{EG}}{14.6}(11\mathbf{i} - 9.6\mathbf{k}) + 240\mathbf{k} - 252\mathbf{j} = 0$$

FEA = (240 16)

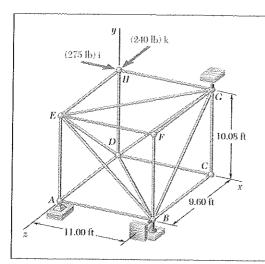
Equate to zero the coefficient of y and k:

**j**: 
$$-\left(\frac{10.08}{14.92}\right)F_{BE} - 252 = 0$$

$$F_{BE} = 373 \, \text{lb} \quad C \blacktriangleleft$$

**k**: 
$$-\left(\frac{9.6}{14.6}\right)F_{EG} + 240 = 0$$

$$F_{EG} = 365 \text{ lb}$$
  $T \blacktriangleleft$ 



# **PROBLEM 6.42\***

The truss shown consists of 18 members and is supported by a ball and socket at A, two short links at B, and one short link at G. (a) Check that this truss is a simple truss, that it is completely constrained, and that the reactions at its supports are statically determinate. (b) For the given loading, determine the force in each of the six members joined at G.

## **SOLUTION**

See solution to Problem 6.41 for Part (a) and for reactions and zero-force members.

(b) Force in each of the members joined at G.

We already know that

$$F_{CG} = F_{DG} = F_{FG} = 0$$

$$\Sigma F_x = 0$$

Yields 
$$F_{GH} = 275 \, \text{lb}$$
  $C \blacktriangleleft$ 

$$\Sigma \mathbf{F} = 0$$
:  $\mathbf{F}_{GB} + \mathbf{F}_{GE} + (275 \text{ lb})\mathbf{i} - (252 \text{ lb})\mathbf{j} = 0$ 

$$\frac{F_{BG}}{13.92}(-10.08\mathbf{j} + 9.6\mathbf{k}) + \frac{F_{EG}}{14.6}(-11\mathbf{i} + 9.6\mathbf{k}) + 275\mathbf{i} - 252\mathbf{j} = 0$$

FGE (275/6) i FGE (252/6) j

Equate to zero the coefficient of i, j, k:

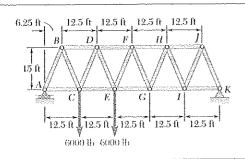
i: 
$$-\left(\frac{11}{14.6}\right)F_{EG} + 275 = 0$$

$$F_{EG} = 365 \text{ lb}$$
  $T \blacktriangleleft$ 

**j**: 
$$-\left(\frac{10.08}{13.92}\right)F_{BG} - 252 = 0$$

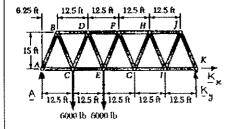
$$F_{RG} = 348 \, \text{lb} \quad C \blacktriangleleft$$

**k**: 
$$\left(\frac{9.6}{13.92}\right)(-348) + \left(\frac{9.6}{14.6}\right)(365) = 0$$
 (Checks)



A Warren bridge truss is loaded as shown. Determine the force in members CE, DE, and DF.

## SOLUTION



Free body: Truss

$$\pm \Sigma F_x = 0$$
:  $\mathbf{k}_x = 0$   
 $\pm \Sigma M_A = 0$ :  $k_y (62.5 \text{ ft}) - (6000 \text{ lb})(12.5 \text{ ft})$   
 $- (6000 \text{ lb})(25 \text{ ft}) = 0$ 

$$\mathbf{k} = \mathbf{k}_y = 3600 \text{ lb}$$

$$+ \sum F_y = 0$$
:  $A + 3600 \text{ lb} - 6000 \text{ lb} - 6000 \text{ lb} = 0$ 

 $\mathbf{A} = 8400 \text{ lb}^{\dagger} \triangleleft$ 

We pass a section through members CE, DE, and DF and use the free body shown.

+)
$$\Sigma M_D = 0$$
:  $F_{CE}$  (15 ft) - (8400 lb)(18.75 ft)  
+ (6000 lb)(6.25 ft) = 0

$$F_{CE} = +8000 \text{ lb}$$

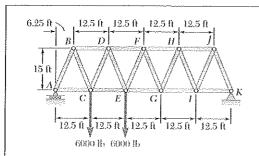
$$F_{CE} = 8000 \, \text{lb}$$
  $T \blacktriangleleft$ 

+ 
$$\Sigma F_y = 0$$
: 8400 lb - 6000 lb -  $\frac{15}{16.25} F_{DE} = 0$ 

$$F_{DE} = +2600 \text{ lb}$$
  $F_{DE} = 2600 \text{ lb}$   $T \blacktriangleleft$ 

+)
$$\Sigma M_E = 0$$
: 6000 lb(12.5 ft) – (8400 lb)(25 ft)  
- $F_{DF}$ (15 ft) = 0

$$F_{DF} = -9000 \text{ lb}$$
  $F_{DF} = 9000 \text{ lb}$   $C \blacktriangleleft$ 



1250 K=3600 lb

# PROBLEM 6.44

A Warren bridge truss is loaded as shown. Determine the force in members EG, FG, and FH.

#### SOLUTION

See solution of Problem 6.43 for free-body diagram of truss and determination of reactions:

$$A = 8400 \text{ lb}$$

 $k = 3600 \text{ lb} \triangleleft$ 

We pass a section through members EG, FG, and FH, and use the free body shown.

+)
$$\Sigma M_F = 0$$
: (3600 lb)(31.25 ft) -  $F_{EG}$ (15 ft) = 0

 $F_{EG} = +7500 \, \text{lb}$ 

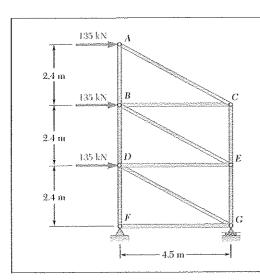
 $F_{EG} = 7500 \, \text{lb} \, T \, \blacktriangleleft$ 

$$+\Sigma F_y = 0$$
:  $\frac{15}{16.25}F_{FG} + 3600 \text{ lb} = 0$ 

$$F_{FG} = -3900 \text{ lb}$$
  $F_{FG} = 3900 \text{ lb}$   $C \blacktriangleleft$ 

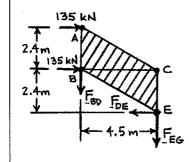
+)
$$\Sigma M_G = 0$$
:  $F_{FH}$  (15 ft) + (3600 lb)(25 ft) = 0

$$F_{FH} = -6000 \ \mathrm{lb}$$
  $F_{FH} = 6000 \ \mathrm{lb}$   $C$ 



Determine the force in members BD and DE of the truss shown.

# **SOLUTION**



#### Member BD:

+)
$$\Sigma M_E = 0$$
:  $F_{BD}(4.5 \text{ m}) - (135 \text{ kN})(4.8 \text{ m}) - (135 \text{ kN})(2.4 \text{ m}) = 0$ 

$$F_{RD} = +216 \text{ kN}$$

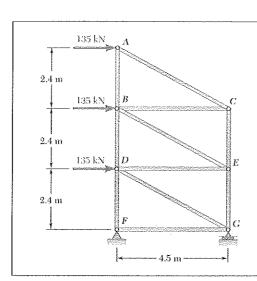
$$F_{BD} = +216 \text{ kN}$$
  $F_{BD} = 216 \text{ kN}$   $T \blacktriangleleft$ 

## Member DE:

$$\pm \Sigma F_x = 0$$
: 135 kN + 135 kN -  $F_{DE} = 0$ 

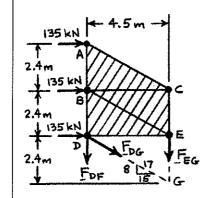
$$F_{DE} = +270 \text{ kN}$$

$$F_{DE} = 270 \text{ kN}$$
  $T \blacktriangleleft$ 



Determine the force in members DG and EG of the truss shown.

## **SOLUTION**



Member *DG*:

$$\pm \Sigma F_x = 0$$
: 135 kN + 135 kN + 135 kN +  $\frac{15}{17} F_{DG} = 0$ 

$$F_{DG} = -459 \text{ kN}$$

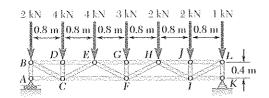
 $F_{DG} = 459 \text{ kN} \quad C \blacktriangleleft$ 

Member EG:

+)
$$\Sigma M_D = 0$$
: (135 kN)(4.8 m) + (135 kN)(2.4 m)  
+ $F_{EG}$ (4.5 m) = 0

$$F_{\rm EC} = -216 \, \rm kN$$

 $F_{EG} = -216 \text{ kN}$   $F_{EG} = 216 \text{ kN}$   $C \blacktriangleleft$ 



A floor truss is loaded as shown. Determine the force in members *CF*, *EF*, and *EG*.

### **SOLUTION**

Free body: Truss

 $k_{\nu} = 7.5 \, \text{kN}$ 

Thus:

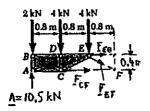
 $k = 7.5 \text{ kN} \uparrow \triangleleft$ 

$$+\int \Sigma F_v = 0$$
:  $A + 7.5 \text{ kN} - 18 \text{ kN} = 0$   $A = 10.5 \text{ kN}$ 

 $A = 10.5 \text{ kN} \uparrow \triangleleft$ 

We pass a section through members CF, EF, and EG and use the free body shown.

$$+)\Sigma M_E = 0$$
:  $F_{CF}(0.4 \text{ m}) - (10.5 \text{ kN})(1.6 \text{ m}) + (2 \text{ kN})(1.6 \text{ m}) + (4 \text{ kN})(0.8 \text{ m}) = 0$ 



$$F_{CF} = +26.0 \text{ kN}$$

 $F_{CF} = 26.0 \,\mathrm{kN}$  T

$$+\frac{1}{2}\Sigma F_y = 0$$
: 10.5 kN - 2 kN - 4 kN - 4 kN -  $\frac{1}{\sqrt{5}}F_{EF} = 0$ 

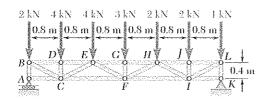
$$F_{EF} = +1.118 \,\mathrm{kN}$$

 $F_{EF} = 1.118 \,\mathrm{kN}$  T

$$+\sum \Sigma M_F = 0: \quad (2 \text{ kN})(2.4 \text{ m}) + (4 \text{ kN})(1.6 \text{ m}) + (4 \text{ kN})(0.8 \text{ m}) - (10.5 \text{ kN})(2.4 \text{ m}) - F_{EG}(0.4 \text{ m}) = 0$$

$$F_{EG} = -27.0 \text{ kN}$$

 $F_{EG} = 27.0 \text{ kN}$  C



A floor truss is loaded as shown. Determine the force in members FI, HI, and HJ.

#### SOLUTION

See solution of Problem 6.47 for free-body diagram of truss and determination of reactions:

$$A = 10.5 \text{ kN} \uparrow$$
,  $k = 7.5 \text{ kN} \uparrow \triangleleft$ 

We pass a section through members FI, HI, and HJ, and use the free body shown.

+) 
$$\Sigma M_H = 0$$
:  $(7.5 \text{ kN})(1.6 \text{ m}) - (2 \text{ kN})(0.8 \text{ m}) - (1 \text{ kN})(1.6 \text{ m})$   
 $-F_{FI}(0.4 \text{ m}) = 0$ 

$$F_{FI} = +22.0 \text{ kN}$$

$$F_{FI} = 22.0 \, \text{kN}$$
 T

$$+\int \Sigma F_y = 0$$
:  $\frac{1}{\sqrt{5}} F_{HI} - 2 \text{ kN} - 1 \text{ kN} + 7.5 \text{ kN} = 0$ 

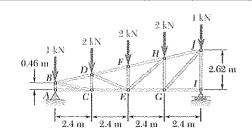
$$F_{HI} = -10.06 \text{ kN}$$

$$F_{HI} = 10.06 \, \text{kN} \cdot C$$

+)
$$\Sigma M_I = 0$$
:  $F_{IIJ}(0.4 \text{ m}) + (7.5 \text{ kN})(0.8 \text{ m}) - (1 \text{ kN})(0.8 \text{ m}) = 0$ 

$$F_{HJ} = -13.00 \text{ kN}$$

$$F_{IIJ} = 13.00 \text{ kN} \cdot C \blacktriangleleft$$



A pitched flat roof truss is loaded as shown. Determine the force in members CE, DE, and DF.

#### **SOLUTION**

Reactions at supports: Because of the symmetry of the loading,

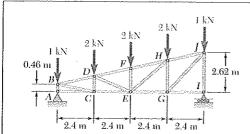
$$A_x = 0$$
  
 $A_y = I = \frac{1}{2} \text{ (Total load)} = \frac{1}{2} (8 \text{ kN})$   $\mathbf{A} = \mathbf{I} = 4 \text{ kN} \uparrow \triangleleft$ 

We pass a section through members CD, DE, and DF, and use the <u>free body shown</u>. (We moved  $F_{DE}$  to E and  $F_{DF}$  to F)

Slope 
$$BJ = \frac{2.16 \text{ m}}{9.6 \text{ m}} = \frac{9}{40}$$
  $\frac{41}{10}$   $\frac{9}{12}$   $\frac{1}{10}$   $\frac{1$ 

 $F_{DF} = -6.39 \text{ kN}$ 

 $F_{DE} = 6.39 \, \text{kN} \cdot C \, \blacktriangleleft$ 



A pitched flat roof truss is loaded as shown. Determine the force in members EG, GH, and HJ.

#### SOLUTION

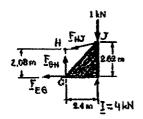
Reactions at supports: Because of the symmetry of the loading,

$$A_{x}=0$$

$$A_y = I = \frac{1}{2} \text{ (Total load)} = \frac{1}{2} (8 \text{ kN})$$

 $A = I = 4 \text{ kN} \uparrow \triangleleft$ 

We pass a section through members EG, GH, and HJ, and use the free body shown.



+)
$$\Sigma M_H = 0$$
:  $(4 \text{ kN})(2.4 \text{ m}) - (1 \text{ kN})(2.4 \text{ m}) - F_{EG}(2.08 \text{ m}) = 0$ 

$$F_{EG} = +3.4615 \text{ kN}$$

 $F_{EG} = 3.46 \text{ kN}$   $T \blacktriangleleft$ 

+)
$$\Sigma M_J = 0$$
:  $-F_{GH}(2.4 \text{ m}) - F_{EG}(2.62 \text{ m}) = 0$ 

$$F_{GH} = -\frac{2.62}{2.4} (3.4615 \text{ kN})$$

$$F_{GH} = -3.7788 \text{ kN}$$

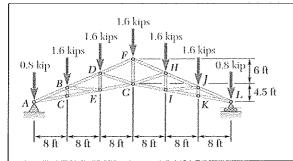
 $F_{GH} = 3.78 \, \text{kN} \, C \blacktriangleleft$ 

$$+\Sigma F_x = 0$$
:  $-F_{EG} - \frac{2.4}{2.46} F_{HJ} = 0$ 

$$F_{HJ} = -\frac{2.46}{2.4} F_{EG} = -\frac{2.46}{2.4} (3.4615 \text{ kN})$$

$$F_{HJ} = -3.548 \text{ kN}$$

$$F_{HJ} = 3.55 \,\mathrm{kN}$$
 C



A Howe scissors roof truss is loaded as shown. Determine the force in members DF, DG, and EG.

## SOLUTION

Reactions at supports.

Because of symmetry of loading.

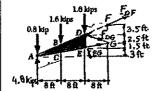
$$A_x = 0$$
,  $A_y = L = \frac{1}{2}$  (Total load) =  $\frac{1}{2}$  (9.60 kips) = 4.80 kips

A = L = 4.80 kips

We pass a section through members DF, DG, and EG, and use the free body shown.

We slide  $\mathbf{F}_{DF}$  to apply it at F:

+) 
$$\Sigma M_G = 0$$
:  $(0.8 \text{ kip})(24 \text{ ft}) + (1.6 \text{ kips})(16 \text{ ft}) + (1.6 \text{ kips})(8 \text{ ft})$   
-  $(4.8 \text{ kips})(24 \text{ ft}) - \frac{8F_{DF}}{\sqrt{8^2 + 3.5^2}}(6 \text{ ft}) = 0$ 



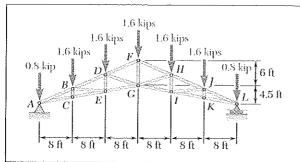
$$F_{DF} = -10.48 \text{ kips}, \quad F_{DF} = 10.48 \text{ kips} \quad C \blacktriangleleft$$

+)
$$\Sigma M_A = 0$$
:  $-(1.6 \text{ kips})(8 \text{ ft}) - (1.6 \text{ kips})(16 \text{ ft})$   
$$-\frac{2.5F_{DG}}{\sqrt{8^2 + 2.5^2}}(16 \text{ ft}) - \frac{8F_{DG}}{\sqrt{8^2 + 2.5^2}}(7 \text{ ft}) = 0$$

$$F_{DG} = -3.35$$
 kips,  $F_{DG} = 3.35$  kips  $C \blacktriangleleft$ 

+) 
$$\Sigma M_D = 0$$
:  $(0.8 \text{ kips})(16 \text{ ft}) + (1.6 \text{ kips})(8 \text{ ft}) - (4.8 \text{ kips})(16 \text{ ft}) - \frac{8F_{EG}}{\sqrt{8^2 + 1.5^2}}(4 \text{ ft}) = 0$ 

$$F_{EG} = +13.02 \text{ kips}, \quad F_{EG} = 13.02 \text{ kips} \quad T \blacktriangleleft$$



A Howe scissors roof truss is loaded as shown. Determine the force in members *GI*, *HI*, and *HJ*.

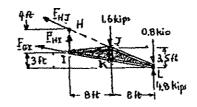
## SOLUTION

Reactions at supports. Because of symmetry of loading:

$$A_x = 0,$$
  $A_y = L = \frac{1}{2} \text{(Total load)}$   
=  $\frac{1}{2} \text{(9.60 kips)}$   
= 4.80 kips

A = L = 4.80 kips

We pass a section through members GI, HI, and HJ, and use the free body shown.



+) 
$$\Sigma M_H = 0$$
:  $-\frac{16F_{GI}}{\sqrt{16^2 + 3^2}}$  (4 ft) + (4.8 kips)(16 ft) – (0.8 kip)(16 ft) – (1.6 kips)(8 ft) = 0

$$F_{GI} = +13.02 \text{ kips}$$

$$F_{GI} = 13.02 \text{ kips} \ T \blacktriangleleft$$

+)
$$\Sigma M_L = 0$$
: (1.6 kips)(8 ft) -  $F_{HI}$ (16 ft) = 0

$$F_{HI} = \pm 0.800 \text{ kips}$$

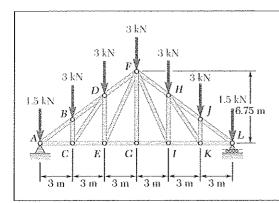
 $F_{HI} = 0.800 \text{ kips} \ T \blacktriangleleft$ 

We slide  $\mathbf{F}_{HG}$  to apply it at H.

+) 
$$\Sigma M_I = 0$$
:  $\frac{8F_{HJ}}{\sqrt{8^2 + 3.5^2}} (4 \text{ ft}) + (4.8 \text{ kips})(16 \text{ ft}) - (1.6 \text{ kips})(8 \text{ ft}) - (0.8 \text{ kip})(16 \text{ ft}) = 0$ 

$$F_{HJ} = -13.97 \text{ kips}$$

$$F_{HJ} = 13.97 \text{ kips} \ C \blacktriangleleft$$

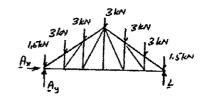


A Pratt roof truss is loaded as shown. Determine the force in members CE, DE, and DF.

## **SOLUTION**

Free body: Entire truss

$$\Sigma F_x = 0$$
:  $A_x = 0$   
Total load = 5(3 kN) + 2(1.5 kN)  
= 18 kN



By symmetry:

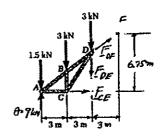
$$A_y = L = \frac{1}{2} (18 \text{ kN})$$

 $A = L = 9 \text{ kN } \blacktriangleleft$ 

Free body: Portion ACD

Slope of ABDF is Note:

$$\frac{6.75}{9.00} = \frac{3}{4}$$



Force in CE:

+)
$$\Sigma M_D = 0$$
:  $F_{CE} \left( \frac{2}{3} \times 6.75 \text{ m} \right) - (9 \text{ kN})(6 \text{ m}) + (1.5 \text{ kN})(6 \text{ m}) + (3 \text{ kN})(3 \text{ m}) = 0$ 

$$F_{CE}(4.5 \text{ m}) - 36 \text{ kN} \cdot \text{m} = 0$$

$$F_{CE} = +8 \text{ kN}$$

 $F_{CF} = 8 \text{ kN}$   $T \blacktriangleleft$ 

Force in DE:

+)
$$\Sigma M_A = 0$$
:  $F_{DE}(6 \text{ m}) + (3 \text{ kN})(6 \text{ m}) + (3 \text{ kN})(3 \text{ m}) = 0$ 

$$F_{DF} = -4.5 \text{ kN}$$

$$F_{DE} = -4.5 \text{ kN}$$
  $F_{DE} = 4.5 \text{ kN}$   $C \blacktriangleleft$ 

# PROBLEM 6.53 (Continued)

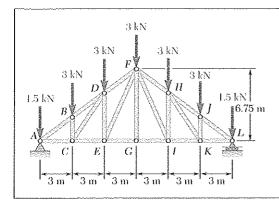
## Force in *DF*:

Sum moments about E where  $F_{CE}$  and  $F_{DE}$  intersect.

+) 
$$\Sigma M_E = 0$$
:  $(1.5 \text{ kN})(6 \text{ m}) - (9 \text{ kN})(6 \text{ m}) + (3 \text{ kN})(3 \text{ m}) + \frac{4}{5} F_{CE} \left(\frac{2}{3} \times 6.75 \text{ m}\right) = 0$   
$$\frac{4}{5} F_{CE} (4.5 \text{ m}) - 36 \text{ kN} \cdot \text{m}$$

$$F_{CE} = -10.00 \text{ kN}$$

 $F_{CE} = 10.00 \text{ kN}$  C



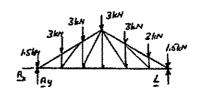
A Pratt roof truss is loaded as shown. Determine the force in members *FH*, *FI*, and *GI*.

# SOLUTION

Free body: Entire truss

$$\Sigma F_{\rm r} = 0$$
:  $A_{\rm r} = 0$ 

Total load = 
$$5(3 \text{ kN}) + 2(1.5 \text{ kN}) = 18 \text{ kN}$$



By symmetry:

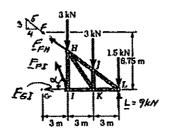
$$A_y = L = \frac{1}{2}(18) = 9 \text{ kN}$$

Free body: Portion HIL

Slope of FHJL

$$\frac{6.75}{9.00} = \frac{3}{4}$$

$$\tan \alpha = \frac{FG}{GI} = \frac{6.75 \text{ m}}{3 \text{ m}}$$
  $\alpha = 66.04^{\circ}$ 



Force in FH:

+)
$$\Sigma M_I = 0$$
:  $\frac{4}{5} F_{FH} \left( \frac{2}{3} \times 6.75 \text{ m} \right) + (9 \text{ kN})(6 \text{ m}) - (1.5 \text{ kN})(6 \text{ m}) - (3 \text{ kN})(3 \text{ m}) = 0$ 

$$\frac{4}{5}F_{FH}(4.5 \text{ m}) + 36 \text{ kN} \cdot \text{m}$$

$$F_{FH} = -10.00 \text{ kN}$$

$$F_{FH} = 10.00 \text{ kN} \cdot C \blacktriangleleft$$

Force in FI:

+)
$$\Sigma M_L = 0$$
:  $F_{FI} \sin \alpha (6 \text{ m}) - (3 \text{ kN})(6 \text{ m}) - (3 \text{ kN})(3 \text{ m}) = 0$ 

$$F_{FI} \sin 66.04^{\circ} (6 \text{ m}) = 27 \text{ kN} \cdot \text{m}$$

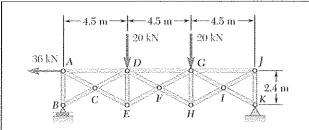
$$F_{FI} = +4.92 \text{ kN}$$

$$F_{FI} = 4.92 \text{ kN}$$
 T

# **PROBLEM 6.54 (Continued)**

Force in GI:

+) 
$$\Sigma M_H = 0$$
:  $F_{GI}(6.75 \text{ m}) + (3 \text{ kN})(3 \text{ m}) + (3 \text{ kN})(6 \text{ m}) + (1.5 \text{ kN})(9 \text{ m}) - (9 \text{ kN})(9 \text{ m}) = 0$   
 $F_{GI}(6.75 \text{ m}) = +40.5 \text{ kN} \cdot \text{m}$   
 $F_{GI} = +6.00 \text{ kN}$   $F_{GI} = 6.00 \text{ kN}$   $T$ 



Determine the force in members AD, CD, and CE of the truss shown.

# **SOLUTION**

Reactions:

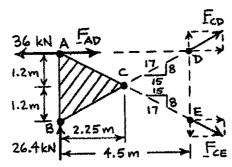
$$+)\Sigma M_k = 0$$
:  $36(2.4) - B(13.5) + 20(9) + 20(4.5) = 0$ 

B = 26.4 kN

$$\pm \Sigma F_{\rm v} = 0$$
:  $-36 + K_{\rm v} = 0$   $K_{\rm v} = 36 \text{ kN} \rightarrow$ 

$$K_v = 36 \text{ kN} \longrightarrow$$

$$+ \sum F_y = 0$$
:  $26.4 - 20 - 20 + K_y = 0$   $\mathbf{K}_y = 13.6 \text{ kN}$ 



+)
$$\Sigma M_C = 0$$
: 36(1.2) - 26.4(2.25) -  $F_{AD}$ (1.2) = 0

$$F_{4D} = -13.5 \text{ kN}$$

$$F_{AD} = -13.5 \text{ kN}$$
  $F_{AD} = 13.5 \text{ kN}$   $C \blacktriangleleft$ 

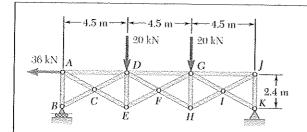
+)
$$\Sigma M_A = 0$$
:  $\left(\frac{8}{17}F_{CD}\right)(4.5) = 0$ 

$$F_{CD} = 0$$

+)
$$\Sigma M_D = 0$$
:  $\left(\frac{15}{17}F_{CE}\right)(2.4) - 26.4(4.5) = 0$ 

$$F_{CE} = +56.1 \text{ kN}$$

$$F_{CE} = 56.1 \text{ kN}$$
  $T \blacktriangleleft$ 

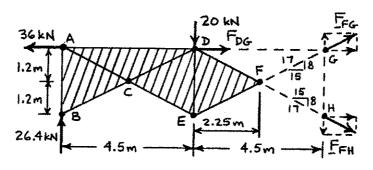


Determine the force in members DG, FG, and FH of the truss shown.

#### SOLUTION

See the solution to Problem 6.55 for free-body diagram and analysis to determine the reactions at the supports *B* and *K*.

$$\mathbf{B} = 26.4 \text{ kN}$$
;  $\mathbf{K}_x = 36.0 \text{ kN} \longrightarrow$ ;  $\mathbf{K}_y = 13.60 \text{ kN}$ 



+)
$$\Sigma M_F = 0$$
:  $36(1.2) - 26.4(6.75) + 20(2.25) - F_{DG}(1.2) = 0$ 

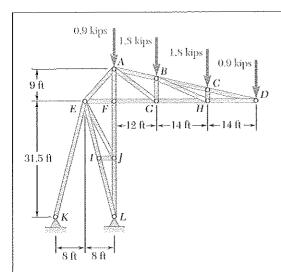
$$F_{DG} = -75 \text{ kN}$$
  $F_{DG} = 75.0 \text{ kN}$   $C \blacktriangleleft$ 

+)
$$\Sigma M_D = 0$$
:  $-26.4(4.5) + \left(\frac{8}{17}F_{FG}\right)(4.5) = 0$ 

$$F_{FG} = +56.1 \text{ kN}$$
  $F_{FG} = 56.1 \text{ kN}$   $T \blacktriangleleft$ 

+)
$$\Sigma M_G = 0$$
:  $20(4.5) - 26.4(9) + \left(\frac{15}{17}F_{FH}\right)(2.4) = 0$ 

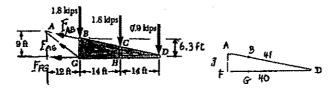
$$F_{FH}$$
 = +69.7 kN  $F_{FH}$  = 69.7 kN  $T$ 



A stadium roof truss is loaded as shown. Determine the force in members AB, AG, and FG.

# **SOLUTION**

We pass a section through members AB, AG, and FG, and use the free body shown.



$$+)\Sigma M_G = 0$$
:  $\left(\frac{40}{41}F_{AB}\right)(6.3 \text{ ft}) - (1.8 \text{ kips})(14 \text{ ft}) - (0.9 \text{ kips})(28 \text{ ft}) = 0$ 

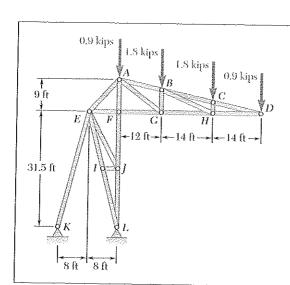
$$F_{AB} = +8.20 \text{ kips}$$
  $F_{AB} = 8.20 \text{ kips}$   $T$ 

+) 
$$\Sigma M_D = 0$$
:  $-\left(\frac{3}{5}F_{AG}\right)(28 \text{ ft}) + (1.8 \text{ kips})(28 \text{ ft}) + (1.8 \text{ kips})(14 \text{ ft}) = 0$ 

$$F_{AG} = +4.50 \text{ kips}$$
  $F_{AG} = 4.50 \text{ kips}$   $T \blacktriangleleft$ 

+) 
$$M_A = 0$$
:  $-F_{FG}(9 \text{ ft}) - (1.8 \text{ kips})(12 \text{ ft}) - (1.8 \text{ kips})(26 \text{ ft}) - (0.9 \text{ kips})(40 \text{ ft}) = 0$ 

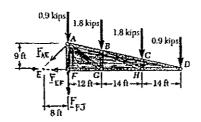
$$F_{FG} = -11.60 \text{ kips}$$
  $F_{FG} = 11.60 \text{ kips}$   $C \blacktriangleleft$ 



A stadium roof truss is loaded as shown. Determine the force in members AE, EF, and FJ.

# **SOLUTION**

We pass a section through members AE, EF, and FJ, and use the free body shown.



+) 
$$\Sigma M_F = 0$$
:  $\left(\frac{8}{\sqrt{8^2 + 9^2}} F_{AE}\right) (9 \text{ ft}) - (1.8 \text{ kips})(12 \text{ ft}) - (1.8 \text{ kips})(26 \text{ ft}) - (0.9 \text{ kips})(40 \text{ ft}) = 0$ 

$$F_{AE} = +17.46 \text{ kips}$$

$$F_{AE} = 17.46 \text{ kips}$$
  $T \blacktriangleleft$ 

+) 
$$\Sigma M_A = 0$$
:  $-F_{EF}$  (9 ft) - (1.8 kips)(12 ft) - (1.8 kips)(26 ft) - (0.9 kips)(40 ft) = 0

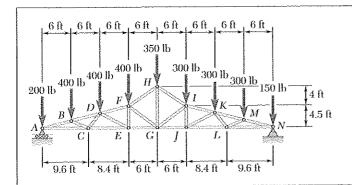
$$F_{EF} = -11.60 \text{ kips}$$

$$F_{EF} = 11.60 \text{ kips}$$
  $C \blacktriangleleft$ 

+) 
$$\Sigma M_E = 0$$
:  $-F_{FJ}(8 \text{ ft}) - (0.9 \text{ kips})(8 \text{ ft}) - (1.8 \text{ kips})(20 \text{ ft}) - (1.8 \text{ kips})(34 \text{ ft}) - (0.9 \text{ kips})(48 \text{ ft}) = 0$ 

$$F_{FJ} = -18.45 \text{ kips}$$

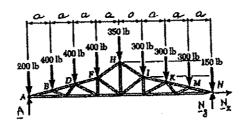
$$F_{FJ} = 18.45 \text{ kips}$$
 C



A Polynesian, or duopitch, roof truss is loaded as shown. Determine the force in members *DF*, *EF*, and *EG*.

## SOLUTION

Free body: Truss



$$\Sigma F_x = 0$$
:  $N_x = 0$ 

+) 
$$\Sigma M_N = 0$$
: (200 lb)(8a) + (400 lb)(7a + 6a + 5a) + (350 lb)(4a) + (300 lb)(3a + 2a + a) -  $A(8a) = 0$ 

 $A = 1500 \text{ lb} \uparrow \triangleleft$ 

$$+ \sum F_y = 0$$
: 1500 lb - 200 lb - 3(400 lb) - 350 lb - 3(300 lb) - 150 lb +  $N_y = 0$ 

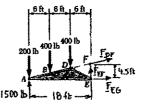
 $N_{v} = 1300 \text{ lb}$  N = 1300 lb | < |

We pass a section through DF, EF, and EG, and use the free body shown.

(We apply  $\mathbf{F}_{DF}$  at F)

+) 
$$\Sigma M_E = 0$$
: (200 lb)(18 ft) + (400 lb)(12 ft) + (400 lb)(6 ft) - (1500 lb)(18 ft)  

$$-\left(\frac{18}{\sqrt{18^2 + 4.5^2}}F_{DF}\right)(4.5 \text{ ft}) = 0$$



$$F_{DF} = -3711 \text{ lb}$$
  $F_{DF} = 3710 \text{ lb}$   $C \blacktriangleleft$ 

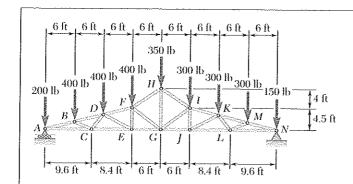
+)
$$\Sigma M_A = 0$$
:  $F_{EF}(18 \text{ ft}) - (400 \text{ lb})(6 \text{ ft}) - (400 \text{ lb})(12 \text{ ft}) = 0$ 

$$F_{FF} = +400 \text{ lb}$$

$$F_{FF} = 400 \text{ lb} \quad T \blacktriangleleft$$

+) 
$$\Sigma M_F = 0$$
:  $F_{EG}(4.5 \text{ ft}) - (1500 \text{ lb})(18 \text{ ft}) + (200 \text{ lb})(18 \text{ ft}) + (400 \text{ lb})(12 \text{ ft}) + (400 \text{ lb})(6 \text{ ft}) = 0$ 

$$F_{EG} = +3600 \text{ lb}$$
  $F_{EG} = 3600 \text{ lb}$   $T \blacktriangleleft$ 



A Polynesian, or duopitch, roof truss is loaded as shown. Determine the force in members *HI*, *GI*, and *GJ*.

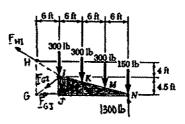
### **SOLUTION**

See solution of Problem 6.59 for reactions:

 $A = 1500 \text{ lb}^{\dagger}$ ,  $N = 1300 \text{ lb}^{\dagger} \triangleleft$ 

We pass a section through HI, GI, and GJ, and use the free body shown.

(We apply  $F_{HI}$  at H.)



+) 
$$\Sigma M_G = 0$$
:  $\left(\frac{6}{\sqrt{6^2 + 4^2}} F_{HI}\right) (8.5 \text{ ft}) + (1300 \text{ lb})(24 \text{ ft}) - (300 \text{ lb})(6 \text{ ft})$   
-  $(300 \text{ lb})(12 \text{ ft}) - (300 \text{ lb})(18 \text{ ft}) - (150 \text{ lb})(24 \text{ ft}) = 0$ 

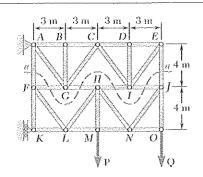
$$F_{HI} = -2375.4 \text{ lb}$$
  $F_{HI} = 2375 \text{ lb}$   $C \blacktriangleleft$ 

+) 
$$\Sigma M_I = 0$$
: (1300 lb)(18 ft) – (300 lb)(6 ft) – (300 lb)(12 ft)  
– (150 lb)(18 ft) –  $F_{GJ}$ (4.5 ft) = 0

$$F_{GJ} = +3400 \text{ lb}$$
  $F_{GJ} = 3400 \text{ lb}$   $T$ 

$$\pm \Sigma F_x = 0$$
:  $-\frac{4}{5}F_{GI} - \frac{6}{\sqrt{6^2 + 4^2}}(-2375.4 \text{ lb}) - 3400 \text{ lb} = 0$ 

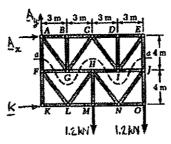
$$F_{GI} = -1779.4 \text{ lb}$$
  $F_{GI} = 1779 \text{ lb}$   $C$ 



Determine the force in members AF and EJ of the truss shown when P = Q = 1.2 kN. (*Hint*: Use section aa.)

## **SOLUTION**

Free body: Entire truss

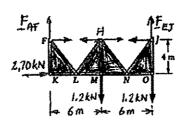


$$+)\Sigma M_A = 0$$
:  $K(8 \text{ m}) - (1.2 \text{ kN})(6 \text{ m}) - (1.2 \text{ kN})(12 \text{ m}) = 0$ 

K = +2.70 kN

 $K = 2.70 \text{ kN} \longrightarrow \triangleleft$ 

Free body: Lower portion



+)
$$\Sigma M_F = 0$$
:  $F_{EJ}(12 \text{ m}) + (2.70 \text{ kN})(4 \text{ m}) - (1.2 \text{ kN})(6 \text{ m}) - (1.2 \text{ kN})(12 \text{ m}) = 0$ 

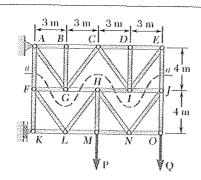
 $F_{EJ} = +0.900 \text{ kN}$ 

 $F_{EI} = 0.900 \text{ kN}$  T

 $+\frac{1}{2}\Sigma F_y = 0$ :  $F_{AF} + 0.9 \text{ kN} - 1.2 \text{ kN} - 1.2 \text{ kN} = 0$ 

 $F_{AF} = +1.500 \text{ kN}$ 

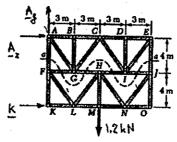
 $F_{AF} = 1.500 \, \text{kN} \, T \, \blacktriangleleft$ 



Determine the force in members AF and EJ of the truss shown when P = 1.2 kN and Q = 0. (*Hint:* Use section aa.)

### SOLUTION

Free body: Entire truss

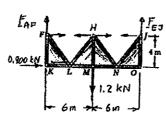


+)
$$\Sigma M_A = 0$$
:  $K(8 \text{ m}) - (1.2 \text{ kN})(6 \text{ m}) = 0$ 

K = +0.900 kN

 $\mathbf{K} = 0.900 \,\mathrm{kN} \longrightarrow \triangleleft$ 

Free body: Lower portion



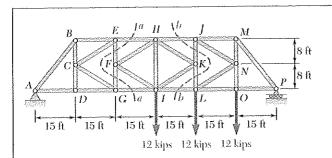
+)
$$\Sigma M_F = 0$$
:  $F_{EJ}(12 \text{ m}) + (0.900 \text{ kN})(4 \text{ m}) - (1.2 \text{ kN})(6 \text{ m}) = 0$ 

$$F_{EJ} = +0.300 \text{ kN}$$

 $F_{EJ} = 0.300 \, \text{kN}$  T

$$F_{AF} = +0.900 \text{ kN}$$

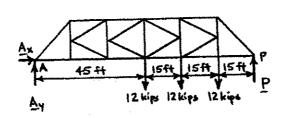
 $F_{AF} = 1.900 \,\text{kN}$  T



Determine the force in members *EH* and *GI* of the truss shown. (*Hint*: Use section *aa*.)

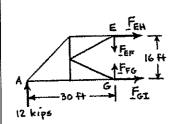
# SOLUTION

Reactions:



$$\Sigma F_x = 0$$
:  $A_x = 0$ 

+)
$$\Sigma M_P = 0$$
:  $12(45) + 12(30) + 12(15) - A_V(90) = 0$ 



$$\mathbf{A}_{y} = 12 \text{ kips}$$

+ 
$$\Sigma F_y = 0$$
:  $12 - 12 - 12 - 12 + P = 0$   $\mathbf{P} = 24 \text{ kips}$ 

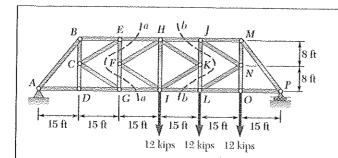
+)
$$\Sigma M_G = 0$$
:  $-(12 \text{ kips})(30 \text{ ft}) - F_{EH}(16 \text{ ft}) = 0$ 

$$F_{EH} = -22.5 \text{ kips}$$

$$F_{EH} = 22.5 \text{ kips} \quad C \blacktriangleleft$$

$$\pm \Sigma F_x = 0$$
:  $F_{GI} - 22.5 \text{ kips} = 0$ 

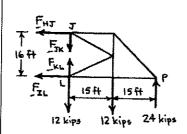
$$F_{GI} = 22.5 \text{ kips}$$
  $T \blacktriangleleft$ 



Determine the force in members HJ and IL of the truss shown. (Hint: Use section bb.)

# **SOLUTION**

See the solution to Problem 6.63 for free body diagram and analysis to determine the reactions at supports A and P.



$$A_x = 0$$
;  $A_y = 12.00 \text{ kips } \dagger$ ;  $P = 24.0 \text{ kips } \dagger$ 

+)
$$\Sigma M_L = 0$$
:  $F_{HJ}(16 \text{ ft}) - (12 \text{ kips})(15 \text{ ft}) + (24 \text{ kips})(30 \text{ ft}) = 0$ 

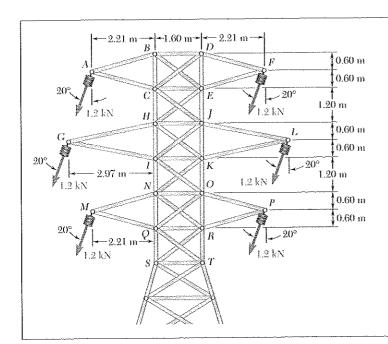
$$F_{HJ} = -33.75 \text{ kips}$$

$$F_{HJ} = 33.8 \text{ kips}$$
  $C \blacktriangleleft$ 

$$\pm \Sigma F_x = 0$$
: 33.75 kips  $-F_{IL} = 0$ 

$$F_{lL} = +33.75 \text{ kips}$$

$$F_{IL} = 33.8 \text{ kips}$$
  $T \blacktriangleleft$ 



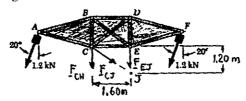
The diagonal members in the center panels of the power transmission line tower shown are very slender and can act only in tension; such members are known as counters. For the given loading, determine (a) which of the two counters listed below is acting, (b) the force in that counter.

Counters CJ and HE.

#### SOLUTION

Free body: Portion ABDFEC of tower

We assume that counter CJ is acting and show the forces exerted by that counter and by members CH and EJ.

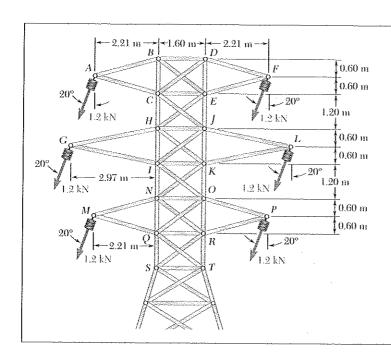


$$\pm \Sigma F_x = 0$$
:  $\frac{4}{5}F_{CJ} - 2(1.2 \text{ kN})\sin 20^\circ = 0$   $F_{CJ} = +1.026 \text{ kN}$ 

Since CJ is found to be in tension, our assumption was correct. Thus, the answers are

(a) CJ

(b) 1.026 kN T

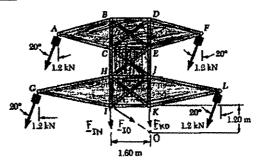


The diagonal members in the center panels of the power transmission line tower shown are very slender and can act only in tension; such members are known as counters. For the given loading, determine (a) which of the two counters listed below is acting, (b) the force in that counter.

Counters IO and KN.

### **SOLUTION**

Free body: Portion of tower shown



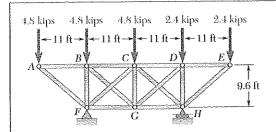
We assume that counter IO is acting and show the forces exerted by that counter and by members IN and KO.

$$\pm \Sigma F_x = 0$$
:  $\frac{4}{5}F_{IO} - 4(1.2 \text{ kN})\sin 20^\circ = 0$   $F_{IO} = +2.05 \text{ kN}$ 

Since IO is found to be in tension, our assumption was correct. Thus, the answers are

(a) IO

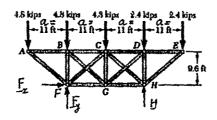
(b) 2.05 kN T ◀



The diagonal members in the center panels of the truss shown are very slender and can act only in tension; such members are known as *counters*. Determine the forces in the counters that are acting under the given loading.

#### SOLUTION

Free body: Truss



$$\Sigma F_x = 0$$
:  $F_x = 0$ 

+)
$$\Sigma M_H = 0$$
:  $4.8(3a) + 4.8(2a) + 4.8a - 2.4a - F_y(2a) = 0$ 

$$F_y = +13.20 \text{ kips}$$

 $\mathbf{F} = 13.20 \text{ kips}^{\dagger} \triangleleft$ 

$$+\frac{1}{2}\Sigma F_{y} = 0$$
:  $H + 13.20 \text{ kips} - 3(4.8 \text{ kips}) - 2(2.4 \text{ kips}) = 0$ 

$$H = +6.00 \text{ kips}$$

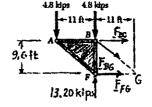
 $\mathbf{H} = 6.00 \text{ kips} \uparrow \triangleleft$ 

### Free body: ABF

We assume that counter BG is acting.

$$+ \sum F_y = 0$$
:  $-\frac{9.6}{14.6} F_{BG} + 13.20 - 2(4.8) = 0$ 

$$F_{BG} = +5.475$$



 $F_{BG} = 5.48 \text{ kips}$  T

Since BG is in tension, our assumption was correct

Free body: DEH

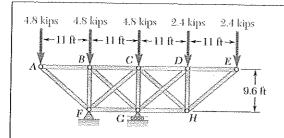
We assume that counter DG is acting.

$$+\Sigma F_y = 0$$
:  $-\frac{9.6}{14.6}F_{DG} + 6.00 - 2(2.4) = 0$ 

$$F_{DG} = +1.825$$

 $F_{DG} = 1.825 \text{ kips}$   $T \blacktriangleleft$ 

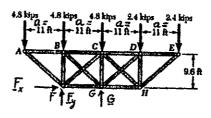
Since DG is in tension, O.K.



The diagonal members in the center panels of the truss shown are very slender and can act only in tension; such members are known as *counters*. Determine the forces in the counters that are acting under the given loading.

#### SOLUTION

Free body: Truss



$$\Sigma F_x = 0$$
:  $F_x = 0$ 

+) 
$$\Sigma M_G = 0$$
:  $-F_y a + 4.8(2a) + 4.8a - 2.4a - 2.4(2a) = 0$ 

$$F_y = 7.20$$

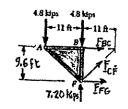
F = 7.20 kips

Free body: ABF

We assume that counter CF is acting.

$$+ \sum F_y = 0$$
:  $\frac{9.6}{14.6} F_{CF} + 7.20 - 2(4.8) = 0$ 

$$F_{CF} = +3.65$$



 $F_{CF} = 3.65 \text{ kips}$   $T \blacktriangleleft$ 

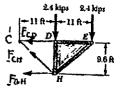
Since CF is in tension, O.K.

Free body: DEH

We assume that counter CH is acting.

$$+\Sigma F_y = 0$$
:  $\frac{9.6}{14.6}F_{CH} - 2(2.4 \text{ kips}) = 0$ 

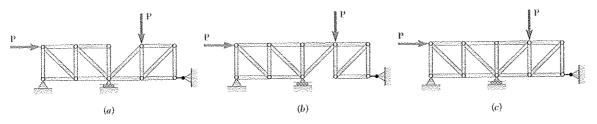
$$F_{CH} = +7.30$$



$$F_{CH} = 7.30 \text{ kips}$$
  $T \blacktriangleleft$ 

Since CH is in tension, O.K.

Classify each of the structures shown as completely, partially, or improperly constrained; if completely constrained, further classify as determinate or indeterminate. (All members can act both in tension and in compression.)



## SOLUTION

#### Structure (a)

Number of members:

$$m = 16$$

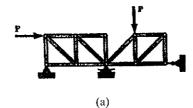
Number of joints:

$$n = 10$$

Reaction components:

$$r = 4$$

$$m+r=20, 2n=20$$



\_

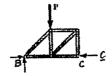
Thus:

 $m+r=2n \triangleleft$ 

To determine whether the structure is actually completely constrained and determinate, we must try to find the reactions at the supports. We divide the structure into two simple trusses and draw the free-body diagram of each truss.



This is a properly supported simple truss – O.K.



This is an improperly supported simple truss. (Reaction at C passes through B. Thus, Eq.  $\Sigma M_B = 0$  cannot be satisfied.)

Structure is improperly constrained ◀

#### Structure (b)

$$m = 16$$

$$n = 10$$

$$r = 4$$

$$m+r=20$$
,  $2n=20$ 

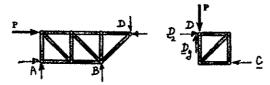
(b)

 $m+r=2n \triangleleft$ 

Thus:

# PROBLEM 6.69 (Continued)

We must again try to find the reactions at the supports dividing the structure as shown.



Both portions are simply supported simple trusses.

Structure is completely constrained and determinate ◀

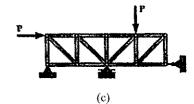
# Structure (c)

$$m = 17$$

$$n = 10$$

$$r = 4$$

$$m + r = 21, \quad 2n = 20$$



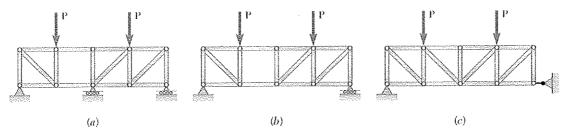
Thus:

m+r>2n

This is a simple truss with an extra support which causes reactions (and forces in members) to be indeterminate.

Structure is completely constrained and indeterminate

Classify each of the structures shown as completely, partially, or improperly constrained; if completely constrained, further classify as determinate or indeterminate. (All members can act both in tension and in compression.)



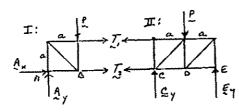
# **SOLUTION**

## Structure (a):

Non-simple truss with r = 4, m = 16, n = 10

so m+r=20=2n, but must examine further.

#### **FBD Sections:**



$$\Sigma M_A = 0 \implies T$$

$$\Sigma F_{\rm r} = 0 \implies T_2$$

$$\Sigma F_x = 0 \implies A_x$$

$$\Sigma F_y = 0 \implies A_y$$

$$\Sigma M_E = 0 \implies C_v$$

$$\Sigma F_{\nu} = 0 \implies E_{\nu}$$

Since each section is a simple truss with reactions determined,

structure is completely constrained and determinate.

Non-simple truss with r = 3, m = 16, n = 10

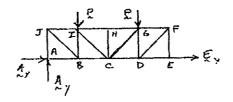
#### Structure (b):

$$m+r=19 < 2n=20$$

structure is partially constrained ◀

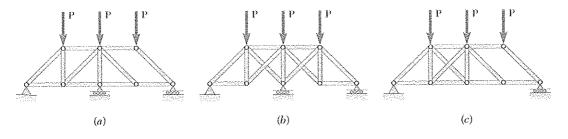
# PROBLEM 6.70 (Continued)

Structure (c):



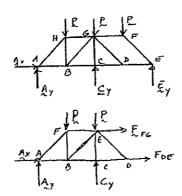
Simple truss with r = 3, m = 17, n = 10 m + r = 20 = 2n, but the horizontal reaction forces  $A_x$  and  $E_x$  are collinear and no equilibrium equation will resolve them, so the structure is improperly constrained and indeterminate

Classify each of the structures shown as completely, partially, or improperly constrained; if completely constrained, further classify as determinate or indeterminate. (All members can act both in tension and in compression.)



# **SOLUTION**

# Structure (a):



Non-simple truss with r = 4, m = 12, n = 8 so r + m = 16 = 2n, check for determinacy:

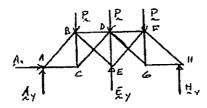
One can solve joint F for forces in EF, FG and then solve joint E for  $\mathbf{E}_y$  and force in DE.

This leaves a simple truss ABCDGH with

$$r = 3, m = 9, n = 6$$
 so  $r + m = 12 = 2n$ 

Structure is completely constrained and determinate

#### Structure (b):



Simple truss (start with ABC and add joints alphabetically to complete truss) with r = 4, m = 13, n = 8

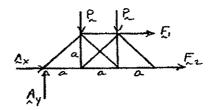
en

$$r + m = 17 > 2n = 16$$

Constrained but indeterminate ◀

# PROBLEM 6.71 (Continued)

## Structure (c):



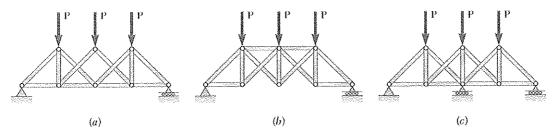
Non-simple truss with r = 3, m = 13, n = 8 so r + m = 16 = 2n. To further examine, follow procedure in Part (a) above to get truss at left.

Since  $\mathbf{F}_1 \neq 0$  (from solution of joint F),

 $\Sigma M_A = aF_1$   $\neq 0$  and there is no equilibrium.

Structure is improperly constrained ◀

Classify each of the structures shown as completely, partially, or improperly constrained; if completely constrained, further classify as determinate or indeterminate. (All members can act both in tension and in compression.)



# **SOLUTION**

## Structure (a)

Number of members:

$$m = 12$$

Number of joints:

$$n = 8$$

Reaction components:

$$r = 3$$

$$m+r=15$$
,  $2n=16$ 

Thus:

$$m+r \le 2n$$

Structure is partially constrained ◀

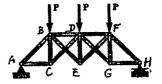
Structure (b)

$$m = 13, n = 8$$

$$r=3$$

$$m+r=16$$
,  $2n=16$ 

. .



 $\triangleleft$ 

Thus:

$$m+r=2n$$

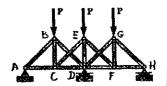
To verify that the structure is actually completely constrained and determinate, we observe that it is a simple truss (follow lettering to check this) and that it is simply supported by a pin-and-bracket and a roller. Thus:

Structure is completely constrained and determinate

### PROBLEM 6.72 (Continued)

### Structure (c)

$$m=13, n=8$$
  
 $r=4$   
 $m+r=17, 2n=16$ 



 $\triangleleft$ 

Thus:

$$m+r > 2n$$

Structure is completely constrained and indeterminate ◀

This result can be verified by observing that the structure is a simple truss (follow lettering to check thus), therefore rigid, and that its supports involve 4 unknowns.

### **PROBLEM 6.74 (Continued)**

### Structure (c):

No. of members

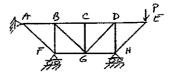
m = 13

No. of joints

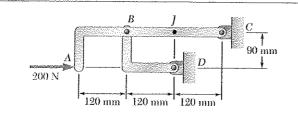
n = 8 m + r = 17 > 2n = 16

No. of react. comps.

$$r = 4$$
 unks  $>$  eqns



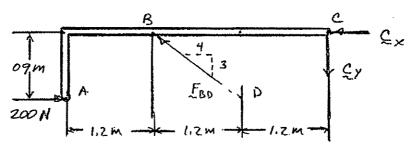
completely constrained but indeterminate ◀



For the frame and loading shown, determine the force acting on member ABC(a) at B, (b) at C.

### **SOLUTION**

### FBD ABC:



Note: BD is two-force member

(a) 
$$\left(\Sigma M_C = 0: (0.09 \text{ m})(200 \text{ N}) - (2.4 \text{ m})\left(\frac{3}{5}F_{BD}\right) = 0$$

 $\mathbf{F}_{BD} = 125.0 \text{ N} \ge 36.9^{\circ} \blacktriangleleft$ 

(b) 
$$\Sigma F_x = 0$$
: 200 N  $-\frac{4}{5}$ (125 N)  $-C_x = 0$   $C_x = 100$  N  $-\frac{3}{5}F_{BD} - C_y = 0$   $C_y = \frac{3}{5}$ (125 N)  $= 75$  N

 $C = 125.0 \text{ N} \nearrow 36.9^{\circ} \blacktriangleleft$ 

# 135 mm 240 mm - 240 m

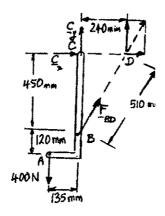
### PROBLEM 6.76

Determine the force in member BD and the components of the reaction at C.

### **SOLUTION**

We note that BD is a two-force member. The force it exerts on ABC, therefore, is directed along the BD.

Free body: ABC



Attaching  $\mathbf{F}_{BD}$  at D and resolving it into components, we write

+)
$$\Sigma M_C = 0$$
:  $(400 \text{ N})(135 \text{ mm}) + \left(\frac{450}{510}F_{BD}\right)(240 \text{ mm}) = 0$ 

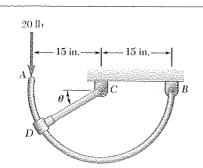
$$F_{BD} = -255 \text{ N}$$
  $F_{BD} = 255 \text{ N}$   $C \blacktriangleleft$ 

$$\pm \Sigma F_x = 0$$
:  $C_x + \frac{240}{510} (-255 \text{ N}) = 0$ 

$$C_x = +120.0 \text{ N}$$
  $C_x = 120.0 \text{ N} \longrightarrow \blacktriangleleft$ 

$$+ \sum F_y = 0$$
:  $C_y - 400 \text{ N} + \frac{450}{510} (-255 \text{ N}) = 0$ 

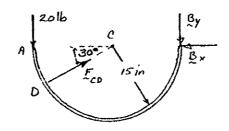
$$C_v = +625 \text{ N}$$
  $C_v = 625 \text{ N}$ 



Rod CD is fitted with a collar at D that can be moved along rod AB, which is bent in the shape of an arc of circle. For the position when  $\theta = 30^{\circ}$ , determine (a) the force in rod CD, (b) the reaction at B.

### **SOLUTION**

FBD:



(a) 
$$(\Sigma M_C = 0: (15 \text{ in.})(20 \text{ lb} - B_y) = 0$$

$$B_y = 20 \text{ lb}$$

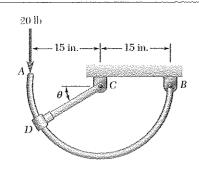
$$\Sigma F_y = 0$$
:  $-20 \text{ lb} + F_{CD} \sin 30^\circ - 20 \text{ lb} = 0$ 

$$F_{CD} = 80.0 \text{ lb}$$
 T

(b) 
$$\longrightarrow \Sigma F_x = 0$$
: (80 lb)  $\cos 30^\circ - B_y = 0$ 

$$B_r = 69.282 \text{ lb} -$$

so **B** =  $72.1 \text{ lb} \nearrow 16.10^{\circ} \blacktriangleleft$ 

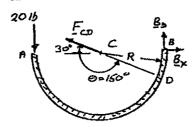


Solve Problem 6.77 when  $\theta = 150^{\circ}$ .

**PROBLEM 6.77** Rod CD is fitted with a collar at D that can be moved along rod AB, which is bent in the shape of an arc of circle. For the position when  $\theta = 30^{\circ}$ , determine (a) the force in rod CD, (b) the reaction at B.

### **SOLUTION**

Note that CD is a two-force member,  $\mathbf{F}_{CD}$  must be directed along DC.



(a) 
$$+\sum M_B = 0$$
:  $(20 \text{ lb})(2R) - (F_{CD} \sin 30^\circ)R = 0$ 

$$F_{CD} = 80 \text{ lb}$$

$$F_{CD} = 80.0 \, \text{lb}$$
  $T$ 

(b) 
$$+\sum M_C = 0$$
:  $(20 \text{ lb})R + (B_v)R = 0$ 

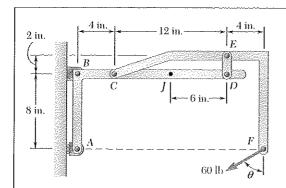
$$B_{v} = -20 \text{ lb}$$

$$\mathbf{B}_{y} = 20.0 \, \text{lb}$$

$$\pm \Sigma F_x = 0$$
:  $-F_{CD} \cos 30^\circ + B_x = 0$   
-(20 lb)  $\cos 30^\circ + B_x = 0$ 

$$B_x = 69.28 \text{ lb}$$
  $\mathbf{B}_x = 69.28 \text{ lb}$ 

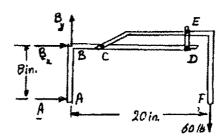
$$B = 72.1 \text{ lb} \le 16.10^{\circ} \blacktriangleleft$$



Determine the components of all forces acting on member ABCD when  $\theta = 0$ .

### **SOLUTION**

Free body: Entire assembly



+) 
$$\Sigma M_B = 0$$
:  $A(8 \text{ in.}) - (60 \text{ lb})(20 \text{ in.}) = 0$ 

$$A = 150 \text{ lb}$$

$$A = 150.0 \text{ lb} \rightarrow \blacktriangleleft$$

$$\pm \Sigma F_x = 0$$
:  $B_x + 150.0 \text{ lb} = 0$   $B_x = -150 \text{ lb}$ 

$$B_x = 150.0 \text{ lb} - \blacksquare$$

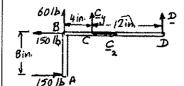
$$+ \sum F_{\nu} = 0$$
:  $B_{\nu} - 60.0 \text{ lb} = 0$   $B_{\nu} = +60.0 \text{ lb}$ 

$${\bf B}_{\nu} = 60.0 \, {\rm lb} \, {\bf \blacksquare}$$

Free body: Member *ABCD* 

We note that **D** is directed along DE, since DE is a two-force member.

+) 
$$\Sigma M_C = 0$$
:  $D(12) - (60 \text{ lb})(4) + (150 \text{ lb})(8) = 0$ 

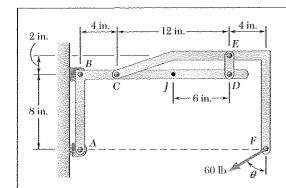


$$+\Sigma F_x = 0$$
:  $C_x + 150.0 - 150.0 = 0$   $C_x = 0$ 

$$C = 20.0 \text{ lb}$$

**D** = 80.0 lb↓ ◀

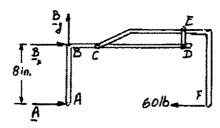
D = -80 lb



Determine the components of all forces acting on member ABCD when  $\theta = 90^{\circ}$ .

### **SOLUTION**

Free body: Entire assembly



$$+)\Sigma M_B = 0$$
:  $A(8 \text{ in.}) - (60 \text{ lb})(8 \text{ in.}) = 0$ 

$$A = +60.0 \text{ lb}$$

$$A = 60.0 \text{ lb} \longrightarrow \blacktriangleleft$$

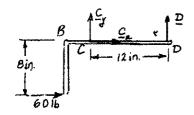
$$\pm \Sigma F_x = 0$$
:  $B_x + 60 \text{ lb} - 60 \text{ lb} = 0$   $B_x = 0$ 

$$+ \sum F_v = 0: \quad B_v = 0$$

 $\mathbf{B} = 0$ 

We note that  $\mathbf{D}$  is directed along DE, since DE is a two-force member. Free body: Member ABCD

+)
$$\Sigma M_C = 0$$
:  $D(12 \text{ in.}) + (60 \text{ lb})(8 \text{ in.}) = 0$ 



$$C_x = -60 \text{ lb}$$

$$C_r = 60.0 \text{ lb} \blacktriangleleft$$

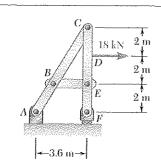
**D** = 40.0 lb↓ ◀

D = -40.0 lb

$$+ \sum F_y = 0$$
:  $C_y - 40 \text{ lb} = 0$ 

$$C_v = +40 \text{ lb}$$

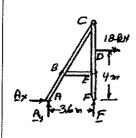
$$C_y = 40.0 \text{ lb} \uparrow \blacktriangleleft$$



For the frame and loading shown, determine the components of all forces acting on member ABC.

### **SOLUTION**

Free body: Entire frame



$$\pm \Sigma F_x = 0$$
:  $A_x + 18 \text{ kN} = 0$ 

$$A_{\rm r} = -18 \, \rm kN$$

$$A_{y} = 18.00 \text{ kN} - 4$$

+)
$$\Sigma M_E = 0$$
:  $-(18 \text{ kN})(4 \text{ m}) - A_p(3.6 \text{ m}) = 0$ 

$$A_{v} = -20 \text{ kN}$$

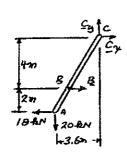
$$A_y = -20 \text{ kN}$$
  $A_y = 20.0 \text{ kN}$ 

$$+\uparrow \Sigma F_y = 0$$
:  $-20 \text{ kN} + F = 0$ 

$$F = +20 \text{ kN}$$
  $\mathbf{F} = 20 \text{ kN}$ 

Free body: Member ABC

Note: BE is a two-force member. Thus **B** is directed along line BE.



+)
$$\Sigma M_C = 0$$
:  $B(4 \text{ m}) - (18 \text{ kN})(6 \text{ m}) + (20 \text{ kN})(3.6 \text{ m}) = 0$ 

$$B = 9 \text{ kN}$$

 $\mathbf{B} = 9.00 \text{ kN} \longrightarrow \blacktriangleleft$ 

$$\pm \Sigma F_x = 0$$
:  $C_x - 18 \text{ kN} + 9 \text{ kN} = 0$ 

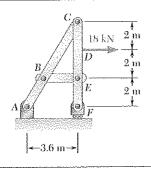
$$C_{\rm r} = 9 \, \rm kN$$

$$C_y = 9.00 \text{ kN} \longrightarrow \blacktriangleleft$$

$$+ \sum F_y = 0$$
:  $C_y - 20 \text{ kN} = 0$ 

$$C_v = 20 \text{ kN}$$

$$C_y = 20.0 \text{ kN}$$

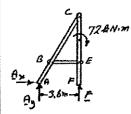


Solve Problem 6.81 assuming that the 18-kN load is replaced by a clockwise couple of magnitude 72 kN · m applied to member CDEF at Point D.

PROBLEM 6.81 For the frame and loading shown, determine the components of all forces acting on member ABC.

### SOLUTION

Free body: Entire frame



$$\pm \Sigma F_x = 0: \quad A_y = 0$$

+)
$$\Sigma M_F = 0$$
:  $-72 \text{ kN} \cdot \text{m} - A_y(3.6 \text{ m}) = 0$ 

$$A_y = -20 \text{ kN}$$
  $A_y = 20 \text{ kN}$   $A = 20.0 \text{ kN}$ 

Free body: Member ABC

*Note:* BE is a two-force member. Thus **B** is directed along line BE.

+)
$$\Sigma M_C = 0$$
:  $B(4 \text{ m}) + (20 \text{ kN})(3.6 \text{ m}) = 0$ 

B = -18 kN

 $\mathbf{B} = 18.00 \text{ kN} \blacktriangleleft$ 

$$+ F_x = 0$$
:  $-18 \text{ kN} + C_x = 0$ 

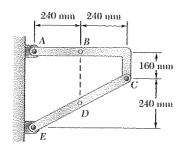
$$C_{\rm x} = 18 \, {\rm kN}$$

 $C_r = 18.00 \text{ kN} \longrightarrow \blacktriangleleft$ 

$$+ \sum F_{\nu} = 0$$
:  $C_{\nu} - 20 \text{ kN} = 0$ 

$$C_{v} = 20 \text{ kN}$$

 $C_v = 20.0 \text{ kN}$ 

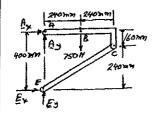


Determine the components of the reactions at A and E if a 750-N force directed vertically downward is applied (a) at B, (b) at D.

### SOLUTION

### Free-body: Entire Frame

The following analysis is valid for both (a) and (b) since position of load on its line of action is immaterial.



$$A_{x} = -450 \text{ N} \quad A_{x} = 450 \text{ N} + \Sigma F_{x} = 0: \quad E_{x} - 450 \text{ N} = 0$$

$$E_{x} = 450 \text{ N} \quad E_{x} = 450 \text{ N} + \Sigma F_{y} = 0: \quad E_{x} - 450 \text{ N} = 0$$

$$E_{x} = 450 \text{ N} \quad E_{x} = 450 \text{ N} = 0$$

$$(1)$$

### (a) Load applied at B.

Free body: Member CE

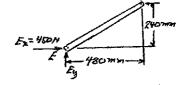
CE is a two-force member. Thus, the reaction at E must be directed along CE.

$$\frac{E_y}{450 \text{ N}} = \frac{240 \text{ mm}}{480 \text{ mm}}; \quad \mathbf{E}_y = 225 \text{ N}$$

From Eq. (1):

$$A_v + 225 - 750 = 0$$
;  $A_v = 525 \text{ lb}^{\dagger}$ 

Thus, reactions are:



$$A_x = 450 \text{ N} - A_y = 525 \text{ lb}$$

$$\mathbf{E}_{x} = 450 \,\mathrm{N} \longrightarrow$$
,  $\mathbf{E}_{y} = 225 \,\mathrm{lb} \,\dagger$ 

### (b) Load applied at D.

Free body: Member AC

AC is a two-force member. Thus, the reaction at A must be directed along AC.

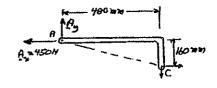
$$\frac{A_y}{450 \text{ N}} = \frac{160 \text{ mm}}{480 \text{ mm}}$$
  $A_y = 150.0 \text{ N}$ 

From Eq. (1):

$$A_y + E_y - 750 \text{ N} = 0$$
  
150 N +  $E_y - 750 \text{ N} = 0$ 

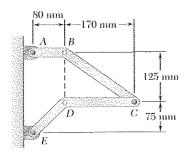
$$E_{\nu} = 600 \text{ N}$$
  $E_{\nu} = 600 \text{ N}$ 

Thus, reactions are:



$$A_x = 450 \text{ N} - A_y = 150.0 \text{ N}$$

$$\mathbf{E}_x = 450 \,\mathrm{N} \longrightarrow$$
,  $\mathbf{E}_y = 600 \,\mathrm{N} \,$ 

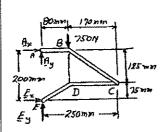


Determine the components of the reactions at A and E if a 750-N force directed vertically downward is applied (a) at B, (b) at D.

### SOLUTION

### Free body: Entire frame

The following analysis is valid for both (a) and (b) since position of load on its line of action is immaterial.



+)
$$\Sigma M_E = 0$$
:  $-(750 \text{ N})(80 \text{ mm}) - A_x(200 \text{ mm}) = 0$ 

$$A_x = -300 \text{ N}$$
  $A_x = 300 \text{ N}$ 

$$\pm \Sigma F_x = 0$$
:  $E_x - 300 \text{ N} = 0$   $E_x = 300 \text{ N}$   $E_x = 300 \text{ N}$ 

+ 
$$\Sigma F_y = 0$$
:  $A_y + E_y - 750 \text{ N} = 0$  (1)

### (a) Load applied at B.

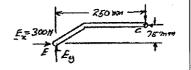
### Free body: Member CE

CE is a two-force member. Thus, the reaction at E must be directed along CE.

$$\frac{E_y}{300 \text{ N}} = \frac{75 \text{ mm}}{250 \text{ mm}}$$
  $E_y = 90 \text{ N}$ 

From Eq. (1): 
$$A_y + 90 \text{ N} - 750 \text{ N} = 0$$
  $A_y = 660 \text{ N}$ 

$$A = 660 \,\text{N}^{\dagger}$$



Thus reactions are:

$$A_x = 300 \text{ N} \longrightarrow$$
,  $A_y = 660 \text{ N}$ 

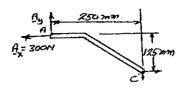
$$\mathbf{E}_x = 300 \text{ N} \longrightarrow, \qquad \mathbf{E}_y = 90.0 \text{ N} \uparrow \blacktriangleleft$$

### Load applied at D. (b)

### Free body: Member AC

AC is a two-force member. Thus, the reaction at A must be directed along AC.

$$\frac{A_y}{300 \text{ N}} = \frac{125 \text{ mm}}{250 \text{ mm}}$$
  $A_y = 150 \text{ N}$ 



### **PROBLEM 6.84 (Continued)**

From Eq. (1): 
$$A_y + E_y - 750 \text{ N} = 0$$
  
 $150 \text{ N} + E_y - 750 \text{ N} = 0$ 

$$E_y = 600 \text{ N}$$
  $E_y = 600 \text{ N}$ 

Thus, reactions are:

$$\mathbf{A}_x = 300 \text{ N} \leftarrow, \quad \mathbf{A}_y = 150.0 \text{ N}$$

$$\mathbf{E}_x = 300 \,\mathrm{N} \longrightarrow$$
,  $\mathbf{E}_y = 600 \,\mathrm{N} \,$ 

### 240 mm 240 mm A B 160 mm 240 mm

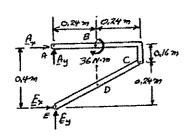
### **PROBLEM 6.85**

Determine the components of the reactions at A and E if the frame is loaded by a clockwise couple of magnitude  $36 \text{ N} \cdot \text{m}$  applied (a) at B, (b) at D.

### **SOLUTION**

Free body: Entire frame

The following analysis is valid for both (a) and (b) since the point of application of the couple is immaterial.



+) 
$$\Sigma M_E = 0$$
:  $-36 \text{ N} \cdot \text{m} - A_x (0.4 \text{ m}) = 0$   
 $A_x = -90 \text{ N}$   $A_x = 90.0 \text{ N} \leftarrow$   
+  $\Sigma F_x = 0$ :  $-90 + E_x = 0$   
 $E_x = 90 \text{ N}$   $E_x = 90.0 \text{ N} \rightarrow$   
+  $\Sigma F_y = 0$ :  $A_y + E_y = 0$  (1)

### (a) Couple applied at B.

Free body: Member CE

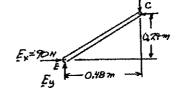
AC is a two-force member. Thus, the reaction at E must be directed along EC.

$$\frac{E_y}{90 \text{ N}} = \frac{0.24 \text{ m}}{0.48 \text{ m}}; \quad \mathbf{E}_y = 45.0 \text{ N}$$

From Eq. (1):

$$A_y + 45 \text{ N} = 0$$

$$A_v = -45 \text{ N}$$
  $A_v = 45.0 \text{ N}$ 



Thus, reactions are

$$A_x = 90.0 \text{ N} - A_y = 45.0 \text{ N}$$

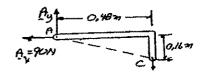
$$\mathbf{E}_x = 90.0 \,\mathrm{N} \longrightarrow$$
,  $\mathbf{E}_y = 45.0 \,\mathrm{N} \,$ 

### (b) Couple applied at D.

Free body: Member AC

AC is a two-force member. Thus, the reaction at A must be directed along AC.

$$\frac{A_y}{90 \text{ N}} = \frac{0.16 \text{ m}}{0.48 \text{ m}}; \quad \mathbf{A}_y = 30 \text{ N}$$



### PROBLEM 6.85 (Continued)

$$A_y + E_y = 0$$

$$30 \text{ N} + E_y = 0$$

$$E_y = -30 \text{ N}$$
  $E_y = 30 \text{ N}$ 

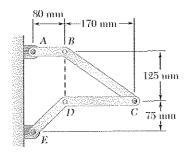
Thus, reactions are:

$$A_{*} = 90.0 \text{ N} -$$

$$\mathbf{A}_x = 90.0 \text{ N} \leftarrow$$
,  $\mathbf{A}_y = 30.0 \text{ N}$ 

$$\mathbf{E}_x = -90.0 \,\mathrm{N} \longrightarrow, \qquad \mathbf{E}_y = 30.0 \,\mathrm{N} \downarrow \blacktriangleleft$$

$$E_{\nu} = 30.0 \text{ N}$$

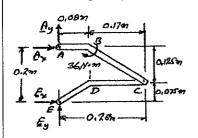


Determine the components of the reactions at A and E if the frame is loaded by a clockwise couple of magnitude 36 N · m applied (a) at B, (b) at D.

### SOLUTION

Free body: Entire frame

The following analysis is valid for both (a) and (b) since the point of application of the couple is immaterial.



+) 
$$\Sigma M_E = 0$$
:  $-36 \text{ N} \cdot \text{m} - A_x (0.2 \text{ m}) = 0$   
 $A_x = -180 \text{ N} \cdot A_x = 180 \text{ N} \leftarrow$   
+  $\Sigma F_x = 0$ :  $-180 \text{ N} + E_x = 0$   
 $E_x = 180 \text{ N} \cdot E_x = 180 \text{ N} \rightarrow$   
+  $\Sigma F_y = 0$ :  $A_y + E_y = 0$  (1)

(a) Couple applied at B.

Free body: Member CE

AC is a two-force member. Thus, the reaction at E must be directed along EC.

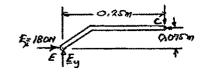
$$\frac{E_y}{180 \text{ N}} = \frac{0.075 \text{ m}}{0.25 \text{ m}}$$
  $\mathbf{E}_y = 54 \text{ N}$ 

From Eq. (1):

$$A_v + 54 \text{ N} = 0$$

$$A_y = -54 \text{ N}$$
  $A_y = 54.0 \text{ N}$ 

Thus reactions are



$$A_x = 180.0 \text{ N} \leftarrow, A_y = 54.0 \text{ N} \blacktriangleleft$$

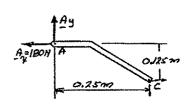
$$\mathbf{E}_{x} = 180.0 \text{ N} \longrightarrow$$
,  $\mathbf{E}_{y} = 54.0 \text{ N} \blacktriangleleft$ 

(b) Couple applied at D.

Free body: Member AC

AC is a two-force member. Thus, the reaction at A must be directed along EC.

$$\frac{A_y}{180 \text{ N}} = \frac{0.125 \text{ m}}{0.25 \text{ m}}$$
  $A_y = 90 \text{ N}$ 



### PROBLEM 6.86 (Continued)

$$A_y + E_y = 0$$

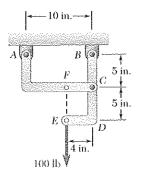
$$90 \text{ N} + E_y = 0$$

$$E_y = -90 \text{ N} \quad \mathbf{E}_y = 90 \text{ N}$$

Thus, reactions are

$$\mathbf{A}_x = 180.0 \text{ N} \longrightarrow$$
,  $\mathbf{A}_y = 90.0 \text{ N} \uparrow \blacktriangleleft$   
 $\mathbf{E}_x = -180.0 \text{ N} \longrightarrow$ ,  $\mathbf{E}_y = 90.0 \text{ N} \downarrow \blacktriangleleft$ 

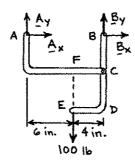
$$\mathbf{E}_{x} = -180.0 \text{ N} \longrightarrow, \quad \mathbf{E}_{y} = 90.0 \text{ N} \downarrow \blacktriangleleft$$



Determine the components of the reactions at A and B, (a) if the 500-N load is applied as shown, (b) if the 500-N load is moved along its line of action and is applied at Point F.

### SOLUTION

Free body: Entire frame

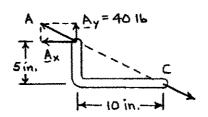


Analysis is valid for either (a) or (b), since position of 100-lb load on its line of action is immaterial.

+) 
$$\Sigma M_A = 0$$
:  $B_y(10) - (100 \text{ lb})(6) = 0$   $B_y = +60 \text{ lb}$   
+|  $\Sigma F_y = 0$ :  $A_y + 60 - 100 = 0$   $A_y = +40 \text{ lb}$   
+  $\Sigma F_x = 0$ :  $A_x + B_x = 0$  (1)

(a) Load applied at E.

Free body: Member AC



Since AC is a two-force member, the reaction at A must be directed along CA. We have

$$\frac{A_x}{10 \text{ in.}} = \frac{40 \text{ lb}}{5 \text{ in.}}$$

$$A_x = 80.0 \text{ lb} -$$
,  $A_y = 40.0 \text{ lb}$ 

From Eq. (1):  $-80 + B_x = 0$   $B_x = +80 \text{ lb}$ 

Thus,

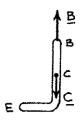
$$\mathbf{B}_x = 80.0 \text{ lb} \longrightarrow$$
,  $\mathbf{B}_y = 60.0 \text{ lb} \uparrow \blacktriangleleft$ 

### PROBLEM 6.87 (Continued)

### (b) Load applied at F.

### Free body: Member BCD

Since BCD is a two-force member (with forces applied at B and C only), the reaction at B must be directed along CB. We have therefore



$$B_x = 0$$

The reaction at B is

$$\mathbf{B}_{x} = 0$$

$$\mathbf{B}_y = 60.0 \, \mathrm{lb} \, \dagger \, \blacktriangleleft$$

$$A_x + 0 = 0 \quad A_x = 0$$

The reaction at 
$$A$$
 is

$$\mathbf{A}_x = 0$$

$$A_y = 40.0 \text{ lb}$$

### 48 lb A 5 in. C D S in. 8 in. 8 in.

### **PROBLEM 6.88**

The 48-lb load can be moved along the line of action shown and applied at A, D, or E. Determine the components of the reactions at B and F if the 48-lb load is applied (a) at A, (b) at D, (c) at E.

### **SOLUTION**

### Free body: Entire frame

The following analysis is valid for (a), (b) and (c) since the position of the load along its line of action is immaterial.

+) 
$$\Sigma M_F = 0$$
: (48 lb)(8 in.)  $-B_x$ (12 in.) = 0

$$B_x = 32 \text{ lb}$$
  $B_x = 32 \text{ lb}$ 

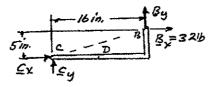
$$+ \Sigma F_x = 0$$
: 32 lb +  $F_x = 0$ 

$$F_x = -32 \text{ lb}$$
  $F_x = 32 \text{ lb}$ 

$$+\sum F_y = 0$$
:  $B_y + F_y - 48 \text{ lb} = 0$  (1)

### (a) Load applied at A.

Free body: Member CDB



CDB is a two-force member. Thus, the reaction at B must be directed along BC.

$$\frac{B_y}{32 \text{ lb}} = \frac{5 \text{ in.}}{16 \text{ in.}}$$
  $\mathbf{B}_y = 10 \text{ lb}$ 

From Eq. (1):  $10 \text{ lb} + F_y - 48 \text{ lb} = 0$ 

$$F_v = 38 \text{ lb}$$
  $\mathbf{F}_v = 38 \text{ lb}$ 

Thus reactions are:

$$B_v = 32.0 \text{ lb} \rightarrow$$
,  $B_v = 10.00 \text{ lb} \uparrow \blacktriangleleft$ 

$$F_x = 32.0 \text{ lb} - , \qquad F_y = 38.0 \text{ lb}$$

### PROBLEM 6.88 (Continued)

(b) Load applied at D.

Free body: Member ACF.

ACF is a two-force member. Thus, the reaction at F must be directed along CF.

$$\frac{F_y}{32 \text{ lb}} = \frac{7 \text{ in.}}{16 \text{ in.}}$$
  $F_y = 14 \text{ lb}$ 

From Eq. (1):  $B_v + 14 \text{ lb} - 48 \text{ lb} = 0$ 

$$B_{v} = 34 \text{ lb}$$
  $B_{v} = 34 \text{ lb}$ 

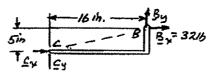
Thus, reactions are:

$$\mathbf{B}_{x} = 32.0 \text{ lb} - \mathbf{B}_{y} = 34.0 \text{ lb}$$

$$\mathbf{F}_x = 32.0 \text{ lb} \rightarrow, \quad \mathbf{F}_y = 14.00 \text{ lb}$$

(c) Load applied at E.

Free body: Member CDB



This is the same free body as in Part (a).

Reactions are same as (a)

## 48 lb A 5 in. C D 1E F -8 in. +8 in. +8

### **PROBLEM 6.89**

The 48-lb load is removed and a 288-lb  $\cdot$  in. clockwise couple is applied successively at A, D, and E. Determine the components of the reactions at B and F if the couple is applied (a) at A, (b) at D, (c) at E.

### **SOLUTION**

Free body: Entire frame

The following analysis is valid for (a), (b), and (c), since the point of application of the couple is immaterial.

$$+\Sigma M_E = 0$$
:  $-288 \text{ lb} \cdot \text{in.} - B_x(12 \text{ in.}) = 0$ 

$$B_{\rm r} = -24 \, \text{lb}$$
  $B_{\rm r} = 24 \, \text{lb}$ 

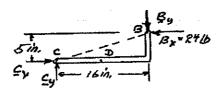
$$\pm \Sigma F_x = 0$$
:  $-24 \text{ lb} + F_x = 0$ 

$$F_x = 24 \text{ lb}$$
  $\mathbf{F}_x = 24 \text{ lb}$ 

$$+ \sum F_y = 0$$
:  $B_y + F_y = 0$  (1)

(a) Couple applied at A.

Free body: Member CDB



CDB is a two-force member. Thus, reaction at B must be directed along BC.

$$\frac{B_y}{24 \text{ lb}} = \frac{5 \text{ in.}}{16 \text{ in.}}$$
  $\mathbf{B}_y = 7.5 \text{ lb} \downarrow$ 

From Eq. (1): 
$$-7.5 \text{ lb} + F_y = 0$$

$$F_{v} = 7.5 \text{ lb} \quad \mathbf{F}_{v} = 7.5 \text{ lb}^{\dagger}$$

Thus, reactions are:

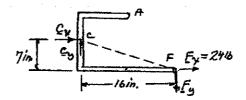
$$B_{y} = 24.0 \text{ lb} - , \quad B_{y} = 7.50 \text{ lb}$$

$$\mathbf{F}_x = 24.0 \text{ lb} \longrightarrow , \quad \mathbf{F}_y = 7.50 \text{ lb} \mid \blacktriangleleft$$

### PROBLEM 6.89 (Continued)

(b) Couple applied at D.

Free body: Member ACF.



ACF is a two-force member. Thus, the reaction at F must be directed along CF.

$$\frac{F_y}{24 \text{ lb}} = \frac{7 \text{ in.}}{16 \text{ in.}}$$
  $F_y = 10.5 \text{ lb}$ 

From Eq. (1):  $B_y - 10.5 \text{ lb}$ :

$$B_y = +10.5 \text{ lb}$$
  $B_y = 10.5 \text{ lb}$ 

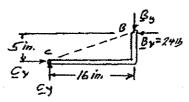
Thus, reactions are:

$$\mathbf{B}_x = 24.0 \text{ lb} - , \quad \mathbf{B}_y = 10.50 \text{ lb}$$

$$\mathbf{F}_x = 24.0 \text{ lb} \longrightarrow$$
,  $\mathbf{F}_y = 10.50 \text{ lb} \downarrow \blacktriangleleft$ 

(c) Couple applied at E.

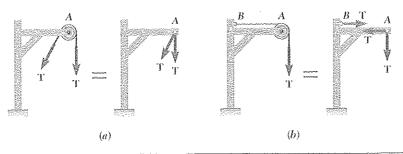
Free body: Member CDB



This is the same free body as in Part (a).

Reactions are same as in (a)

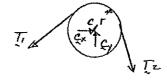
(a) Show that when a frame supports a pulley at A, an equivalent loading of the frame and of each of its component parts can be obtained by removing the pulley and applying at A two forces equal and parallel to the forces that the cable exerted on the pulley. (b) Show that if one end of the cable is attached to the frame at a point B, a force of magnitude equal to the tension in the cable should also be applied at B.



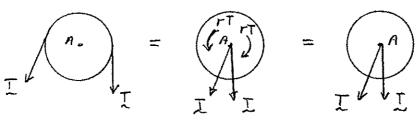
### SOLUTION

First note that, when a cable or cord passes over a *frictionless, motionless* pulley, the tension is unchanged.

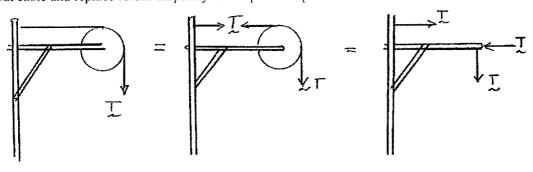
$$\sum M_C = 0$$
:  $rT_1 - rT_2 = 0$   $T_1 = T_2$ 

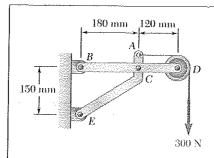


(a) Replace each force with an equivalent force-couple.



(b) Cut cable and replace forces on pulley with equivalent pair of forces at A as above.

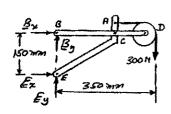




Knowing that the pulley has a radius of 50 mm, determine the components of the reactions at B and E.

### SOLUTION

Free body: Entire assembly



+)
$$\Sigma M_E = 0$$
:  $-(300 \text{ N})(350 \text{ mm}) - B_x(150 \text{ mm}) = 0$ 

$$B_{\rm v} = -700 \, \text{N}$$

$$\mathbf{B}_{x} = 700 \text{ N} \longrightarrow$$

$$\pm \Sigma F_x = 0$$
:  $-700N + E_x = 0$ 

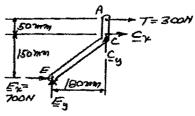
$$E_x = 700 \text{ N}$$

$$E_x = 700 \text{ N} \longrightarrow$$

$$+ \sum F_y = 0$$
:  $B_y + E_y - 300 \text{ N} = 0$ 

(1)

Free body: Member ACE



+)
$$\Sigma M_C = 0$$
: (700 N)(150 mm) – (300 N)(50 mm) –  $E_y$ (180 mm) = 0

$$E_{\nu} = 500 \text{ N}$$

$$E_{\nu} = 500 \text{ N}^{\dagger}$$

$$B_v + 500 \text{ N} - 300 \text{ N} = 0$$

$$B_{\rm v} = -200 \, \rm N$$

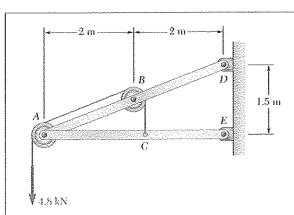
$$B_v = 200 \text{ N}$$

Thus, reactions are:

$$\mathbf{B}_x = 700 \,\mathrm{N}$$
,  $\mathbf{B}_y = 200 \,\mathrm{N}$ 

$$\mathbf{E}_{x} = 700 \text{ N} \longrightarrow, \qquad \mathbf{E}_{y} = 500 \text{ N} \uparrow \blacktriangleleft$$

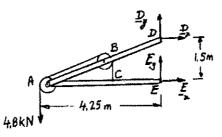
$$E_{\nu} = 500 \text{ N} \uparrow \blacktriangleleft$$



Knowing that each pulley has a radius of 250 mm, determine the components of the reactions at D and E.

### SOLUTION

Free body: Entire assembly



+)
$$\Sigma M_E = 0$$
:  $(4.8 \text{ kN})(4.25 \text{ m}) - D_x(1.5 \text{ m}) = 0$ 

$$D_x = +13.60 \text{ kN}$$

$$\mathbf{D}_{x} = 13.60 \text{ kN} \longrightarrow \blacktriangleleft$$

$$\pm \Sigma F_x = 0$$
:  $E_x + 13.60 \text{ kN} = 0$ 

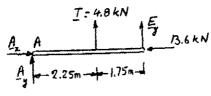
$$E_x = -13.60 \text{ kN}$$

$$\mathbf{E}_{\rm r} = 13.60 \, \text{kN} \blacktriangleleft$$

(1)

$$+\frac{1}{2}\Sigma F_y = 0$$
:  $D_y + E_y - 4.8 \text{ kN} = 0$ 

Free body: Member ACE



+)
$$\Sigma M_A = 0$$
:  $(4.8 \text{ kN})(2.25 \text{ m}) + E_y(4 \text{ m}) = 0$ 

$$E_v = -2.70 \text{ kN}$$

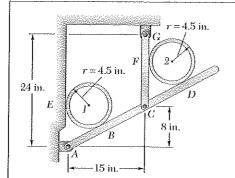
$$E_v = 2.70 \text{ kN} \downarrow \blacktriangleleft$$

From Eq. (1):

$$D_v - 2.70 \text{ kN} - 4.80 \text{ kN} = 0$$

$$D_{\nu} = +7.50 \text{ kN}$$

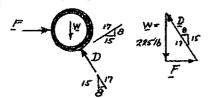
$$D_v = 7.50 \text{ kN}$$



Two 9-in.-diameter pipes (pipe 1 and pipe 2) are supported every 7.5 ft by a small frame like that shown. Knowing that the combined weight of each pipe and its contents is 30 lb/ft and assuming frictionless surfaces, determine the components of the reactions at A and G.

### SOLUTION

Free-body: Pipe 2



$$W = (30 \text{ lb/ft})(7.5 \text{ ft}) = 225 \text{ lb}$$

$$\frac{F}{8} = \frac{D}{17} = \frac{225 \text{ lb}}{15}$$

$$\mathbf{F} = 120 \text{ lb} \longrightarrow$$

$$D = 255 \text{ lb} \times$$

Geometry of pipe 2

$$r = 4.5$$
 in.

By symmetry:

$$CF = CD$$

(1)

Equate horizontal distance:

$$r + \frac{8}{17}r = CD\left(\frac{15}{17}\right)$$
$$\frac{25}{17}r = CD\left(\frac{15}{17}\right)$$
$$CD = \frac{25}{15}r = \frac{5}{3}r$$

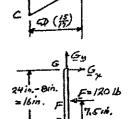
From Eq. (1):

$$CF = \frac{5}{3}r = \frac{5}{3}(4.5 \text{ in.})$$
  
 $CF = 7.5 \text{ in.}$ 



+)
$$\Sigma M_C = 0$$
: (120 lb)(7.5 in.) –  $G_x$ (16 in.) = 0

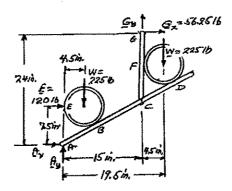
$$G_{\rm v} = 56.25 \, {\rm lb}$$



$$G_r = 56.3 \text{ lb} \longrightarrow \blacktriangleleft$$

### PROBLEM 6.93 (Continued)

Free body: Frame and pipes



Note: Pipe 2 is similar to pipe 1.

$$AE = CF = 7.5 \text{ in.}$$
  
 $E = F = 120 \text{ lb}$ 

+) 
$$\Sigma M_A = 0$$
:  $G_y(15 \text{ in.}) - (56.25 \text{ lb})(24 \text{ in.}) - (225 \text{ lb})(4.5 \text{ in.})$   
-(225 lb)(19.5 in.) - (120 lb)(7.5 in.) = 0

$$G_y = 510 \text{ lb}$$
  $G_y = 510 \text{ lb}$ 

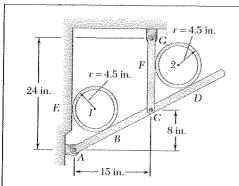
$$\pm \Sigma F_x = 0$$
:  $A_x + 120 \text{ lb} + 56.25 \text{ lb} = 0$ 

$$A_{..} = 176.25 \text{ lb}$$

$$+ \sum F_y = 0$$
:  $A_y + 510 \text{ lb} - 225 \text{ lb} - 225 \text{ lb} = 0$ 

$$A_y = -60 \text{ lt}$$

$$A_y = -60 \text{ lb}$$
  $A_y = 60.0 \text{ lb} \downarrow \blacktriangleleft$ 



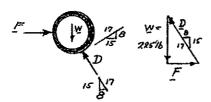
Solve Problem 6.93 assuming that pipe 1 is removed and that only pipe 2 is supported by the frames.

PROBLEM 6.93 Two 9-in.-diameter pipes (pipe 1 and pipe 2) are supported every 7.5 ft by a small frame like that shown. Knowing that the combined weight of each pipe and its contents is 30 lb/ft and assuming frictionless surfaces, determine the components of the reactions at A and G.

### **SOLUTION**

Free-body: Pipe 2

$$W = (30 \text{ lb/ft})(7.5 \text{ ft}) = 225 \text{ lb}$$



$$\frac{F}{8} = \frac{D}{17} = \frac{225 \text{ lb}}{15}$$
$$\mathbf{F} = 120 \text{ lb} \longrightarrow$$

$$\mathbf{D} = 255 \, \mathrm{lb} \, \times$$

Geometry of pipe 2

$$r = 4.5 \text{ in}.$$

By symmetry:

$$CF = CD$$

(1)

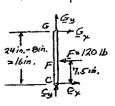
Equate horizontal distance:

$$r + \frac{8}{17}r = CD\left(\frac{15}{17}\right)$$
$$\frac{25}{17}r = CD\left(\frac{15}{17}\right)$$
$$CD = \frac{25}{15}r = \frac{5}{15}r$$

$$CD = \frac{25}{15}r = \frac{5}{3}r$$

From Eq. (1):

$$CF = \frac{5}{3}r = \frac{5}{3}(4.5 \text{ in.})$$
  
 $CF = 7.5 \text{ in.}$ 



Free body: Member CFG

+)
$$\Sigma M_C = 0$$
: (120 lb)(7.5 in.) –  $G_x$ (16 in.) = 0

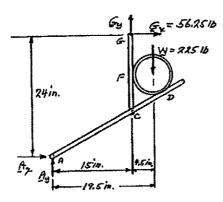
$$G_x = 56.25 \text{ lb}$$

$$G_x = 56.3 \text{ lb} \longrightarrow \blacktriangleleft$$

### PROBLEM 6.94 (Continued)

### Free body: Frame and pipe 2

 $G_x = 56.3 \text{ lb} \longrightarrow \blacktriangleleft$ 



+) 
$$\Sigma M_A = 0$$
:  $G_y(15 \text{ in.}) - (56.25 \text{ lb})(24 \text{ in.}) - (225 \text{ lb})(19.5 \text{ in.}) = 0$ 

$$G_y = 382.5 \text{ lb}$$
  $G_y = 383 \text{ lb} \uparrow \blacktriangleleft$ 

$$\pm \Sigma F_x = 0$$
:  $A_x + 56.25 \text{ lb}$ 

$$A_x = -56.25 \text{ lb}$$

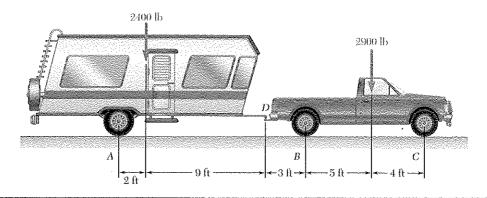
$$A_r = 56.3 \text{ lb} \leftarrow \blacktriangleleft$$

$$+ \sum F_v = 0$$
:  $A_v + 382.5 \text{ lb} - 225 \text{ lb} = 0$ 

$$A_y = -157.5 \text{ lb}$$

$$A_{\nu} = 157.5$$

A trailer weighing 2400 lb is attached to a 2900-lb pickup truck by a ball-and-socket truck hitch at D. Determine (a) the reactions at each of the six wheels when the truck and trailer are at rest, (b) the additional load on each of the truck wheels due to the trailer.



### **SOLUTION**

### (a) Free body: Trailer

(We shall denote by A, B, C the reaction at one wheel)

+)
$$\Sigma M_A = 0$$
:  $-(2400 \text{ lb})(2 \text{ ft}) + D(11 \text{ ft}) = 0$ 

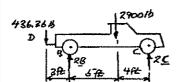


$$+ \sum F_y = 0$$
:  $2A - 2400 \text{ lb} + 436.36 \text{ lb} = 0$ 

$$A = 981.82 \text{ lb}$$

A = 982 lb ↑ ◀

Free body: Truck



+) 
$$\Sigma M_B = 0$$
:  $(436.36 \text{ lb})(3 \text{ ft}) - (2900 \text{ lb})(5 \text{ ft}) + 2C(9 \text{ ft}) = 0$ 

$$C = 732.83 \text{ lb}$$

$$+ \int \Sigma F_y = 0$$
:  $2B - 436.36 \text{ lb} - 2900 \text{ lb} + 2(732.83 \text{ lb}) = 0$ 

$$B = 935.35 \text{ lb}$$

### (b) Additional load on truck wheels

Use free body diagram of truck without 2900 lb.

+)
$$\Sigma M_B = 0$$
:  $(436.36 \text{ lb})(3 \text{ ft}) + 2C(9 \text{ ft}) = 0$ 

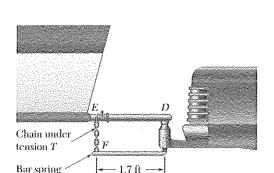
$$C = -72.73$$
 lb

 $\Delta C = -72.7 \text{ lb}$ 

$$+1 \Sigma F_v = 0$$
:  $2B - 436.36 \text{ lb} - 2(72.73 \text{ lb}) = 0$ 

$$B = 290.9 \text{ lb}$$

 $\Delta B = \pm 291 \, \text{lb}$ 



In order to obtain a better weight distribution over the four wheels of the pickup truck of Problem 6.95, a compensating hitch of the type shown is used to attach the trailer to the truck. The hitch consists of two bar springs (only one is shown in the figure) that fit into bearings inside a support rigidly attached to the truck. The springs are also connected by chains to the trailer frame, and specially designed hooks make it possible to place both chains in tension. (a) Determine the tension T required in each of the two chains if the additional load due to the trailer is to be evenly distributed over the four wheels of the truck. (b) What are the resulting reactions at each of the six wheels of the trailer-truck combination?

**PROBLEM 6.95** A trailer weighing 2400 lb is attached to a 2900-lb pickup truck by a ball-and-socket truck hitch at *D*. Determine (*a*) the reactions at each of the six wheels when the truck and trailer are at rest, (*b*) the additional load on each of the truck wheels due to the trailer.

### SOLUTION

(a) We small first find the additional reaction  $\Delta$  at each wheel due the trailer.

Free body diagram (Same  $\Delta$  at each truck wheel)

+) 
$$\Sigma M_A = 0$$
:  $-(2400 \text{ lb})(2 \text{ ft}) + 2\Delta(14 \text{ ft}) + 2\Delta(23 \text{ ft}) = 0$ 

 $\Delta = 64.86 \text{ lb}$ 

$$+\uparrow \Sigma F_{y} = 0$$
:  $2A - 2400 \text{ lb} + 4(64.86 \text{ lb}) = 0$ ;

A = 1070 lb;

24 2400 b)

A 2400 b)

B 220 944 220

A = 1070 lb

Free body: Truck

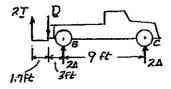
(Trailer loading only)

+) 
$$\Sigma M_D = 0$$
:  $2\Delta(12 \text{ ft}) + 2\Delta(3 \text{ ft}) - 2T(1.7 \text{ ft}) = 0$ 

 $T = 8.824\Delta$ 

= 8.824(64.86 lb)

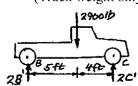
T = 572.3 lb



T = 572 lb

Free body: Truck

(Truck weight only)



+)
$$\Sigma M_B = 0$$
:  $-(2900 \text{ lb})(5 \text{ ft}) + 2C'(9 \text{ ft}) = 0$ 

C' = 805.6 lb

 $C' = 805.6 \text{ lb} \uparrow$ 

### PROBLEM 6.96 (Continued)

+ 
$$\Sigma F_y = 0$$
:  $2B' - 2900 \text{ lb} + 2(805.6 \text{ lb}) = 0$ 

$$B' = 644.4 \text{ lb}$$

B' = 644.4 lb

Actual reactions

$$B = B' + \Delta = 644.4 \text{ lb} + 64.86 = 709.2 \text{ lb}$$

**B** = 709 lb † ◀

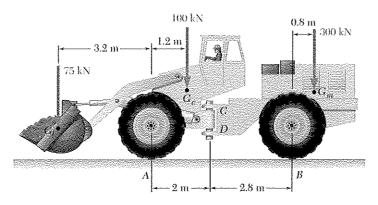
$$C = C' + \Delta = 805.6 \text{ lb} + 64.86 = 870.46 \text{ lb}$$

**C** = 870 lb ↑ ◀

(From Part a):

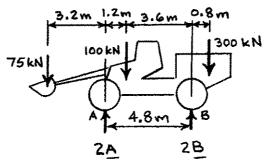
**A** = 1070 lb ↑ ◀

The cab and motor units of the front-end loader shown are connected by a vertical pin located 2 m behind the cab wheels. The distance from C to D is 1 m. The center of gravity of the 300-kN motor unit is located at  $G_m$ , while the centers of gravity of the 100-kN cab and 75-kN load are located, respectively, at  $G_c$  and  $G_l$ . Knowing that the machine is at rest with its brakes released, determine (a) the reactions at each of the four wheels, (b) the forces exerted on the motor unit at C and D.



### **SOLUTION**

### (a) Free body: Entire machine



A = Reaction at each front wheel

 $\mathbf{B} = \text{Reaction at each rear wheel}$ 

+)
$$\Sigma M_A = 0$$
: 75(3.2 m) -100(1.2 m) + 2B(4.8 m) -300(5.6 m) = 0

2B = 325 kN

 $B = 162.5 \text{ kN}^{\dagger}$ 

$$+\uparrow \Sigma F_v = 0$$
:  $2A + 325 - 75 - 100 - 300 = 0$ 

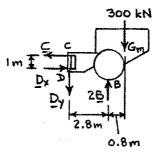
2A = 150 kN

 $A = 75.0 \text{ kN}^{\dagger}$ 

### PROBLEM 6.97 (Continued)

### (b) Free body: Motor unit

+)
$$\Sigma M_D = 0$$
:  $C(1 \text{ m}) + 2B(2.8 \text{ m}) - 300(3.6 \text{ m}) = 0$   
 $C = 1080 - 5.6B$  (1)



Recalling

$$B = 162.5 \text{ kN}, \quad C = 1080 - 5.6(162.5) = 170 \text{ kN}$$

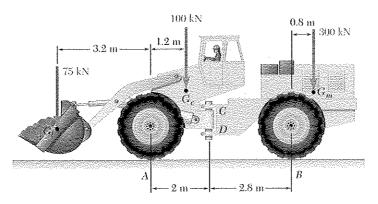
$$C = 170.0 \text{ kN} \leftarrow \blacktriangleleft$$

$$\pm \Sigma F_x = 0$$
:  $D_x - 170 = 0$   $D_x = 170.0 \text{ kN} - \blacksquare$ 

+ 
$$\Sigma F_y = 0$$
:  $2(162.5) - D_y - 300 = 0$   $\mathbf{D}_y = 25.0 \text{ kN}$ 

Solve Problem 6.97 assuming that the 75-kN load has been removed.

**PROBLEM 6.97** The cab and motor units of the front-end loader shown are connected by a vertical pin located 2 m behind the cab wheels. The distance from C to D is 1 m. The center of gravity of the 300-kN motor unit is located at  $G_m$ , while the centers of gravity of the 100-kN cab and 75-kN load are located, respectively, at  $G_c$  and  $G_t$ . Knowing that the machine is at rest with its brakes released, determine (a) the reactions at each of the four wheels, (b) the forces exerted on the motor unit at C and D.



### **SOLUTION**

### (a) Free body: Entire machine

A = Reaction at each front wheel

 $\mathbf{B} = \text{Reaction at each rear wheel}$ 

+)
$$\Sigma M_A = 0$$
:  $2B(4.8 \text{ m}) - 100(1.2 \text{ m}) - 300(5.6 \text{ m}) = 0$ 

$$2B = 375 \text{ kN}$$

B = 187.5 kN

$$+ \sum F_v = 0$$
:  $2A + 375 - 100 - 300 = 0$ 

$$2A = 25 \text{ kN}$$

 $A = 12.50 \text{ kN}^{\dagger}$ 

### (b) Free body: Motor unit

See solution of Problem 6.97 for free body diagram and derivation of Eq. (1). With B = 187.5 kN, we have

$$C = 1080 - 5.6(187.5) = 30 \text{ kN}$$

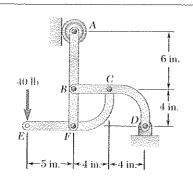
$$C = 30.0 \text{ kN} \blacktriangleleft$$

$$\pm \Sigma F_{x} = 0$$
:  $D_{x} - 30 = 0$ 

$$\mathbf{D}_{x} = 30.0 \text{ kN} \longrightarrow \blacktriangleleft$$

+ 
$$\Sigma F_y = 0$$
:  $2(187.5) - D_y - 300 = 0$ 

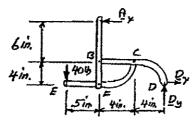
$$\mathbf{D}_{v} = 75.0 \,\mathrm{kN} \,\mathbf{\downarrow} \,\blacktriangleleft$$



For the frame and loading shown, determine the components of the forces acting on member CFE at C and F.

# **SOLUTION**

Free body: Entire frame



+)
$$\Sigma M_D = 0$$
: (40 lb)(13 in.) +  $A_x$ (10 in.) = 0

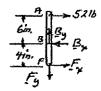
$$A_{\rm v} = -52 \, {\rm lb},$$

$$A_x = 52 \text{ lb} \longrightarrow$$

Free body: Member ABF

+)
$$\Sigma M_B = 0$$
:  $-(52 \text{ lb})(6 \text{ in.}) + F_x(4 \text{ in.}) = 0$ 

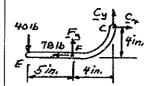
$$F_{\rm x} = \pm 78 \, {\rm lb}$$



Free body: Member CFE

From above:

$$\mathbf{F}_{x} = 78.0 \text{ lb} - \blacktriangleleft$$



$$+\sum \Sigma M_C = 0$$
: (40 lb)(9 in.) - (78 lb)(4 in.) -  $F_{\nu}$ (4 in.) = 0

$$F_{v} = +12 \text{ lb}$$

$$F_{\nu} = 12.00 \text{ lb} \, | \, \blacktriangleleft$$

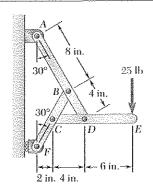
$$\pm \Sigma F_x = 0$$
:  $C_x - 78 \text{ lb} = 0$ 

$$C_{\rm v} = +78 \, {\rm lb}$$

$$C_v = 78.0 \text{ lb} \longrightarrow \blacktriangleleft$$

$$+ \int \Sigma F_y = 0$$
:  $-40 \text{ lb} + 12 \text{ lb} + C_y = 0$ ;  $C_y = +28 \text{ lb}$ 

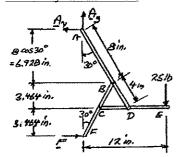
$$C_y = 28.0 \text{ lb} \uparrow \blacktriangleleft$$



For the frame and loading shown, determine the components of the forces acting on member CDE at C and D.

### **SOLUTION**

Free body: Entire frame



$$+ \sum M_y = 0$$
:  $A_y - 25 \text{ lb} = 0$ 

$$A_y = 25 \, \text{lb}$$

 $A_v = 25 \text{ lb} \uparrow$ 

+)
$$\Sigma M_F = 0$$
:  $A_x(6.928 + 2 \times 3.464) - (25 \text{ lb})(12 \text{ in.}) = 0$ 

$$A_{\rm r} = 21.651 \, \text{lb}$$

$$A_v = 21.651 \text{ lb}$$
  $A_v = 21.65 \text{ lb}$ 

$$+\Sigma F_{y} = 0$$
:  $F - 21.651 \text{ lb} = 0$ 

$$F = 21.651$$
 lb

$$\mathbf{F} = 21.65 \, \mathrm{lb} \longrightarrow$$

Free body: Member CDE

$$+)\Sigma M_C = 0$$
:  $D_v(4 \text{ in.}) - (25 \text{ lb})(10 \text{ in.}) = 0$ 

$$D_{y} = +62.5 \text{ lb}$$

$$D_y = +62.5 \text{ lb}$$
  $\mathbf{D}_y = 62.5 \text{ lb}$ 

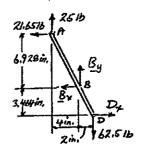
$$+ \sum F_y = 0$$
:  $-C_y + 62.5 \text{ lb} - 25 \text{ lb} = 0$ 

$$C_v = +37.5 \,\text{lb}$$

$$C_{v} = 37.5 \, lb$$

Free body: Member ABD

+) 
$$\Sigma M_B = 0$$
:  $D_x (3.464 \text{ in.}) + (21.65 \text{ lb})(6.928 \text{ in.})$   
-(25 lb)(4 in.) - (62.5 lb)(2 in.)  
 $D_x = +21.65 \text{ lb}$ 



Return to free body: Member CDE

From above

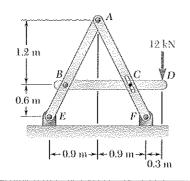
$$D_x = \pm 21.65 \text{ lb}$$

$$\pm \Sigma F_x = 0$$
:  $C_x - 21.65 \text{ lb}$ 

$$C_x = +21.65 \text{ lb}$$

$$\mathbf{D}_x = 21.7 \text{ lb} \longleftarrow \blacktriangleleft$$

 $C_x = 21.7 \text{ lb} \longrightarrow \blacktriangleleft$ 



For the frame and loading shown, determine the components of all forces acting on member ABE.

#### **SOLUTION**

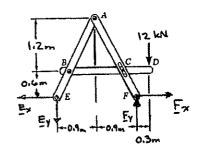
#### **FBD Frame:**

$$(\Sigma M_E = 0: (1.8 \text{ m})F_v - (2.1 \text{ m})(12 \text{ kN}) = 0$$

$$F_{\nu} = 14.00 \text{ kN}$$

$$\Sigma F_{y} = 0$$
:  $-E_{y} + 14.00 \text{ kN} - 12 \text{ kN} = 0$ 

$$E_y = 2 \text{ kN}$$



 $\mathbf{E}_{v} = 2.00 \,\mathrm{kN}$ 

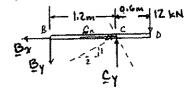
### FBD member BCD:

$$\sum M_B = 0$$
:  $(1.2 \text{ m})C_y - (12 \text{ kN})(1.8 \text{ m}) = 0$   $C_y = 18.00 \text{ kN}$ 

But C is 
$$\perp ACF$$
, so  $C_x = 2C_y$ ;  $C_x = 36.0 \text{ kN} \longrightarrow$ 

$$\rightarrow \Sigma F_x = 0$$
:  $-B_x + C_x = 0$   $B_x = C_x = 36.0 \text{ kN}$ 

$$B_x = 36.0 \text{ kN} \leftarrow \text{on } BCD$$



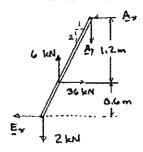
$$\Sigma F_{\nu} = 0$$
:  $-B_{\nu} + 18.00 \text{ kN} - 12 \text{ kN} = 0$   $B_{\nu} = 6.00 \text{ kN}$  on  $BCD$ 

On ABE:

$$\mathbf{B}_x = 36.0 \,\mathrm{kN} \longrightarrow \blacktriangleleft$$

$$\mathbf{B}_y = 6.00 \text{ kN} \dagger \blacktriangleleft$$

## FBD member ABE:



$$\sum M_A = 0$$
:  $(1.2 \text{ m})(36.0 \text{ kN}) - (0.6 \text{ m})(6.00 \text{ kN}) + (0.9 \text{ m})(2.00 \text{ kN}) - (1.8 \text{ m})(E_x) = 0$ 

$$(0.5 \text{ m})(2.50 \text{ kH}) = (1.5 \text{ m})(2_x) = 0$$

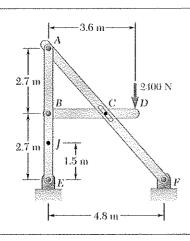
$$\rightarrow \Sigma F_x = 0$$
:  $-23.0 \text{ kN} + 36.0 \text{ kN} - A_x = 0$ 

$$A_x = 13.00 \text{ kN} \leftarrow \blacksquare$$

$$\Sigma F_v = 0$$
:  $-2.00 \text{ kN} + 6.00 \text{ kN} - A_v = 0$ 

$$A_p = 4.00 \text{ kN} \downarrow \blacktriangleleft$$

 $\mathbf{E}_{x} = 23.0 \text{ kN} \blacktriangleleft$ 

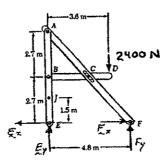


For the frame and loading shown, determine the components of all forces acting on member ABE.

**PROBLEM 6.101** For the frame and loading shown, determine the components of all forces acting on member *ABE*.

#### **SOLUTION**

**FBD Frame:** 



$$\sum M_F = 0$$
:  $(1.2 \text{ m})(2400 \text{ N}) - (4.8 \text{ m})E_v = 0$ 

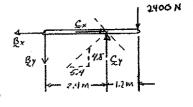
 $\mathbf{E}_{v} = 600 \,\mathrm{N}$ 

 $B_v = 1200 \text{ N}^{\dagger} \blacktriangleleft$ 

FBD member BC:

$$C_y = \frac{4.8}{5.4} C_x = \frac{8}{9} C_x$$

$$\sum M_C = 0$$
:  $(2.4 \text{ m})B_y - (1.2 \text{ m})(2400 \text{ N}) = 0$   $B_y = 1200 \text{ N}$ 



on ABE:

 $\uparrow \Sigma F_y = 0$ : -1200 N +  $C_y$  - 2400 N = 0  $C_y$  = 3600 N  $\uparrow$ 

$$C_x = \frac{9}{8}C_y \quad C_x = 4050 \text{ N} \longrightarrow$$

so

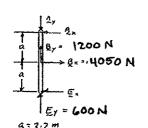
$$\rightarrow \Sigma F_x = 0$$
:  $-B_x + C_x = 0$   $B_x = 4050 \text{ N} \leftarrow \text{on } BC$ 

on ABE:

$$\mathbf{B}_{v} = 4050 \text{ N} \longrightarrow \blacktriangleleft$$

# PROBLEM 6.102 (Continued)

FBD member  $AB\theta E$ :



$$\sum M_A = 0$$
:  $a(4050 \text{ N}) - 2aE_x = 0$ 

$$E_x = 2025 \text{ N}$$

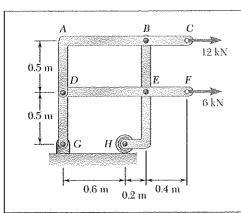
$$\mathbf{E}_x = 2025 \,\mathrm{N} - \blacktriangleleft$$

$$\rightarrow \Sigma F_x = 0$$
:  $-A_x + (4050 - 2025) \text{ N} = 0$ 

$$A_x = 2025 \text{ N} \longleftarrow \blacktriangleleft$$

$$\Sigma F_y = 0$$
: 600 N + 1200 N -  $A_y = 0$ 

$$A_y = 1800 \text{ N} \downarrow \blacktriangleleft$$



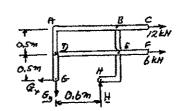
For the frame and loading shown, determine the components of the forces acting on member DABC at B and D.

### SOLUTION

Free body: Entire frame

+)
$$\Sigma M_G = 0$$
:  $H(0.6 \text{ m}) - (12 \text{ kN})(1 \text{ m}) - (6 \text{ kN})(0.5 \text{ m}) = 0$ 

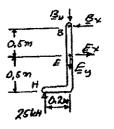
$$H = 25 \text{ kN}$$
  $H = 25 \text{ kN}$ 



Free body: Member BEH

+)
$$\Sigma M_F = 0$$
:  $B_x(0.5 \text{ m}) - (25 \text{ kN})(0.2 \text{ m}) = 0$ 

$$B_{\rm s} = +10 \, {\rm kN}$$



Free body: Member DABC

From above:

+)
$$\Sigma M_D = 0$$
:  $-B_y (0.8 \text{ m}) + (10 \text{ kN} + 12 \text{ kN})(0.5 \text{ m}) = 0$ 

$$B_{v} = +13.75 \text{ kN}$$

$$\mathbf{B}_{v} = 13.75 \,\mathrm{kN}$$

 $\mathbf{B}_x = 10.00 \text{ kN} \longrightarrow \blacktriangleleft$ 

$$+\Sigma F_{\rm x} = 0$$
:  $-D_{\rm x} + 10 \,\text{kN} + 12 \,\text{kN} = 0$ 

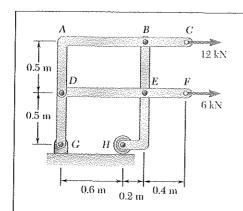
$$D_{\rm v} = +22 \, {\rm kN}$$

$$D_x = +22 \text{ kN}$$
  $D_x = 22.0 \text{ kN} - \blacksquare$ 

$$+\sum F_{\nu} = 0$$
:  $-D_{\nu} + 13.75 \text{ kN} = 0$ 

$$D_{v} = +13.75 \text{ kN}$$

$$D_y = 13.75 \text{ kN}$$



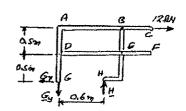
Solve Problem 6.105 assuming that the 6-kN load has been removed.

PROBLEM 6.105 For the frame and loading shown, determine the components of the forces acting on member DABC at B and D.

# **SOLUTION**

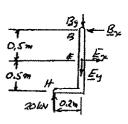
Free body: Entire frame

+)
$$\Sigma M_G = 0$$
:  $H(0.6 \text{ m}) - (12 \text{ kN})(1 \text{ m}) = 0$   
 $H = 20 \text{ kN}$   $H = 20 \text{ kN}$ 



Free body: Member BEH

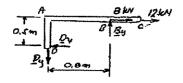
+)
$$\Sigma M_E = 0$$
:  $B_x (0.5 \text{ m}) - (20 \text{ kN})(0.2 \text{ m}) = 0$   
 $B_x = +8 \text{ kN}$ 



Free body: Member DABC

From above:

$$\mathbf{B}_x = -8.00 \text{ kN} \longrightarrow$$



+)
$$\Sigma M_D = 0$$
:  $-B_y(0.8 \text{ m}) + (8 \text{ kN} + 12 \text{ kN})(0.5 \text{ m}) = 0$ 

$$B_v = +12.5 \text{ kN}$$

$$B_v = 12.50 \text{ kN} \uparrow \blacktriangleleft$$

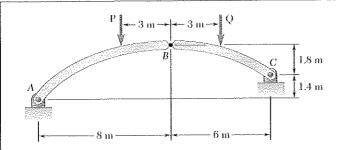
$$+\Sigma F_x = 0$$
:  $-D_x + 8 \text{ kN} + 12 \text{ kN} = 0$ 

$$D_{\rm v} = +20 \,\mathrm{kN}$$

$$D_x = +20 \text{ kN}$$
  $D_x = 20.0 \text{ kN} -$ 

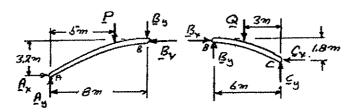
+ 
$$\Sigma F_y = 0$$
:  $-D_y + 12.5 \text{ kN} = 0$ ;  $D_y = +12.5 \text{ kN}$   $D_y = 12.50 \text{ kN}$ 

$$\mathbf{D}_y = 12.50 \,\mathrm{kN} \,\mathsf{\downarrow} \,\mathsf{\blacktriangleleft}$$



The axis of the three-hinge arch ABC is a parabola with vertex at B. Knowing that P = 112 kN and Q = 140 kN, determine (a) the components of the reaction at A, (b) the components of the force exerted at B on segment AB.

### SOLUTION



Free body: Segment AB:

+)
$$\Sigma M_A = 0$$
:  $B_x(3.2 \text{ m}) - B_y(8 \text{ m}) - P(5 \text{ m}) = 0$  (1)

0.75 (Eq. 1) 
$$B_v(2.4 \text{ m}) - B_v(6 \text{ m}) - P(3.75 \text{ m}) = 0$$
 (2)

Free body: Segment BC:

+)
$$\Sigma M_C = 0$$
:  $B_x(1.8 \text{ m}) + B_y(6 \text{ m}) - Q(3 \text{ m}) = 0$  (3)

Add (2) and (3):

$$4.2B_x - 3.75P - 3Q = 0$$

$$B_x = (3.75P + 3Q)/4.2 \tag{4}$$

$$(3.75P + 3Q)\frac{3.2}{4.2} - 8B_y - 5P = 0$$

$$B_v = (-9P + 9.6Q)/33.6 \tag{5}$$

Given that P = 112 kN and Q = 140 kN

### (a) Reaction at A:

Considering again AB as a free body

$$A_{\nu} = +122 \text{ kN}$$
  $A_{\nu} = 122.0 \text{ kN}$ 

# PROBLEM 6.107 (Continued)

(b) Force exerted at B on AB

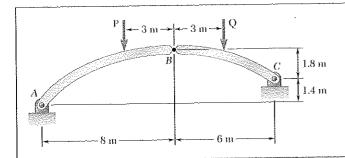
$$B_x = (3.75 \times 112 + 3 \times 140)/4.2 = 200 \text{ kN}$$

$$\mathbf{B}_x = 200 \text{ kN} \longleftarrow \blacktriangleleft$$

Eq. 
$$(5)$$
:

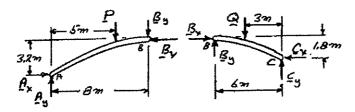
$$B_y = (-9 \times 112 + 9.6 \times 140)/33.6 = +10 \text{ kN}$$

 $\mathbf{B}_{y} = 10.00 \,\mathrm{kN}$ 



The axis of the three-hinge arch ABC is a parabola with vertex at B. Knowing that P = 140 kN and Q = 112 kN, determine (a) the components of the reaction at A, (b) the components of the force exerted at B on segment AB.

#### SOLUTION



Free body: Segment AB:

+)
$$\Sigma M_A = 0$$
:  $B_x(3.2 \text{ m}) - B_y(8 \text{ m}) - P(5 \text{ m}) = 0$  (1)

0.75 (Eq. 1) 
$$B_x(2.4 \text{ m}) - B_y(6 \text{ m}) - P(3.75 \text{ m}) = 0$$
 (2)

Free body: Segment BC:

+)
$$\Sigma M_C = 0$$
:  $B_v(1.8 \text{ m}) + B_v(6 \text{ m}) - Q(3 \text{ m}) = 0$  (3)

Add (2) and (3):

$$4.2B_x - 3.75P - 3Q = 0$$

$$B_{x} = (3.75P + 3Q)/4.2 \tag{4}$$

Eq. (1):

$$(3.75P + 3Q)\frac{3.2}{4.2} - 8B_y - 5P = 0$$

$$B_{y} = (-9P + 9.6Q)/33.6 \tag{5}$$

Given that P = 140 kN and Q = 112 kN

#### (a) Reaction at $\underline{A}$ :

$$A_y - 140 \text{ kN} - (-5.5 \text{ kN}) = 0$$

$$A_y = 134.5 \text{ kN}$$
  $A_y = 134.5 \text{ kN}$ 

# PROBLEM 6.108 (Continued)

(b) Force exerted at B on AB

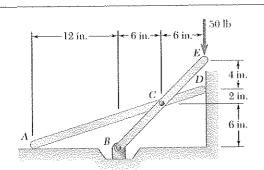
$$B_x = (3.75 \times 140 + 3 \times 112)/4.2 = 205 \text{ kN}$$

 $\mathbf{B}_{x} = 205 \,\mathrm{kN} \blacktriangleleft$ 

Eq. (5):

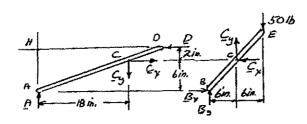
$$B_y = (-9 \times 140 + 9.6 \times 112)/33.6 = -5.5 \text{ kN}$$

 $\mathbf{B}_{v} = 5.5 \, \mathrm{kN} \, \mathbf{\blacktriangleleft}$ 



Knowing that the surfaces at A and D are frictionless, determine the forces exerted at B and C on member BCE.

### **SOLUTION**



Free body of Member ACD

$$+\sum_{y} \sum M_{H} = 0$$
:  $C_{x}(2 \text{ in.}) - C_{y}(18 \text{ in.}) = 0$   $C_{x} = 9C_{y}$  (1)

Free body of Member BCE

$$+\sum M_B = 0$$
:  $C_x(6 \text{ in.}) + C_y(6 \text{ in.}) - (50 \text{ lb})(12 \text{ in.}) = 0$ 

Substitute from (1):

$$9C_v(6) + C_v(6) - 600 = 0$$

$$C_v = +10 \text{ lb}; \quad C_x = 9C_y = 9(10) = +90 \text{ lb}$$

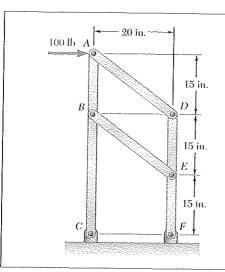
$$C = 90.6 \text{ lb} \ge 6.34^{\circ} \blacktriangleleft$$

$$\pm \sum F_x = 0$$
:  $B_x - 90 \text{ lb} = 0$   $B_x = 90 \text{ lb}$ 

$$B = 90 \, \text{lb}$$

$$+1 \Sigma F_y = 0$$
:  $B_y + 10 \text{ lb} - 50 \text{ lb} = 0$   $B_y = 40 \text{ lb}$ 

 $B = 98.5 \text{ lb} \angle 24.0^{\circ} \blacktriangleleft$ 



For the frame and loading shown, determine (a) the reaction at C, (b) the force in member AD.

#### SOLUTION

Free body: Member ABC

+) 
$$\Sigma M_C = 0$$
: + (100 lb)(45 in.) +  $\frac{4}{5}F_{AD}$ (45 in.) +  $\frac{4}{5}F_{BE}$ (30 in.) = 0  
 $3F_{AD} + 2F_{BF} = -375$  lb (1)

Free Body: Member DEF

+) 
$$\Sigma M_F = 0$$
:  $\frac{4}{5} F_{AD}(30 \text{ in.}) + \frac{4}{5} F_{BF}(15 \text{ in.}) = 0$ 

$$F_{BE} = -2F_{AD} \tag{2}$$

(a) Substitute from (2) into (1)

$$3F_{AD} + 2(-2F_{AD}) = -375 \text{ lb}$$

$$F_{AD} = +375 \text{ lb}$$
  $F_{AD} = 375 \text{ lb ten.}$ 

(2) 
$$F_{BE} = -2F_{AD} = -2(375 \text{ lb})$$

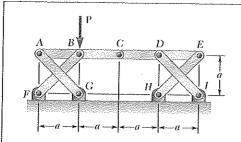
$$F_{BE} = -750 \text{ lb} \quad F_{BE} = 750 \text{ lb comp}.$$

# PROBLEM 6.110 (Continued)

(b) Return to free body of member ABC

$$\begin{array}{c} + \sum F_x = 0 \colon \quad C_x + 100 \text{ lb} + \frac{4}{5} F_{AD} + \frac{4}{5} F_{BE} = 0 \\ \\ C_x + 100 + \frac{4}{5} (375) + \frac{4}{5} (-750) = 0 \\ \\ C_x = +200 \text{ lb} \quad C_x = 200 \text{ lb} \\ \\ + \mid \sum F_y = 0 \colon \quad C_y - \frac{3}{5} F_{AD} - \frac{3}{5} F_{BF} = 0 \\ \\ C_y - \frac{3}{5} (375) - \frac{3}{5} (-750) = 0 \\ \\ C_y = -225 \text{ lb} \quad C_y = 225 \text{ lb} \downarrow \\ \\ \alpha = 48.37^\circ \\ C = 301.0 \text{ lb} \end{array}$$

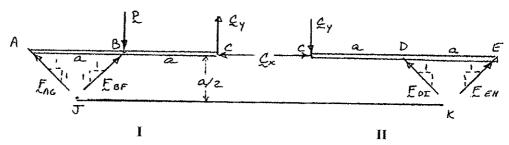
 $C = 301 \text{ lb} \le 48.4^{\circ} \blacktriangleleft$ 



Members ABC and CDE are pin-connected at C and supported by four links. For the loading shown, determine the force in each link.

### SOLUTION

**Member FBDs:** 



From FBD I:

$$\sum M_J = 0$$
:  $\frac{a}{2}C_x + \frac{3a}{2}C_y - \frac{a}{2}P = 0$   $C_x + 3C_y = P$ 

FBD II:

$$\sum M_K = 0$$
:  $\frac{a}{2}C_x - \frac{3a}{2}C_y = 0$   $C_x - 3C_y = 0$ 

Solving:

$$C_x = \frac{P}{2}$$
;  $C_y = \frac{P}{6}$  as drawn

FBD 1:

$$\sum M_B = 0: aC_y - a\frac{1}{\sqrt{2}}F_{AG} = 0 \quad F_{AG} = \sqrt{2}C_y = \frac{\sqrt{2}}{6}P \qquad F_{AG} = \frac{\sqrt{2}}{6}P \quad C \blacktriangleleft$$

$$\longrightarrow \Sigma F_x = 0: \quad -\frac{1}{\sqrt{2}} F_{AG} + \frac{1}{\sqrt{2}} F_{BF} - C_x = 0 \quad F_{BF} = F_{AG} + C_x \sqrt{2} = \frac{\sqrt{2}}{6} P + \frac{\sqrt{2}}{2} P$$

$$F_{BF} = \frac{2\sqrt{2}}{3}P \quad C \blacktriangleleft$$

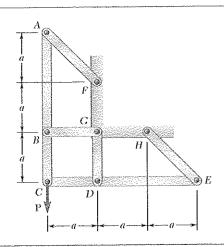
FBD II:

$$\sum M_D = 0$$
:  $a \frac{1}{\sqrt{2}} F_{EH} + a C_y = 0$   $F_{EH} = -\sqrt{2} C_y = -\frac{\sqrt{2}}{6} P$ 

$$F_{EH} = \frac{\sqrt{2}}{6}P \quad T \blacktriangleleft$$

$$\longrightarrow \Sigma F_x = 0 \colon \quad C_x - \frac{1}{\sqrt{2}} \, F_{DI} + \frac{1}{\sqrt{2}} \, F_{EH} = 0 \quad F_{DI} = F_{EH} + C_x \sqrt{2} = -\frac{\sqrt{2}}{6} \, P + \frac{\sqrt{2}}{2} \, P_{EH} = 0$$

$$F_{DI} = \frac{\sqrt{2}}{3}P \quad C \blacktriangleleft$$



Members ABC and CDE are pin-connected at C and supported by the four links AF, BG, DG, and EH. For the loading shown, determine the force in each link.

## **SOLUTION**

Free body: Member ABC

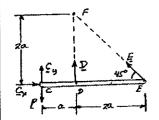
+)
$$\Sigma M_H = 0$$
:  $C_x(a) - C_y(2a) = 0$ 

$$C_x = 2C_y$$

Note: This checks that for 3-force member the forces are concurrent.

Free body: Member CDE

+)
$$\Sigma M_F = 0$$
:  $C_x(2a) - C_y(a) + P(a) = 0$ 



$$2C_x - C_y + P = 0$$

$$2(2C_y) - C_y + P = 0$$

$$C_x = 2C_v$$

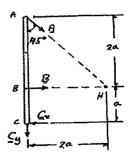
$$\pm \Sigma F = 0$$
:  $C_x - \frac{E}{\sqrt{2}} = 0$ ;  $-\frac{2}{3}P - \frac{E}{\sqrt{2}} = 0$ 

$$E = -\frac{2\sqrt{2}}{3}P$$

+) 
$$\Sigma M_E = 0$$
:  $D(2a) + C_y(3a) - P(3a) = 0$ 

$$D(2a) - \frac{P}{3}(3a) - P(3a) = 0$$

$$D = \pm 2P$$



$$C_y = -\frac{1}{3}P \triangleleft$$

$$C_x = -\frac{2}{3}P$$
 <

$$F_{EH} = \frac{2\sqrt{2}}{3}P$$
 comp.

$$F_{DG} = 2P T \blacktriangleleft$$

# PROBLEM 6.114 (Continued)

Return to free body of ABC

$$\frac{A}{\sqrt{2}} - \frac{P}{3} = 0$$

$$\frac{A}{\sqrt{2}} - \frac{P}{3} = 0$$

$$A = +\frac{\sqrt{2}}{3}P$$

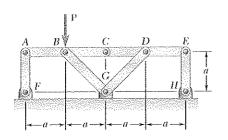
$$+) \Sigma M_A = 0: \quad B(2a) - C_x(3a) = 0$$

$$B(2a) + \frac{2}{3}P(3a) = 0$$

 $F_{AF} = \frac{\sqrt{2}}{3}P \quad T \blacktriangleleft$ 

B = -P

 $F_{BG} = P \quad C \blacktriangleleft$ 



Solve Problem 6.113 assuming that the force **P** is replaced by a clockwise couple of moment  $M_0$  applied to member CDE at D.

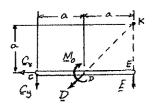
**PROBLEM 6.113** Members *ABC* and *CDE* are pin-connected at *C* and supported by four links. For the loading shown, determine the force in each link.

# **SOLUTION**

Free body: Member ABC

$$+\sum M_J = 0$$
:  $C_y(2a) + C_x(a) = 0$   
 $C_y = -2C_y$ 

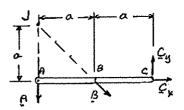
Free body: Member CDE



+)
$$\Sigma M_K = 0$$
:  $C_y(2a) - C_x(a) - M_0 = 0$ 

$$C_v(2a) - (-2C_v)(a) - M_0 = 0$$

$$C_x = -2C_v$$
:



$$C_y = \frac{M_0}{4a} \triangleleft$$

$$C_x = -\frac{M_0}{2a} \triangleleft$$

$$\frac{1}{\sqrt{2}} \Sigma F_x = 0$$
:  $\frac{D}{\sqrt{2}} + C_x = 0$ ;  $\frac{D}{\sqrt{2}} - \frac{M_0}{2a} = 0$ 

$$D = \frac{M_0}{\sqrt{2}a}$$

$$F_{DG} = \frac{M_0}{\sqrt{2}a} \quad T \quad \blacktriangleleft$$

+)
$$\Sigma M_D = 0$$
:  $E(a) - C_v(a) + M_0 = 0$ 

$$E(a) - \left(\frac{M_0}{4a}\right)(a) + M_0 = 0$$

$$E = -\frac{3}{4} \frac{M_0}{a}$$

$$F_{EH} = \frac{3}{4} \frac{M_0}{a} \quad C \blacktriangleleft$$

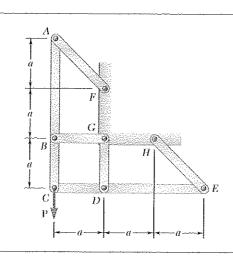
Return to Free Body of ABC

$$F_{BG} = \frac{M_0}{\sqrt{2}a} \quad T \quad \blacktriangleleft$$

+)
$$\Sigma M_B = 0$$
:  $A(a) + C_y(a)$ ;  $A(a) + \frac{M_0}{4a}(a) = 0$ 

$$A = -\frac{M_0}{4a}$$

$$F_{AF} = \frac{M_0}{4a} \quad C \blacktriangleleft$$



# **PROBLEM 6,116**

Solve Problem 6.114 assuming that the force **P** is replaced by a clockwise couple of moment  $\mathbf{M}_0$  applied to member *CDE* at *D*.

**PROBLEM 6.114** Members *ABC* and *CDE* are pin-connected at *C* and supported by the four links *AF*, *BG*, *DG*, and *EH*. For the loading shown, determine the force in each link.

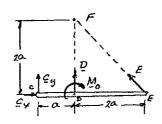
#### **SOLUTION**

Free body: Member ABC

+)
$$\Sigma M_H = 0$$
:  $C_{\nu}(a) - C_{\nu}(2a) = 0$ 

$$C_x = 2C_y$$

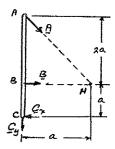
Free body: Member CDE



$$+)\Sigma M_F = 0$$
:  $C_x(2a) - C_y(a) - M_0 = 0$ 

$$(2C_y)(2a) - C_y(a) - M_0 = 0$$

$$C_x = 2C_y$$
:



$$C_y = \frac{M_0}{3a} < 1$$

$$C_x = \frac{2M_0}{3a} \triangleleft$$

$$+ \Sigma F_x = 0$$
:  $C_x - \frac{E}{\sqrt{2}} = 0$ ;  $\frac{2M_0}{3a} - \frac{E}{\sqrt{2}} = 0$ 

$$E = \frac{2\sqrt{2}}{3} \frac{M_0}{a}$$

$$+ \sum F_y = 0: D + \frac{E}{\sqrt{2}} + C_y = 0$$

$$D + \frac{2\sqrt{2}}{3} \frac{M_0}{a} \frac{1}{\sqrt{2}} + \frac{M_0}{3a} = 0$$

$$D = -\frac{M_0}{a}$$

$$F_{EH} = \frac{2\sqrt{2}}{3} \frac{M_0}{a} \blacktriangleleft$$

$$F_{DG} = \frac{M_0}{a} \quad C \blacktriangleleft$$

# PROBLEM 6.116 (Continued)

Return to free body of ABC

$$+ \sum F_y = 0: \quad \frac{A}{\sqrt{2}} + C_y = 0; \quad \frac{A}{\sqrt{2}} + \frac{M_0}{3a} = 0$$

$$A = -\frac{\sqrt{2}}{3} \frac{M_0}{a}$$

$$+ \sum M_A = 0: \quad B(2a) - C_x(3a) = 0$$

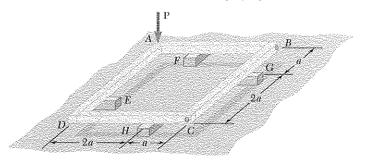
$$B(2a) - \left(\frac{2}{3} \frac{M_0}{a}\right)(3a) = 0$$

 $B = +\frac{M_0}{a}$ 

$$F_{AF} = \frac{\sqrt{2} M_0}{3 a} C \blacktriangleleft$$

$$F_{BG} = \frac{M_0}{a} T \blacktriangleleft$$

Four beams, each of length 3a, are held together by single nails at A, B, C, and D. Each beam is attached to a support located at a distance a from an end of the beam as shown. Assuming that only vertical forces are exerted at the connections, determine the vertical reactions at E, F, G, and H.

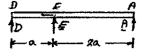


### SOLUTION

We shall draw the free body of each member. Force P will be applied to member AFB. Starting with member AED, we shall express all forces in terms of reaction E.

Member AFB:

+)
$$\Sigma M_D = 0$$
:  $A(3a) + E(a) = 0$   
 $A = -\frac{E}{3}$   
+) $\Sigma M_A = 0$ :  $-D(3a) - E(2a) = 0$   
 $D = -\frac{2E}{3}$ 



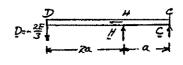
Member DHC:

$$+ \sum M_C = 0: \left(-\frac{2E}{3}\right)(3a) - H(a) = 0$$

$$H = -2E$$

$$+ \sum M_H = 0: \left(-\frac{2E}{3}\right)(2a) + C(a) = 0$$

$$C = +\frac{4E}{3}$$



(1)

 $C = +\frac{4E}{3}$ 

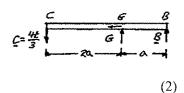
Member CGB:

$$+\sum M_B = 0: +\left(\frac{4E}{3}\right)(3a) - G(a) = 0$$

$$G = +4E$$

$$+\sum M_G = 0: +\left(\frac{4E}{3}\right)(2a) + B(a) = 0$$

$$B = -\frac{8E}{3}$$



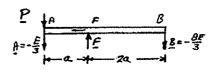
# PROBLEM 6.117 (Continued)

Member AFB:

$$+\frac{1}{2}\Sigma F_y = 0$$
:  $F - A - B - P = 0$ 

$$F - \left(-\frac{E}{3}\right) - \left(-\frac{8E}{3}\right) - P = 0$$

$$F = P - 3E$$



+) $\Sigma M_A = 0$ : F(a) - B(3a) = 0

$$(P-3E)(a) - \left(-\frac{8E}{3}\right)(3a) = 0$$

$$P - 3E + 8E = 0$$
;  $E = -\frac{P}{5}$ 

$$\mathbf{E} = \frac{P}{5} \downarrow \blacktriangleleft$$

(3)

Substitute  $E = -\frac{P}{5}$  into Eqs. (1), (2), and (3).

$$H = -2E = -2\left(-\frac{P}{5}\right) \qquad H = +\frac{2P}{5}$$

$$H = \pm \frac{2P}{5}$$

$$\mathbf{H} = \frac{2P}{5} \uparrow \blacktriangleleft$$

$$G = +4E = 4\left(-\frac{P}{5}\right) \qquad G = -\frac{4P}{5}$$

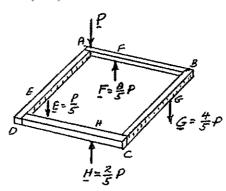
$$G = -\frac{4P}{5}$$

$$\mathbf{G} = \frac{4P}{5} \downarrow \blacktriangleleft$$

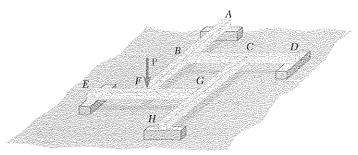
$$F = P - 3E = P - 3\left(-\frac{P}{5}\right)$$
  $F = +\frac{8P}{5}$ 

$$F = +\frac{8P}{5}$$

$$\mathbf{F} = \frac{8P}{5} \mid \blacktriangleleft$$



Four beams, each of length 2a, are nailed together at their midpoints to form the support system shown. Assuming that only vertical forces are exerted at the connections, determine the vertical reactions at A, D, E, and H.

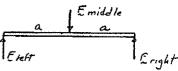


### **SOLUTION**

Note that, if we assume P is applied to EG, each individual member FBD looks like

SC

$$2F_{\text{left}} = 2F_{\text{right}} = F_{\text{middle}}$$



Labeling each interaction force with the letter corresponding to the joint of its application, we see that

$$B = 2A = 2F$$

$$C = 2B = 2D$$

$$G = 2C = 2H$$

$$P + F = 2G(= 4C = 8B = 16F) = 2E$$

From

$$P + F = 16F$$
,  $F = \frac{P}{15}$ 

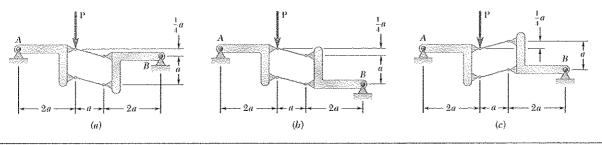
so 
$$\mathbf{A} = \frac{P}{15} \mathbf{A}$$

$$\mathbf{D} = \frac{2P}{15} \mathbf{1}$$

$$\mathbf{H} = \frac{4P}{15} \stackrel{!}{|} \blacktriangleleft$$

$$\mathbf{E} = \frac{8P}{15} \mathbf{1}$$

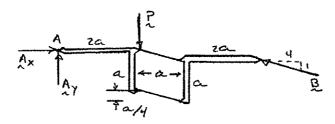
Each of the frames shown consists of two L-shaped members connected by two rigid links. For each frame, determine the reactions at the supports and indicate whether the frame is rigid.



#### **SOLUTION**

*Note:* In all three cases, the right member has only three forces acting, two of which are parallel. Thus the third force, at *B*, must be parallel to the link forces.

# (a) FBD whole:



$$\sum M_A = 0$$
:  $-2aP - \frac{a}{4} \frac{4}{\sqrt{17}} B + 5a \frac{1}{\sqrt{17}} B = 0$   $B = 2.06P$ 

$$B = 2.06P \ge 14.04^{\circ} \blacktriangleleft$$

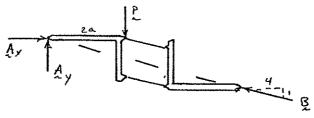
$$- \Sigma F_x = 0: \quad A_x - \frac{4}{\sqrt{17}}B = 0 \qquad \qquad \mathbf{A}_x = 2P - \mathbf{A}_x = \mathbf{A}_x =$$

$$\uparrow \Sigma F_y = 0: \quad A_y - P + \frac{1}{\sqrt{17}} B = 0 \quad \mathbf{A}_y = \frac{P}{2} \uparrow$$

$$A = 2.06P \angle 14.04^{\circ} \blacktriangleleft$$

rigid **4** 

## (b) FBD whole:



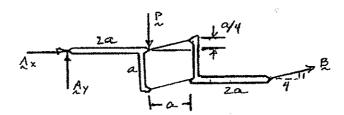
Since **B** passes through A,  $\sum M_A = 2aP = 0$  only if P = 0

no equilibrium if  $P \neq 0$ 

not rigid ◀

# PROBLEM 6.119 (Continued)

(c) FBD whole:



$$\sum M_A = 0$$
:  $5a\frac{1}{\sqrt{17}}B + \frac{3a}{4}\frac{4}{\sqrt{17}}B - 2aP = 0$   $B = \frac{\sqrt{17}}{4}P$ 

$$\mathbf{B} = 1.031P \angle 14.04^{\circ} \blacktriangleleft$$

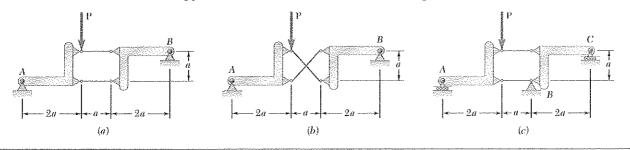
$$- \Sigma F_x = 0: \quad A_x + \frac{4}{\sqrt{17}}B = 0 \quad A_x = -P$$

$$abla \Sigma F_y = 0: A_y - P + \frac{1}{\sqrt{17}}B = 0 \quad A_y = P - \frac{P}{4} = \frac{3P}{4}$$

$$A = 1.250P \ge 36.9^{\circ}$$

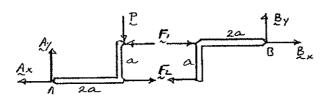
System is rigid ◀

Each of the frames shown consists of two L-shaped members connected by two rigid links. For each frame, determine the reactions at the supports and indicate whether the frame is rigid.



### **SOLUTION**

### (a) Member FBDs:



FBD I: 
$$\left(\sum M_A = 0: aF_1 - 2aP = 0 \quad F_1 = 2P; \mid \sum F_y = 0: A_y - P = 0 \quad \mathbf{A}_y = P\right)$$

FBD II: 
$$(\sum M_B = 0: -aF_2 = 0 \quad F_2 = 0)$$
  
 $\rightarrow \sum F_x = 0: \quad B_x + F_1 = 0, \quad B_x = -F_1 = -2P \quad \mathbf{B}_x = 2P \rightarrow \mathbf{E}_y = 0: \quad B_y = 0$ 

so  $\mathbf{B} = 2P$ 

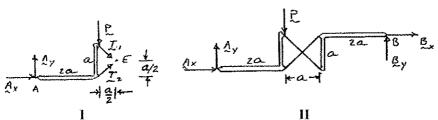
FBD I: 
$$\Sigma F_v = 0$$
:  $-A_r - F_1 + F_2 = 0$   $A_r = F_2 - F_1 = 0 - 2P$   $A_v = 2P \longrightarrow$ 

so  $A = 2.24P \angle 26.6^{\circ}$ 

frame is rigid ◀

#### (b) FBD left:

#### FBD whole:



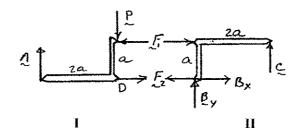
FBD I:  $\left(\sum M_E = 0: \frac{a}{2}P + \frac{a}{2}A_x - \frac{5a}{2}A_y = 0 \quad A_x - 5A_y = -P\right)$ 

FBD II:  $(\Sigma M_B = 0: 3aP + aA_x - 5aA_y = 0 A_x - 5A_y = -3P)$ 

This is impossible unless P = 0 not rigid

# PROBLEM 6.120 (Continued)

### (c) Member FBDs:



FBD I: 
$$\Sigma F_{\nu} = 0$$
:  $A - P = 0$ 

$$A = P \uparrow \blacktriangleleft$$

$$\sum M_D = 0$$
:  $aF_1 - 2aA = 0$   $F_1 = 2P$ 

$$\Sigma F_x = 0$$
:  $F_2 - F_1 = 0$   $F_2 = 2P$ 

FBD II: 
$$(\Sigma M_B = 0: 2aC - aF_1 = 0 C = \frac{F_1}{2} = P$$

$$\mathbf{C} = P \mid \blacktriangleleft$$

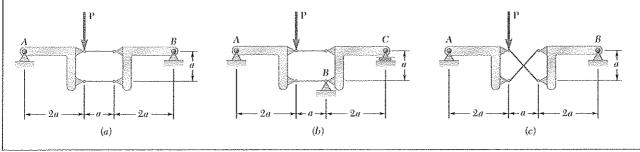
$$\longrightarrow \Sigma F_x = 0: \quad F_1 - F_2 + B_x = 0 \quad B_x = P - P = 0$$

$$\uparrow \Sigma F_x = 0: \quad B_y + C = 0 \quad B_y = -C = -P$$

 $\mathbf{B} = P$ 

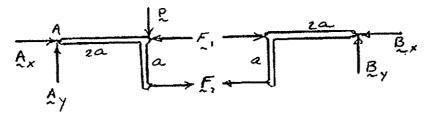
Frame is rigid ◀

Each of the frames shown consists of two L-shaped members connected by two rigid links. For each frame, determine the reactions at the supports and indicate whether the frame is rigid.



#### **SOLUTION**

# (a) Member FBDs:



I

 $\mathbf{H}$ 

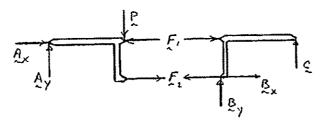
FBD II:

$$\label{eq:sigma_bound} \begin{split} & \uparrow \Sigma F_y = 0 \colon \quad B_y = 0 \qquad \Big( \ \Sigma M_B = 0 \colon \quad a F_2 = 0 \qquad F_2 = 0 \end{split}$$

FBD I:  $(\Sigma M_A = 0: aF_2 - 2aP = 0 \text{ but } F_2 = 0)$ 

so P = 0 not rigid for  $P \neq 0$ 

#### (b) Member FBDs:



Note: 7 Unknowns  $(A_x, A_y, B_x, B_y, F_1, F_2, C)$  but only 6 independent equations.

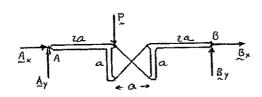
System is statically indeterminate ◀

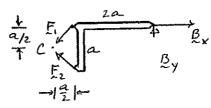
System is, however, rigid ◀

# PROBLEM 6.121 (Continued)

(c) FBD whole:







 $\mathbf{H}$ 

FBD I: 
$$\left(\sum M_A = 0: 5aB_y - 2aP = 0\right)$$

$$\mathbf{B}_y = \frac{2}{5}P^{\dagger}$$

$$abla F_y = 0: \quad A_y - P + \frac{2}{5}P = 0$$

$$\mathbf{A}_y = \frac{3}{5}P^{\dagger}$$

FBD II: 
$$(\Sigma M_c = 0: \frac{a}{2}B_x - \frac{5a}{2}B_y = 0 \quad B_x = 5B_y$$
  $\mathbf{B}_x = 2P$ 

$$\mathbf{B}_x = 2P$$

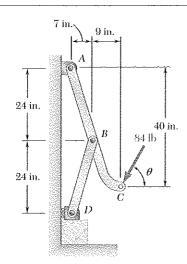
FBD I: 
$$\Sigma F_x = 0$$
:  $A_x + B_x = 0$   $A_x = -B_x$   $A_x = 2P$ 

$$\mathbf{A}_{\mathbf{r}} = 2P$$

 $A = 2.09P \ge 16.70^{\circ} \blacktriangleleft$ 

 $B = 2.04P \angle 11.31^{\circ} \blacktriangleleft$ 

System is rigid ◀



An 84-lb force is applied to the toggle vise at C. Knowing that  $\theta = 90^{\circ}$ , determine (a) the vertical force exerted on the block at D, (b) the force exerted on member ABC at B.

#### **SOLUTION**

We note that BD is a two-force member.

Free body: Member ABC

We have

$$BD = \sqrt{(7)^2 + (24)^2} = 25 \text{ in.}$$
  
 $(F_{BD})_x = \frac{7}{25} F_{BD}, \quad (F_{BD})_y = \frac{24}{25} F_{BD}$ 

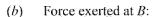
+)
$$\Sigma M_A = 0$$
:  $(F_{BD})_x(24) + (F_{BD})_y(7) - 84(16) = 0$ 

$$\left(\frac{7}{25}F_{BD}\right)(24) + \left(\frac{24}{25}F_{BD}\right)(7) = 84(16)$$

$$\frac{336}{25}F_{BD} = 1344$$

$$F_{BD} = 100 \text{ lb}$$

$$\tan \alpha = \frac{24}{7} \quad \alpha = 73.7^{\circ}$$

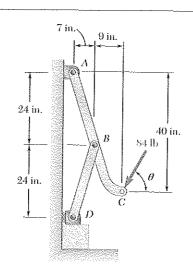


(a) Vertical force exerted on block



$$(F_{BD})_y = \frac{24}{25}F_{BD} = \frac{24}{25}(100 \text{ lb}) = 96 \text{ lb}$$

 $(\mathbf{F}_{BD})_{y} = 96.0 \text{ lb} \downarrow \blacktriangleleft$ 



Solve Problem 6.122 when  $\theta = 0$ .

**PROBLEM 6.122** An 84-lb force is applied to the toggle vise at C. Knowing that  $\theta = 90^{\circ}$ , determine (a) the vertical force exerted on the block at D, (b) the force exerted on member ABC at B.

## **SOLUTION**

We note that BD is a two-force member.

Free body: Member ABC

We have

$$BD = \sqrt{(7)^2 + (24)^2} = 25 \text{ in.}$$
  
 $(F_{BD})_x = \frac{7}{25} F_{BD}, \quad (F_{BD})_y = \frac{24}{25} F_{BD}$ 

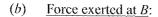
+)
$$\Sigma M_A = 0$$
:  $(F_{BD})_x(24) + (F_{BD})_y(7) - 84(40) = 0$ 

$$\left(\frac{7}{25}F_{BD}\right)(24) + \left(\frac{24}{25}F_{BD}\right)(7) = 84(40)$$

$$\frac{336}{25}F_{BD} = 3360$$

$$F_{BD} = 250 \text{ lb}$$

$$\tan \alpha = \frac{24}{7} \quad \alpha = 73.7^{\circ}$$



(a)

 $\mathbf{F}_{BD} = 250.0 \text{ lb} \ 273.7^{\circ} \ 4$ 

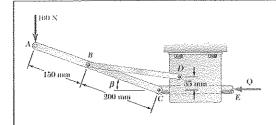
24in



Vertical force exerted on block

$$(F_{BD})_y = \frac{24}{25}F_{BD} = \frac{24}{25}(250 \text{ lb}) = 240 \text{ lb}$$

 $(\mathbf{F}_{BD})_{v} = 240 \text{ lb} \ \ \, \blacksquare$ 

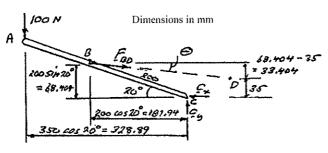


The control rod CE passes through a horizontal hole in the body of the toggle system shown. Knowing that link BD is 250 mm long, determine the force  $\mathbf{Q}$  required to hold the system in equilibrium when  $\beta = 20^{\circ}$ .

### **SOLUTION**

We note that BD is a two-force member.

Free body: Member ABC



Since

$$BD = 250$$
,  $\theta = \sin^{-1} \frac{33.404}{250}$ ;  $\theta = 7.679^{\circ}$ 

+)
$$\Sigma M_C = 0$$
:  $(F_{BD} \sin \theta)187.94 - (F_{BD} \cos \theta)68.404 + (100 \text{ N})328.89 = 0$ 

$$F_{BD}[187.94\sin 7.679^{\circ} - 68.404\cos 7.679^{\circ}] = 32889$$

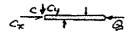
$$F_{BD} = 770.6 \text{ N}$$

$$\pm \Sigma F_x = 0$$
: (770.6 N) cos 7.679° =  $C_x = 0$ 

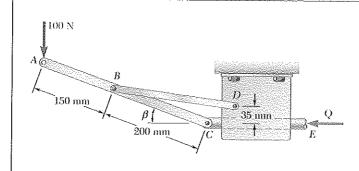
$$C_r = +763.7 \text{ N}$$

Member CO:

$$\Sigma F_{\rm v} = 0$$
:  $Q = C_{\rm x} = 763.7 \text{ N}$ 



 $Q = 764 \text{ N} \blacktriangleleft$ 



Solve Problem 6.124 when (a)  $\beta = 0$ , (b)  $\beta = 6^{\circ}$ .

**PROBLEM 6.124** The control rod *CE* passes through a horizontal hole in the body of the toggle system shown. Knowing that link *BD* is 250 mm long, determine the force **Q** required to hold the system in equilibrium when  $\beta = 20^{\circ}$ .

### **SOLUTION**

We note that BD is a two-force member.

(a) 
$$\beta = 0$$
:

Free body: Member ABC

Since

$$BD = 250 \text{ mm}, \quad \sin \theta = \frac{35 \text{ mm}}{250 \text{ mm}}; \quad \theta = 8.048^{\circ}$$

+) $\Sigma M_C = 0$ : (100 N)(350 mm) –  $F_{BD} \sin \theta$ (200 mm) = 0

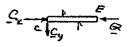
$$F_{BD} = 1250 \text{ N}$$

$$+ \Sigma F_x = 0$$
:  $F_{BD} \cos \theta - C_x = 0$ 

$$(1250 \text{ N})(\cos 8.048^\circ) - C_x = 0$$
  $C_x = 1237.7 \text{ N}$ 

Member CE:

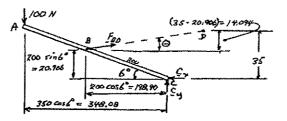
$$\pm \Sigma F_x = 0$$
: (1237.7 N)  $-Q = 0$   
 $Q = 1237.7$  N



Q = 1238 N ---

$$(b) \qquad \beta = 6^{\circ}$$

Free body: Member ABC



Dimensions in mm

Since

$$BD = 250 \text{ mm}, \quad \theta = \sin^{-1} \frac{14.094 \text{ mm}}{250 \text{ mm}}$$

$$\theta = 3.232^{\circ}$$

+)
$$\Sigma M_C = 0$$
:  $(F_{BD} \sin \theta)198.90 + (F_{BD} \cos \theta)20.906 - (100 \text{ N})348.08 = 0$ 

$$F_{BD}[198.90 \sin 3.232^{\circ} + 20.906 \cos 3.232^{\circ}] = 34808$$

$$F_{BD} = 1084.8 \text{ N}$$

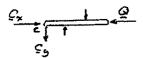
# PROBLEM 6.125 (Continued)

$$+ \sum F_x = 0: \quad F_{BD} \cos \theta - C_x = 0$$

$$(1084.8 \text{ N})\cos 3.232^{\circ} - C_x = 0$$

$$C_x = +1083.1 \text{ N}$$

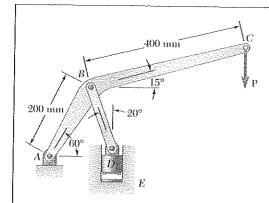
Member DE:



$$\Sigma F_x = 0$$
:  $Q = C_x$ 

$$Q = 1083.1 \text{ N}$$

 $Q = 1083 \text{ N} \blacktriangleleft$ 



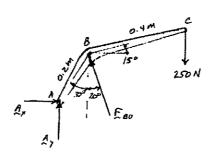
The press shown is used to emboss a small seal at E. Knowing that P = 250 N, determine (a) the vertical component of the force exerted on the seal, (b) the reaction at A.

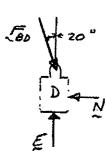
### **SOLUTION**

# FBD Stamp D:

$$\Sigma F_y = 0$$
:  $E - F_{BD} \cos 20^\circ = 0$ ,  $E = F_{BD} \cos 20^\circ$ 

FBD ABC:





$$(\Sigma M_A = 0: (0.2 \text{ m})(\sin 30^\circ)(F_{BD}\cos 20^\circ) + (0.2 \text{ m})(\cos 30^\circ)(F_{BD}\sin 20^\circ)$$
$$-[(0.2 \text{ m})\sin 30^\circ + (0.4 \text{ m})\cos 15^\circ](250 \text{ N}) = 0$$

$$F_{BD} = 793.64 \text{ N} \cdot C$$

and, from above,

$$E = (793.64 \text{ N})\cos 20^{\circ}$$

(a) 
$$\mathbf{E} = 746 \,\mathrm{N} \,\mathbf{\downarrow} \,\mathbf{\blacktriangleleft}$$

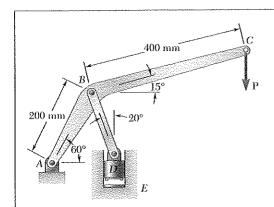
$$\Sigma F_x = 0$$
:  $A_x - (793.64 \text{ N}) \sin 20^\circ = 0$ 

$$A_x = 271.44 \text{ N} \longrightarrow$$

$$\Sigma F_y = 0$$
:  $A_y + (793.64 \text{ N})\cos 20^\circ - 250 \text{ N} = 0$ 

$$A_v = 495.78 \text{ N}$$

so (b) 
$$A = 565 \text{ N} \le 61.3^{\circ} \blacktriangleleft$$



The press shown is used to emboss a small seal at E. Knowing that the vertical component of the force exerted on the seal must be 900 N, determine (a) the required vertical force  $\mathbf{P}$ , (b) the corresponding reaction at A.

### **SOLUTION**

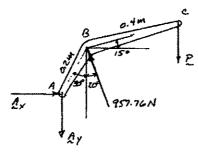
### FBD Stamp D:

$$\Sigma F_y = 0$$
: 900 N -  $F_{BD} \cos 20^\circ = 0$ ,  $F_{BD} = 957.76$  N C

20° N

(a)

FBD ABC:



 $\sum M_A = 0: [(0.2 \text{ m})(\sin 30^\circ)](957.76 \text{ N})\cos 20^\circ + [(0.2 \text{ m})(\cos 30^\circ)](957.76 \text{ N})\sin 20^\circ - [(0.2 \text{ m})\sin 30^\circ + (0.4 \text{ m})\cos 15^\circ]P = 0$ 

$$P = 301.70 \text{ N},$$

P = 302 N

(b) 
$$\Sigma F_x = 0$$
:  $A_x - (957.76 \text{ N}) \sin 20^\circ = 0$ 

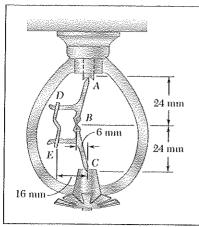
 $A_x = 327.57 \text{ N} \longrightarrow$ 

$$\uparrow \Sigma F_v = 0$$
:  $-A_v + (957.76 \text{ N})\cos 20^\circ - 301.70 \text{ N} = 0$ 

 $A_y = 598.30 \text{ N}$ 

SO

 $A = 682 \text{ N} \le 61.3^{\circ} \blacktriangleleft$ 



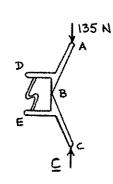
Water pressure in the supply system exerts a downward force of 135 N on the vertical plug at A. Determine the tension in the fusible link DE and the force exerted on member BCE at B.

## **SOLUTION**

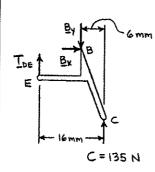
Free body: Entire linkage

$$+\Sigma F_y = 0$$
:  $C - 135 = 0$ 

$$C = +135 \text{ N}$$



Free body: Member BCE



$$+\Sigma F_x = 0$$
:  $B_x = 0$ 

+)
$$\Sigma M_B = 0$$
: (135 N)(6 mm) –  $T_{DE}$ (10 mm) = 0

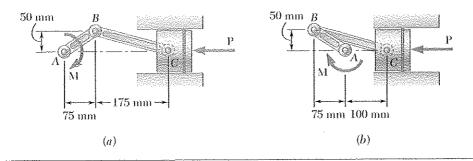
 $T_{DE} = 81.0 \text{ N}$ 

$$+ \uparrow \Sigma F_y = 0$$
:  $135 + 81 - B_y = 0$ 

$$B_v = +216 \text{ N}$$

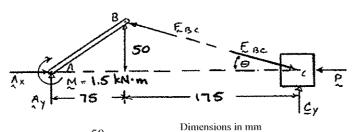
$$\mathbf{B} = 216 \,\mathrm{N} \, \blacktriangleleft$$

A couple M of magnitude 1.5 kN  $\cdot$  m is applied to the crank of the engine system shown. For each of the two positions shown, determine the force P required to hold the system in equilibrium.



### **SOLUTION**

# (a) FBDs:



Note:

$$\tan \theta = \frac{50 \text{ mm}}{175 \text{ mm}}$$

$$\sum M_A = 0$$
:  $(0.250 \text{ m})C_y - 1.5 \text{ kN} \cdot \text{m} = 0$   $C_y = 6.00 \text{ kN}$ 

FBD piston:

FBD whole:

$$\uparrow \Sigma F_y = 0: \quad C_y - F_{BC} \sin \theta = 0 \quad F_{BC} = \frac{C_y}{\sin \theta} = \frac{6.00 \text{ kN}}{\sin \theta}$$

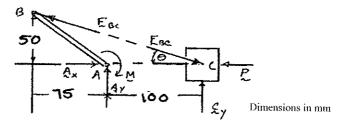
$$\rightarrow \Sigma F_x = 0$$
:  $F_{BC} \cos \theta - P = 0$ 

$$P = F_{BC} \cos \theta = \frac{6.00 \text{ kN}}{\tan \theta} = 7 \text{ kips}$$

 $P = 21.0 \text{ kN} - \blacktriangleleft$ 

# PROBLEM 6.129 (Continued)

(b) FBDs:



Note:

$$\tan \theta = \frac{2}{7}$$
 as above

FBD whole:

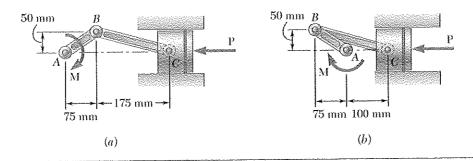
$$\sum M_A = 0$$
:  $(0.100 \text{ m})C_y - 1.5 \text{ kN} \cdot \text{m} = 0$   $C_y = 15 \text{ kN}$ 

$$\Sigma F_y = 0$$
:  $C_y - F_{BC} \sin \theta = 0$   $F_{BC} = \frac{C_y}{\sin \theta}$ 

$$\Sigma F_x = 0: \quad F_{BC} \cos \theta - P = 0$$

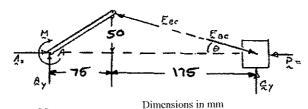
$$P = F_{BC} \cos \theta = \frac{C_y}{\tan \theta} = \frac{15 \text{ kN}}{2/7} \qquad \mathbf{P} = 52.5 \text{ kN} \longleftarrow \blacktriangleleft$$

A force P of magnitude 16 kN is applied to the piston of the engine system shown. For each of the two positions shown, determine the couple M required to hold the system in equilibrium.



# **SOLUTION**

# (a) FBDs:



Note:

$$\tan \theta = \frac{50 \text{ mm}}{175 \text{ mm}}$$
$$= \frac{2}{100}$$

FBD piston:

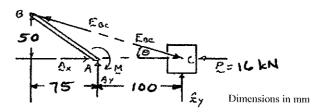
$$\rightarrow \Sigma F_x = 0$$
:  $F_{BC} \cos \theta - P = 0$   $F_{BC} = \frac{P}{\cos \theta}$ 

$$\uparrow \Sigma F_y = 0: \quad C_y - F_{BC} \sin \theta = 0 \quad C_y = F_{BC} \sin \theta = P \tan \theta = \frac{2}{7}P$$

FBD whole:  $(\Sigma M_A = 0: (0.250 \text{ m})C_y - M = 0)$ 

# PROBLEM 6.130 (Continued)

(b) **FBDs:** 



Note:

$$\tan \theta = \frac{2}{7}$$
 as above

FBD piston: as above

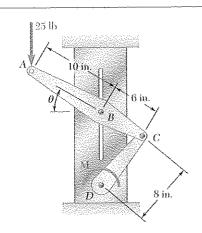
$$C_y = P \tan \theta = \frac{2}{7}P$$

FBD whole:

$$\sum M_A = 0$$
:  $(0.100 \text{ m})C_y - M = 0$   $M = (0.100 \text{ m})\frac{2}{7}(16 \text{ kN})$ 

 $M = 0.45714 \text{ kN} \cdot \text{m}$ 

 $\mathbf{M} = 457 \,\mathrm{N \cdot m}$ 



The pin at B is attached to member ABC and can slide freely along the slot cut in the fixed plate. Neglecting the effect of friction, determine the couple  $\mathbf{M}$  required to hold the system in equilibrium when  $\theta = 30^{\circ}$ .

# **SOLUTION**

Free body: Member ABC

+)
$$\Sigma M_C = 0$$
: (25 lb)(13.856 in.) –  $B(3 \text{ in.}) = 0$ 

$$B = +115.47 \text{ lb}$$

$$+ \sum F_y = 0$$
:  $-25 \text{ lb} + C_y = 0$ 

$$C_{v} = +25 \text{ lb}$$

$$\pm \Sigma F_x = 0$$
: 115.47 lb  $-C_x = 0$ 

$$C_{\rm v} = +115.47 \, \text{lb}$$

Free body: Member CD

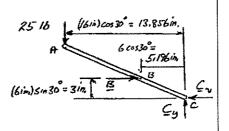
$$\beta = \sin^{-1} \frac{5.196}{8}$$
;  $\beta = 40.505^{\circ}$ 

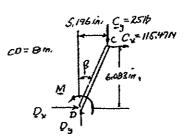
 $CD\cos\beta = (8 \text{ in.})\cos 40.505^{\circ} = 6.083 \text{ in.}$ 

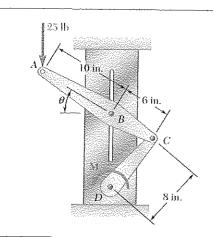
$$+$$
) $\Sigma M_D = 0$ :  $M - (25 \text{ lb})(5.196 \text{ in.}) - (115.47 \text{ lb})(6.083 \text{ in.}) = 0$ 

$$M = +832.3 \text{ lb} \cdot \text{in}$$
.

 $M = 832 \text{ lb} \cdot \text{in.} \blacktriangleleft$ 







The pin at B is attached to member ABC and can slide freely along the slot cut in the fixed plate. Neglecting the effect of friction, determine the couple M required to hold the system in equilibrium when  $\theta = 60^{\circ}$ .

### SOLUTION

Free body: Member ABC

+)
$$\Sigma M_C = 0$$
: (25 lb)(8 in.) –  $B(5.196 \text{ in.}) = 0$ 

B = +38.49 lb

$$\pm \Sigma F_x = 0$$
: 38.49 lb  $-C_x = 0$ 

$$C_{\rm v} = +38.49 \, \text{lb}$$

$$+ \sum F_y = 0$$
:  $-25 \text{ lb} + C_y = 0$ 

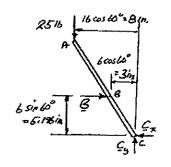
$$C_{y} = +25 \text{ lb}$$

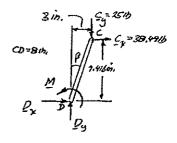
Free body: Member CD

$$\beta = \sin^{-1}\frac{3}{8}$$
;  $\beta = 22.024^{\circ}$ 

 $CD\cos\beta = (8 \text{ in.})\cos 22.024^\circ = 7.416 \text{ in.}$ 

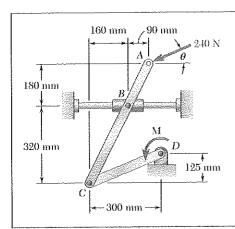
+)
$$\Sigma M_D = 0$$
:  $M - (25 \text{ lb})(3 \text{ in.}) - (38.49 \text{ lb})(7.416 \text{ in.}) = 0$ 





 $M = +360.4 \text{ lb} \cdot \text{in}$ .

 $\mathbf{M} = 360 \text{ lb} \cdot \text{in.}$ 



Arm ABC is connected by pins to a collar at B and to crank CD at C. Neglecting the effect of friction, determine the couple M required to hold the system in equilibrium when  $\theta = 0$ .

## **SOLUTION**

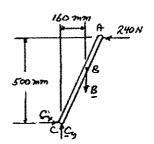
Free body: Member ABC

$$\pm \Sigma F_x = 0$$
:  $C_x - 240 \text{ N} = 0$ 

$$C_{\rm x} = +240 \, \text{N}$$

+)
$$\Sigma M_C = 0$$
: (240 N)(500 mm) –  $B$ (160 mm) = 0

$$B = +750 \text{ N}$$

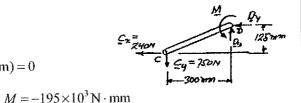


$$+ \sum F_y = 0$$
:  $C_y - 750 \text{ N} = 0$ 

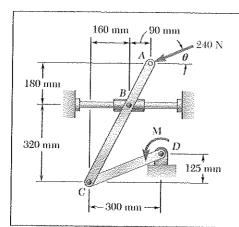
$$C_y = +750 \text{ N}$$

Free body: Member CD

+)
$$\Sigma M_D = 0$$
:  $M + (750 \text{ N})(300 \text{ mm}) - (240 \text{ N})(125 \text{ mm}) = 0$ 



 $\mathbf{M} = 195.0 \,\mathrm{kN \cdot m}$ 



Arm ABC is connected by pins to a collar at B and to crank CD at C. Neglecting the effect of friction, determine the couple M required to hold the system in equilibrium when  $\theta = 90^{\circ}$ .

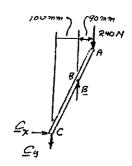
# SOLUTION

Free body: Member ABC

$$+ \Sigma F_x = 0$$
:  $C_x = 0$ 

+)
$$\Sigma M_B = 0$$
:  $C_y (160 \text{ mm}) - (240 \text{ N})(90 \text{ mm}) = 0$ 

$$C_{y} = +135 \text{ N}$$

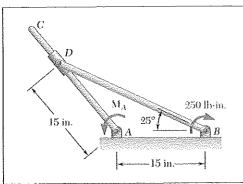


Free body: Member CD

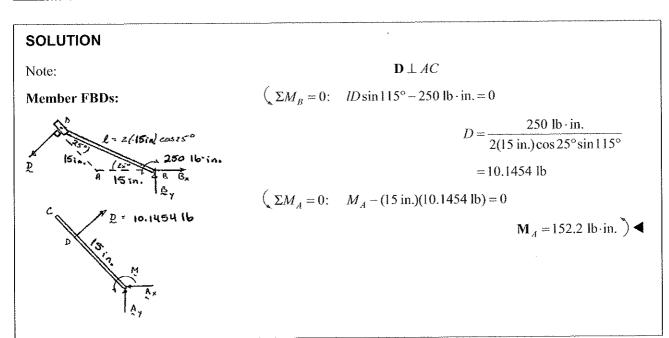
+)
$$\Sigma M_D = 0$$
:  $M - (135 \text{ N})(300 \text{ mm}) = 0$ 

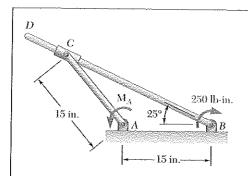
$$M = +40.5 \times 10^3 \text{ N} \cdot \text{mm}$$

$$\mathbf{M} = 40.5 \, \mathrm{kN \cdot m} \, \mathbf{M} = 40.5 \, \mathrm{kN \cdot m} \, \mathbf{M}$$



Two rods are connected by a slider block as shown. Neglecting the effect of friction, determine the couple  $\mathbf{M}_{\mathcal{A}}$  required to hold the system in equilibrium.



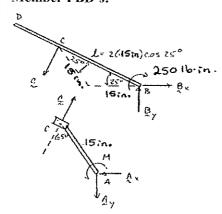


Two rods are connected by a slider block as shown. Neglecting the effect of friction, determine the couple  $\mathbf{M}_A$  required to hold the system in equilibrium.

## SOLUTION

Note:

Member FBD's:



 $\mathbf{C} \perp BD$ 

$$\sum M_B = 0$$
:  $lC - 250$  lb·in. = 0

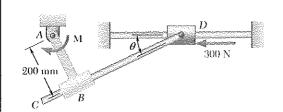
$$C = \frac{250 \text{ lb} \cdot \text{in.}}{2(15 \text{ in.})\cos 25^{\circ}}$$

$$C = 9.1948 \text{ lb}$$

$$\sum M_A = 0$$
:  $M_A - (15 \text{ in.})C \sin 65^\circ = 0$ 

$$M_A = (15 \text{ in.})(9.1948 \text{ lb}) \sin 65^\circ$$

$$\mathbf{M}_{A} = 125.0 \text{ lb · in.}$$



Rod CD is attached to the collar D and passes through a collar welded to end B of lever AB. Neglecting the effect of friction, determine the couple M required to hold the system in equilibrium when  $\theta = 30^{\circ}$ .

### **SOLUTION**

Note:

$$\mathbf{B} \perp CD$$

FBD DC:

$$/\Sigma F_{x'} = 0$$
:  $D_y \sin 30^\circ - (300 \text{ N})\cos 30^\circ = 0$ 

$$D_y = \frac{300 \text{ N}}{\tan 30^\circ} = 519.62 \text{ N}$$

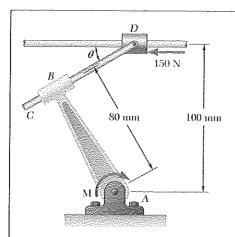
200N

FBD machine:

$$\sum M_A = 0$$
:  $\frac{0.200 \text{ m}}{\sin 30^\circ} 519.62 \text{ N} - M = 0$ 

$$M = 207.85 \text{ N} \cdot \text{m}$$

$$\mathbf{M} = 208 \,\mathrm{N \cdot m}$$



Rod CD is attached to the collar D and passes through a collar welded to end B of lever AB. Neglecting the effect of friction, determine the couple M required to hold the system in equilibrium when  $\theta = 30^{\circ}$ .

# **SOLUTION**

### FBD DC:

$$\int \Sigma F_{x'} = 0$$
:  $D_y \sin 30^\circ - (150 \text{ N}) \cos 30^\circ = 0$ 

$$D_y = (150 \text{ N}) \text{ ctn } 30^\circ = 259.81 \text{ N}$$

#### FBD machine:

$$\sum M_A = 0$$
:  $(0.100 \text{ m})(150 \text{ N}) + d(259.81 \text{ N}) - M = 0$ 

$$d = b - 0.040 \text{ m}$$

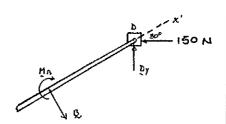
$$b = \frac{0.030718 \text{ m}}{\tan 30}$$

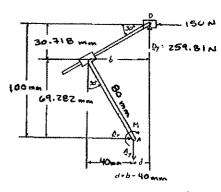
so

$$b = 0.053210 \text{ m}$$

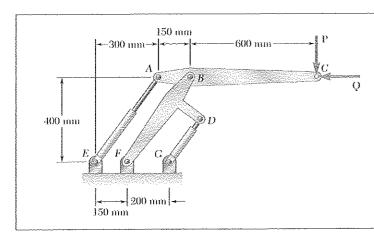
$$d = 0.0132100 \text{ m}$$

$$M = 18.4321 \text{ N} \cdot \text{m}$$





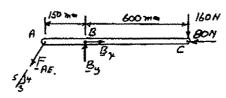
$$M = 18.43 \text{ N} \cdot \text{m}.$$



Two hydraulic cylinders control the position of the robotic arm ABC. Knowing that in the position shown the cylinders are parallel, determine the force exerted by each cylinder when P = 160 N and Q = 80 N.

## SOLUTION

Free body: Member ABC



+
$$\Sigma M_B = 0$$
:  $\frac{4}{5} F_{AE} (150 \text{ mm}) - (160 \text{ N})(600 \text{ mm}) = 0$ 

$$F_{AE} = +800 \text{ N}$$

$$F_{AE} = 800 \text{ N}$$
  $T \blacktriangleleft$ 

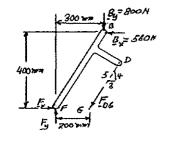
$$\pm \Sigma F_x = 0$$
:  $-\frac{3}{5}(800 \text{ N}) + B_x - 80 \text{ N} = 0$ 

$$B_{\rm x} = +560 \, \rm N$$

$$+\int \Sigma F_y = 0$$
:  $-\frac{4}{5}(800 \text{ N}) + B_y - 160 \text{ N} = 0$ 

$$B_v = +800 \text{ N}$$

Free body: Member BDF

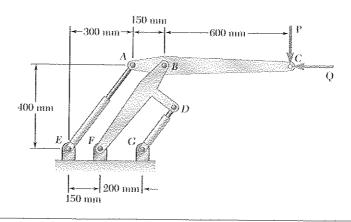


+)
$$\Sigma M_F = 0$$
: (560 N)(400 mm) - (800 N)(300 mm) -  $\frac{4}{5}F_{DG}$  (200 mm) = 0

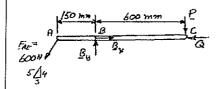
$$F_{\rm max} = -100 \text{ N}$$

$$F_{DG} = -100 \text{ N}$$
  $F_{DG} = 100.0 \text{ N}$   $C \blacktriangleleft$ 

Two hydraulic cylinders control the position of the robotic arm ABC. In the position shown, the cylinders are parallel and both are in tension. Knowing the  $F_{AE} = 600 \text{ N}$  and  $F_{DG} = 50 \text{ N}$ , determine the forces **P** and **Q** applied at C to arm ABC.



### SOLUTION



Free body: Member ABC

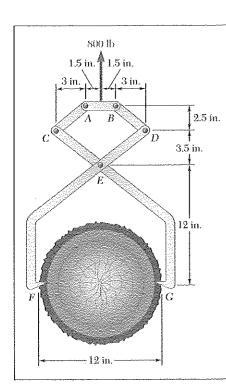
+) 
$$\Sigma M_C = 0$$
:  $\frac{4}{5} (600)(750 \text{ mm}) - B_y (600 \text{ mm}) = 0$   
 $B_y = +600 \text{ N}$ 

Free body: Member BDF

$$+ \sum M_F = 0: \quad B_x(400 \text{ mm}) - (600 \text{ N})(300 \text{ mm})$$
$$-\frac{4}{5}(50 \text{ N})(200 \text{ mm}) = 0$$
$$B_x = +470 \text{ N}$$

Return to free body: Member ABC



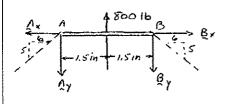


A log weighing 800 lb is lifted by a pair of tongs as shown. Determine the forces exerted at *E* and *F* on tong *DEF*.

### **SOLUTION**

FBD AB:

FBD DEF:



By symmetry:

$$A_y = B_y = 400 \text{ lb}$$

and

$$A_x = B_x$$

$$= \frac{6}{5} (400 \text{ lb})$$

$$= 480 \text{ lb}$$

Note:

 $\mathbf{D} = -\mathbf{B}$ 

so

$$D_x = 480 \text{ lb}$$

$$D_{\nu} = 400 \text{ lb}$$

 $\sum M_F = (10.5 \text{ in.})(400 \text{ lb}) + (15.5 \text{ in.})(480 \text{ lb}) - (12 \text{ in.})E_x = 0$ 

$$E_{\rm r} = 970 \; {\rm lb}$$

**E** = 970 lb →

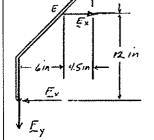
$$- \Sigma F_x = 0$$
:  $-480 \text{ lb} + 970 \text{ lb} - F_x = 0$ 

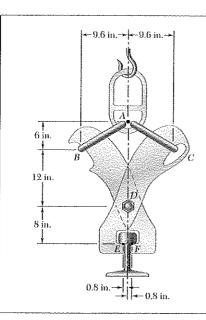
$$F_x = 490 \text{ lb}$$

$$\sum F_y = 0$$
: 400 lb -  $F_y = 0$ 

$$F_{\rm v} = 400 \; {\rm lb}$$

 $F = 633 \text{ lb } \nearrow 39.2^{\circ} \blacktriangleleft$ 

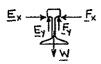




A 39-ft length of railroad rail of weight 44 lb/ft is lifted by the tongs shown. Determine the forces exerted at *D* and *F* on tong *BDF*.

### **SOLUTION**

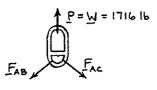
Free body: Rail



W = (39 ft)(44 lb/ft) = 1716 lb

By symmetry 
$$E_y = F_y = \frac{1}{2}W = 858 \text{ lb}$$

Free body: Upper link

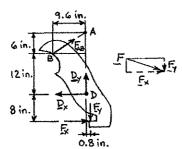


By symmetry,  $(F_{AB})_y = (F_{AC})_y = \frac{1}{2}W = 858 \text{ lb}$ 

Since AB is a two-force member,

$$\frac{(F_{AB})_x}{9.6} = \frac{(F_{AB})_y}{6}$$
  $(F_{AB})_x = \frac{9.6}{6}(858) = 1372.8 \text{ lb}$ 

Free Body: Tong BDF



+)
$$\Sigma M_D = 0$$
: (Attach  $F_{AB}$  at  $A$ )  

$$F_x(8) - (F_{AB})_x(18) - F_y(0.8) = 0$$

$$F_x(8) = (1372.8 \text{ lb})(18) - (858 \text{ lb})(0.8) = 0$$

$$F_x(8) - (1372.8 \text{ lb})(18) - (858 \text{ lb})(0.8) = 0$$

$$+ \Sigma F_x = 0$$
:  $-D_x + (F_{AB})_x + F_x = 0$ 

$$D_x = (F_{AB})_x + F_x = 1372.8 + 3174.6 = 4547.4 \text{ lb}$$

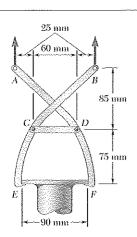
$$+ \sum F_v = 0$$
:  $D_v + (F_{AB})_v - F_v = 0$ 

 $F_{\rm r} = +3174.6 \, \text{lb}$ 

$$D_v = 0$$

$$D = 4550 \text{ lb} - \blacksquare$$

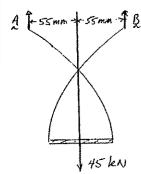
 $F = 3290 \text{ lb} \le 15.12^{\circ} \blacktriangleleft$ 



The tongs shown are used to apply a total upward force of 45 kN on a pipe cap. Determine the forces exerted at D and F on tong ADF.

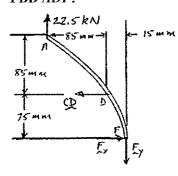
# **SOLUTION**

#### FBD whole:



By symmetry A = B = 22.5 kN

FBD ADF:



 $\sum M_F = 0$ : (75 mm)CD - (100 mm)(22.5 kN) = 0

 $CD = 30.0 \text{ kN} \longleftarrow \blacktriangleleft$ 

 $\longrightarrow \Sigma F_x = 0: \quad F_x - CD = 0$ 

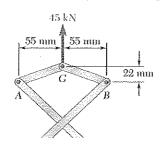
 $F_x = CD = 30 \text{ kN}$ 

 $\Delta F_y = 0: 22.5 \text{ kN} - F_y = 0$ 

 $F_y = 22.5 \text{ kN}$ 

so

 $F = 37.5 \text{ kN} \le 36.9^{\circ} \blacktriangleleft$ 



If the toggle shown is added to the tongs of Problem 6.143 and a single vertical force is applied at G, determine the forces exerted at D and F on tong ADF.

### SOLUTION

Free body: Toggle

By symmetry

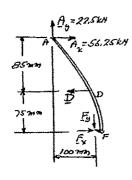
$$A_y = \frac{1}{2} (45 \text{ kN}) = 22.5 \text{ kN}$$

Pr Play By By

AG is a two-force member

$$\frac{22.5 \text{ kN}}{22 \text{ mm}} = \frac{A_x}{55 \text{ mm}}$$
$$A_x = 56.25 \text{ kN}$$

Free body: Tong ADF



$$+ \sum F_y = 0$$
: 22.5 kN  $- F_y = 0$ 

$$F_y = +22.5 \text{ kN}$$

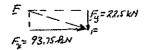
+)
$$\Sigma M_F = 0$$
:  $D(75 \text{ mm}) - (22.5 \text{ kN})(100 \text{ mm}) - (56.25 \text{ kN})(160 \text{ mm}) = 0$ 

$$D = +150 \text{ kN}$$

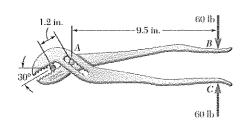
 $D = 150.0 \text{ kN} - \blacktriangleleft$ 

$$\pm \Sigma F_x = 0$$
: 56.25 kN - 150 kN +  $F_x = 0$ 

$$F_v = 93.75 \text{ kN}$$



 $F = 96.4 \text{ kN} \le 13.50^{\circ} \blacktriangleleft$ 



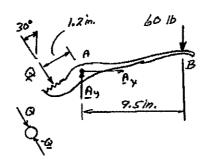
The pliers shown are used to grip a 0.3-in.-diameter rod. Knowing that two 60-lb forces are applied to the handles, determine (a) the magnitude of the forces exerted on the rod, (b) the force exerted by the pin at A on portion AB of the pliers.

# SOLUTION

Free body: Portion AB

+)
$$\Sigma M_A = 0$$
:  $Q(1.2 \text{ in.}) - (60 \text{ lb})(9.5 \text{ in.}) = 0$ 

 $Q = 475 \, \text{lb} \, \blacktriangleleft$ 



$$+ \Sigma F_x = 0$$
:  $Q(\sin 30^\circ) + A_x = 0$ 

$$(475 \text{ lb})(\sin 30^\circ) + A_x = 0$$

$$A_x = -237.5 \text{ lb}$$
  $A_y = 237.5 \text{ lb}$ 

+ 
$$\Sigma F_y = 0$$
:  $-Q(\cos 30^\circ) + A_y - 60 \text{ lb} = 0$ 

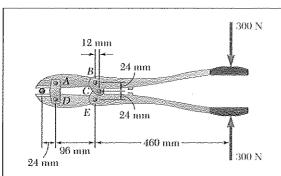
$$-(475 \text{ lb})(\cos 30^\circ) + A_v - 60 \text{ lb} = 0$$

237.58 471.416

$$A_v = +471.4 \text{ lb}$$

$$A_v = 471.4 \text{ lb}^{\dagger}$$

$$A = 528 \text{ lb} \ge 63.3^{\circ} \blacktriangleleft$$



In using the bolt cutter shown, a worker applies two 300-N forces to the handles. Determine the magnitude of the forces exerted by the cutter on the bolt.

## SOLUTION

FBD cutter AB:

Bx B €y

12 + €y

12 + €y

448

FBD handle BC:

Dimensions in mm

П

FBD 1:  $\rightarrow \Sigma F_x = 0$ :  $B_x = 0$ 

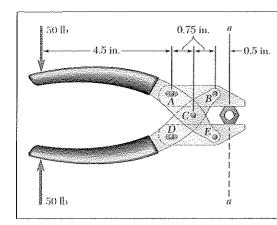
FBD II:  $(\Sigma M_C = 0)$ :  $(12 \text{ mm})B_y - (448 \text{ mm})300 \text{ N} = 0$ 

 $B_y = 11,200.0 \text{ N}$ 

Then

FBD I:  $(\Sigma M_A = 0: (96 \text{ mm})B_y - (24 \text{ mm})F = 0$ 

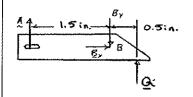
F = 44,800 N = 44.8 kN



Determine the magnitude of the gripping forces exerted along line aa on the nut when two 50-lb forces are applied to the handles as shown. Assume that pins A and D slide freely in slots cut in the jaws.

## **SOLUTION**

FBD jaw AB:



$$\Sigma F_x = 0$$
:  $B_x = 0$ 

$$\sum M_B = 0$$
:  $(0.5 \text{ in.})Q - (1.5 \text{ in.})A = 0$ 

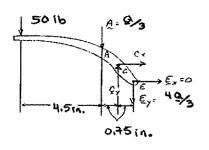
$$A = \frac{Q}{3}$$

$$\uparrow \Sigma F_y = 0: \quad A + Q - B_y = 0$$

$$B_y = A + Q = \frac{4Q}{3}$$

FBD handle ACE:

By symmetry and FBD jaw *DE*:



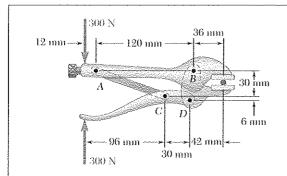
$$D = A = \frac{Q}{3}$$

$$E_x = B_x = 0$$

$$E_y = B_y = \frac{4Q}{3}$$

$$\sum M_C = 0$$
:  $(5.25 \text{ in.})(50 \text{ lb}) + (0.75 \text{ in.})\frac{Q}{3} - (0.75 \text{ in.})\frac{4Q}{3} = 0$ 

 $Q = 350 \, \text{lb} \, \blacktriangleleft$ 

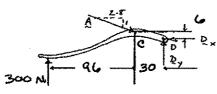


Determine the magnitude of the gripping forces produced when two 300-N forces are applied as shown.

# SOLUTION

We note that AC is a two-force member

### FBD handle CD:

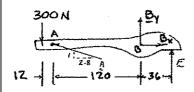


Dimensions in mm

$$\left(\sum M_D = 0: -(126 \text{ mm})(300 \text{ N}) - (6 \text{ mm})\frac{2.8}{\sqrt{8.84}}A\right)$$
  
+  $(30 \text{ mm})\left(\frac{1}{\sqrt{8.84}}A\right) = 0$ 

$$A = 2863.6\sqrt{8.84}$$
 N

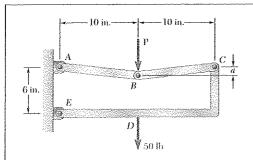
# FBD handle AB:



Dimensions in mm

$$\sum M_B = 0$$
:  $(132 \text{ mm})(300 \text{ N}) - (120 \text{ mm}) \frac{1}{\sqrt{8.84}} (2863.6\sqrt{8.84} \text{ N}) + (36 \text{ mm})F = 0$ 

F = 8.45 kN



Knowing that the frame shown has a sag at B of a = 1 in., determine the force **P** required to maintain equilibrium in the position shown.

### **SOLUTION**

We note that AB and BC are two-force members

Free body: Toggle

By symmetry:

$$C_y = \frac{P}{2}$$

$$\frac{C_x}{10 \text{ in.}} = \frac{C_y}{a}$$

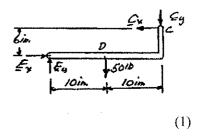
$$C_x = \frac{10}{a}C_y = \frac{10}{a} \cdot \frac{P}{2} = \frac{5P}{a}$$

Free body: Member CDE

+) 
$$\Sigma M_E = 0$$
:  $C_x(6 \text{ in.}) - C_y(20 \text{ in.}) - (50 \text{ lb})(10 \text{ in.}) = 0$   

$$\frac{5P}{a}(b) - \frac{P}{2}(20) = 500$$

$$P\left(\frac{30}{a} - 10\right) = 500$$

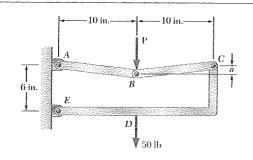


For

$$a = 1.0 \text{ in.}$$

$$P\left(\frac{30}{1} - 10\right) = 500$$

$$20P = 500$$



Knowing that the frame shown has a sag at B of a = 0.5 in., determine the force **P** required to maintain equilibrium in the position shown.

### **SOLUTION**

We note that AB and BC are two-force members

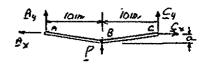
Free body: Toggle

By symmetry:

$$C_y = \frac{P}{2}$$

$$\frac{C_x}{10 \text{ in.}} = \frac{C_y}{a}$$

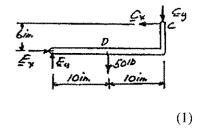
$$C_x = \frac{10}{a}C_y = \frac{10}{a} \cdot \frac{P}{2} = \frac{5P}{a}$$



Free body: Member CDE

+) 
$$\Sigma M_E = 0$$
:  $C_x(6 \text{ in.}) - C_y(20 \text{ in.}) - (50 \text{ lb})(10 \text{ in.}) = 0$ 

$$\frac{5P}{a}(6) - \frac{P}{2}(20) = 500$$
$$P\left(\frac{30}{a} - 10\right) = 500$$



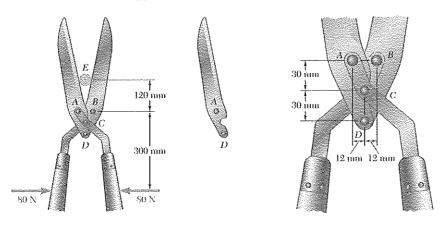
For

$$a = 0.5$$
 in.

$$P\left(\frac{30}{0.5} - 10\right) = 500$$
$$50P = 500$$

$$P = 10.00 \text{ lb} \downarrow \blacktriangleleft$$

The garden shears shown consist of two blades and two handles. The two handles are connected by pin C and the two blades are connected by pin D. The left blade and the right handle are connected by pin A; the right blade and the left handle are connected by pin B. Determine the magnitude of the forces exerted on the small branch at E when two 80-N forces are applied to the handles as shown.



### **SOLUTION**

By symmetry vertical components  $C_y$ ,  $D_y$ ,  $E_y$  are 0. Then by considering  $\Sigma F_y = 0$  on the blades or handles, we find that  $A_y$  and  $B_y$  are 0.

Thus forces at A, B, C, D, and E are horizontal.

Free body: Right handle

+)
$$\Sigma M_C = 0$$
:  $A(30 \text{ mm}) - (80 \text{ N})(270 \text{ mm}) = 0$ 

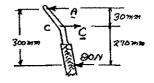
A = +720 N

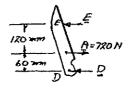
$$\pm \Sigma F_x = 0$$
:  $C - 720 \text{ N} - 80 \text{ N} = 0$ 

C = +800 N

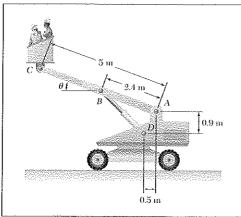
Free body: Left blade

+)
$$\Sigma M_D = 0$$
:  $E(180 \text{ mm}) - (720 \text{ N})(60 \text{ mm}) = 0$ 





E = 240 N



The telescoping arm ABC is used to provide an elevated platform for construction workers. The workers and the platform together have a mass of 200 kg and have a combined center of gravity located directly above C. For the position when  $\theta = 20^{\circ}$ , determine (a) the force exerted at B by the single hydraulic cylinder BD, (b) the force exerted on the supporting carriage at A.

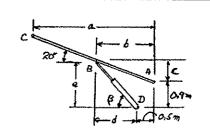
#### **SOLUTION**

Geometry:

$$a = (5 \text{ m})\cos 20^{\circ} = 4.6985 \text{ m}$$
  
 $b = (2.4 \text{ m})\cos 20^{\circ} = 2.2553 \text{ m}$   
 $c = (2.4 \text{ m})\sin 20^{\circ} = 0.8208 \text{ m}$   
 $d = b - 0.5 = 1.7553 \text{ m}$ 

$$\tan \beta = \frac{e}{d} = \frac{1.7208}{1.7553}; \quad \beta = 44.43^{\circ}$$

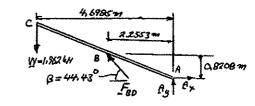
e = c + 0.9 = 1.7208 m

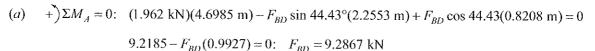


# Free body: Arm ABC

We note that BD is a two-force member

$$W = (200 \text{ kg})(9.81 \text{ m/s}^2) = 1.962 \text{ kN}$$





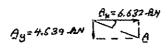
(b) 
$$\mathbf{F}_{BD} = 9.29 \text{ kN} \implies 44.4^{\circ} \blacktriangleleft$$

$$\frac{+}{\Delta F_{x}} = 0: \quad A_{x} - F_{BD} \cos \beta = 0$$

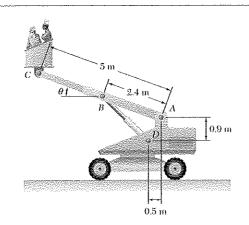
$$A_{x} = (9.2867 \text{ kN}) \cos 44.43^{\circ} = 6.632 \text{ kN} \qquad \mathbf{A}_{x} = 6.632 \text{ kN} \rightarrow$$

$$+ | \Sigma F_{y} = 0: \quad A_{y} - 1.962 \text{ kN} + F_{BD} \sin \beta = 0$$

$$A_{y} = 1.962 \text{ kN} - (9.2867 \text{ kN}) \sin 44.43^{\circ} = -4.539 \text{ kN}$$



$$A_y = 4.539 \text{ kN} \downarrow$$
 $A = 8.04 \text{ kN} \le 34.4^{\circ} \blacktriangleleft$ 



The telescoping arm ABC can be lowered until end C is close to the ground, so that workers can easily board the platform. For the position when  $\theta = -20^{\circ}$ , determine (a) the force exerted at B by the single hydraulic cylinder BD, (b) the force exerted on the supporting carriage at A.

# **SOLUTION**

Geometry:

(b)

$$a = (5 \text{ m})\cos 20^{\circ} = 4.6985 \text{ m}$$

$$b = (2.4 \text{ m})\cos 20^{\circ} = 2.2552 \text{ m}$$

$$c = (2.4 \text{ m})\sin 20^{\circ} = 0.8208 \text{ m}$$

$$d = b - 0.5 = 1.7553$$
 m

$$e = 0.9 - c = 0.0792$$
 m

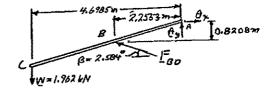
$$\tan \beta = \frac{e}{d} = \frac{0.0792}{1.7552}; \quad \beta = 2.584^{\circ}$$



We note that BD is a two-force member

$$W = (200 \text{ kg})(9.81 \text{ m/s}^2)$$

$$W = 1962 \text{ N} = 1.962 \text{ kN}$$



(a) 
$$+^{\times} \Sigma M_A = 0$$
:  $(1.962 \text{ kN})(4.6985 \text{ m}) - F_{BD} \sin 2.584^{\circ}(2.2553 \text{ m}) - F_{BD} \cos 2.584^{\circ}(0.8208 \text{ m}) = 0$ 

$$9.2185 - F_{BD}(0.9216) = 0$$
  $F_{BD} = 10.003$  kN

$$+\sum F_{x}=0$$
:  $A_{x}-F_{BD}\cos\beta=0$ 

$$A_{\rm r} = (10.003 \text{ kN})\cos 2.583^{\circ} = 9.993 \text{ kN}$$

$$A_r = 9.993 \text{ kN} \longrightarrow$$

 $F_{BD} = 10.00 \text{ kN} \ge 2.58^{\circ} \blacktriangleleft$ 

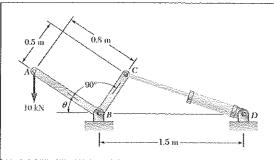
$$+\sum F_y = 0$$
:  $A_y - 1.962 \text{ kN} + F_{BD} \sin \beta = 0$ 

$$A_v = 1.962 \text{ kN} - (10.003 \text{ kN}) \sin 2.583^\circ = -1.5112 \text{ kN}$$

Ay= 1.51241 - A

$$A_y = 1.5112 \text{ kN}$$

$$A = 10.11 \text{ kN} \le 8.60^{\circ} \blacktriangleleft$$



The position of member ABC is controlled by the hydraulic cylinder CD. Knowing that  $\theta = 30^{\circ}$ , determine for the loading shown (a) the force exerted by the hydraulic cylinder on pin C, (b) the reaction at B.

### **SOLUTION**

Geometry: In  $\Delta BCD$ 

Law of cosines

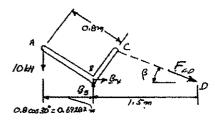
$$(CD)^2 = (0.5)^2 + (1.5)^2 - 2(0.5)(1.5)\cos 60^\circ$$

$$CD = 1.3229 \text{ m}$$

Law of sines

$$\frac{\sin \beta}{0.5 \text{ m}} = \frac{\sin 60^{\circ}}{1.3229 \text{ m}}$$

$$\sin \beta = 0.3273$$
  $\beta = 19.107^{\circ}$ 



Free body: Entire system

Move force  $F_{CD}$  along its line of action so it acts at D.

(a) 
$$+ \sum M_B = 0$$
:  $(10 \text{ kN})(0.69282 \text{ m}) - F_{CD} \sin \beta (1.5 \text{ m}) = 0$ 

$$6.9282 \text{ kN} \cdot \text{m} - F_{CD} \sin 19.107^{\circ} (1.5 \text{ m}) = 0$$

$$F_{CD} = 14.111 \text{ kN}$$

$$\mathbf{F}_{CD} = 14.11 \text{ kN} \le 19.11^{\circ} \blacktriangleleft$$

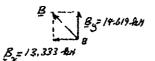
(b) 
$$\xrightarrow{+} \Sigma F_x = 0: \quad B_x + F_{CD} \cos \beta = 0$$

$$B_{\rm y} + (14.111 \, \text{kN}) \cos 19.107^{\circ} = 0$$

$$B_x = -13.333 \text{ kN}$$

$$B_r = 13.333 \text{ kN} \leftarrow$$

$$+ \sum F_y = 0$$
:  $B_y - 10 \text{ kN} - F_{CD} \sin 19.107^\circ = 0$ 



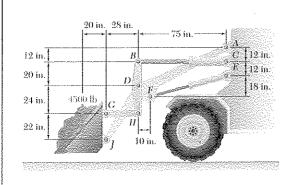
$$B_y - 10 \text{ kN} - (14.111 \text{ kN}) \sin 19.107^\circ = 0$$

$$B_y = +14.619 \text{ kN}$$

$$B_v = +14.619 \text{ kN}$$

$$B_y = 14.619 \text{ kN}$$

$$B = 19.79 \text{ kN} \implies 47.6^{\circ} \blacktriangleleft$$



The motion of the bucket of the front-end loader shown is controlled by two arms and a linkage that are pin-connected at D. The arms are located symmetrically with respect to the central, vertical, and longitudinal plane of the loader; one arm AFJ and its control cylinder EF are shown. The single linkage GHDB and its control cylinder BC are located in the plane of symmetry. For the position and loading shown, determine the force exerted (a) by cylinder BC, (b) by cylinder EF.

## **SOLUTION**

Free body: Bucket

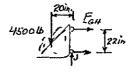
$$+ \Sigma M_I = 0$$
: (4500 lb)(20 in.)  $- F_{GH}$ (22 in.) = 0

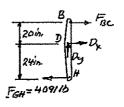
$$F_{GH} = 4091 \text{ lb}$$

Free body: Arm BDH

+)
$$\Sigma M_D = 0$$
:  $-(4091 \text{ lb})(24 \text{ in.}) - F_{BC}(20 \text{ in.}) = 0$ 

 $F_{BC} = -4909 \text{ lb}$ 

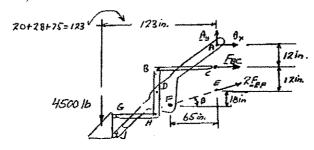




 $F_{RC} = 4.91 \text{ kips}$  C

Free body: Entire mechanism

(Two arms and cylinders AFJE)



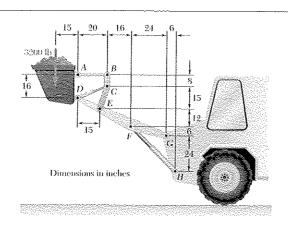
*Note*: Two arms thus  $2F_{FF}$ 

$$\tan \beta = \frac{18 \text{ in.}}{65 \text{ in.}}$$
$$\beta = 15.48^{\circ}$$

+)
$$\Sigma M_A = 0$$
: (4500 lb)(123 in.) +  $F_{BC}$ (12 in.) +  $2F_{EF}$  cos  $\beta$ (24 in.) = 0  
(4500 lb)(123 in.) - (4909 lb)(12 in.) +  $2F_{EF}$  cos 15.48°(24 in.) = 0

$$F_{FF} = -10.690 \text{ lb}$$

$$F_{EF} = 10.69 \text{ kips}$$
  $C \blacktriangleleft$ 



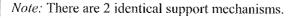
The bucket of the front-end loader shown carries a 3200-lb load. The motion of the bucket is controlled by two identical mechanisms, only one of which is shown. Knowing that the mechanism shown supports one-half of the 3200-lb load, determine the force exerted (a) by cylinder CD, (b) by cylinder FH.

### SOLUTION

Free body: Bucket (One mechanism)

+)
$$\Sigma M_D = 0$$
: (1600 lb)(15 in.) -  $F_{AB}$ (16 in.) = 0

$$F_{AB} = 1500 \text{ lb}$$

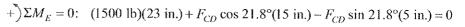


Free body: One arm BCE

$$\tan \beta = \frac{8}{20}$$

$$\beta = 21.8$$

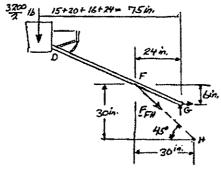
$$\beta = 21.8^{\circ}$$



$$F_{CD} = -2858 \text{ lb}$$

 $F_{CD} = 2.86 \text{ kips}$  C

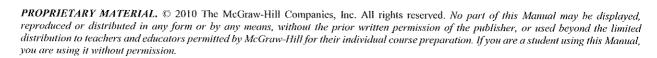
Free body: Arm DFG

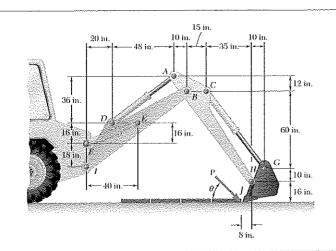


+) 
$$\Sigma M_G = 0$$
: (1600 lb)(75 in.) +  $F_{FH}$  sin 45°(24 in.) -  $F_{FH}$  cos 45°(6 in.) = 0

$$F_{FH} = -9.428 \text{ kips}$$

$$F_{FH} = 9.43 \text{ kips}$$
  $C \blacktriangleleft$ 





The motion of the backhoe bucket shown is controlled by the hydraulic cylinders AD, CG, and EF. As a result of an attempt to dislodge a portion of a slab, a 2-kip force **P** is exerted on the bucket teeth at J. Knowing that  $\theta = 45^{\circ}$ , determine the force exerted by each cylinder.

### **SOLUTION**

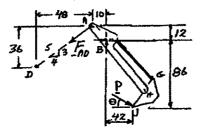
Free body: Bucket

+) 
$$\Sigma M_H = 0$$
 (Dimensions in inches)
$$\frac{4}{5} F_{CG}(10) + \frac{3}{5} F_{CG}(10) + P \cos \theta(16) + P \sin \theta(8) = 0$$

$$F_{CG} = -\frac{P}{14} (16 \cos \theta + 8 \sin \theta)$$

Free body: Arm ABH and bucket

(Dimensions in inches)



$$+\sum M_B = 0: \quad \frac{4}{5} F_{AD}(12) + \frac{3}{5} F_{AD}(10) + P\cos\theta(86) - P\sin\theta(42) = 0$$

$$F_{AD} = -\frac{P}{15.6} (86\cos\theta - 42\sin\theta) \tag{2}$$

Free body: Bucket and arms IEB + ABH

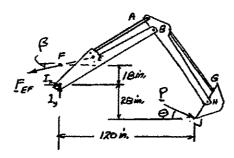
Geometry of cylinder EF

 $\tan \beta = \frac{16 \text{ in.}}{40 \text{ in.}}$  $\beta = 21.801^{\circ}$ 



(1)

# PROBLEM 6.157 (Continued)



+) $\Sigma M_I = 0$ :  $F_{EF} \cos \beta (18 \text{ in.}) + P \cos \theta (28 \text{ in.}) - P \sin \theta (120 \text{ in.}) = 0$ 

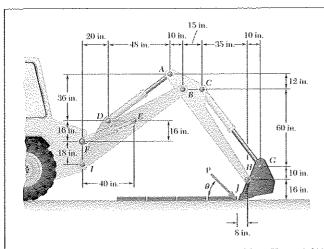
$$F_{EF} = \frac{P(120\sin\theta - 28\cos\theta)}{\cos 21.8^{\circ}(18)}$$
$$= \frac{P}{16.7126} (120\sin\theta - 28\cos\theta) \tag{3}$$

For P = 2 kips,  $\theta = 45^{\circ}$ 

Eq. (1): 
$$F_{CG} = -\frac{2}{14} (16\cos 45^\circ + 8\sin 45^\circ) = -2.42 \text{ kips}$$
  $F_{CG} = 2.42 \text{ kips}$   $C \blacktriangleleft$ 

Eq. (2): 
$$F_{AD} = -\frac{2}{15.6} (86 \cos 45^\circ - 42 \sin 45^\circ) = -3.99 \text{ kips}$$
  $F_{AD} = 3.99 \text{ kips}$   $C \blacktriangleleft$ 

Eq. (3): 
$$F_{EF} = \frac{2}{16.7126} (120 \sin 45^\circ - 28 \cos 45^\circ) = +7.79 \text{ kips}$$
  $F_{EF} = 7.79 \text{ kips}$   $T \blacktriangleleft$ 



Solve Problem 6.157 assuming that the 2-kip force **P** acts horizontally to the right ( $\theta = 0$ ).

**PROBLEM 6.157** The motion of the backhoe bucket shown is controlled by the hydraulic cylinders AD, CG, and EF. As a result of an attempt to dislodge a portion of a slab, a 2-kip force **P** is exerted on the bucket teeth at J. Knowing that  $\theta = 45^{\circ}$ , determine the force exerted by each cylinder.

# **SOLUTION**

Free body: Bucket

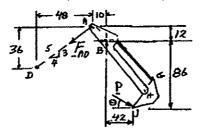
$$+)\Sigma M_H = 0$$
 (Dimensions in inches)

$$\frac{4}{5}F_{CG}(10) + \frac{3}{5}F_{CG}(10) + P\cos\theta(16) + P\sin\theta(8) = 0$$

$$F_{CG} = -\frac{P}{14}(16\cos\theta + 8\sin\theta) \tag{1}$$

Free body: Arm ABH and bucket

(Dimensions in inches)



+)
$$\Sigma M_B = 0$$
:  $\frac{4}{5}F_{AD}(12) + \frac{3}{5}F_{AD}(10) + P\cos\theta(86) - P\sin\theta(42) = 0$ 

$$F_{AD} = -\frac{P}{15.6} (86\cos\theta - 42\sin\theta)$$
 (2)

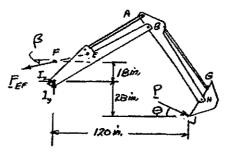
Free body: Bucket and arms IEB + ABH

Geometry of cylinder EF

$$\tan \beta = \frac{16 \text{ in.}}{40 \text{ in.}}$$
$$\beta = 21.801^{\circ}$$



# PROBLEM 6.158 (Continued)



+) $\Sigma M_I = 0$ :  $F_{EF} \cos \beta (18 \text{ in.}) + P \cos \theta (28 \text{ in.}) - P \sin \theta (120 \text{ in.}) = 0$ 

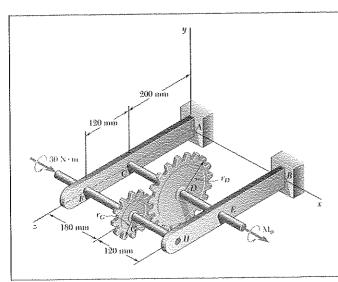
$$F_{EF} = \frac{P(120\sin\theta - 28\cos\theta)}{\cos 21.8^{\circ}(18)}$$
$$= \frac{P}{16.7126} (120\sin\theta - 28\cos\theta) \tag{3}$$

For P = 2 kips,  $\theta = 0$ 

Eq. (1): 
$$F_{CG} = -\frac{2}{14} (16 \cos 0 + 8 \sin 0) = -2.29 \text{ kips}$$
  $F_{CG} = 2.29 \text{ kips}$   $C \blacktriangleleft$ 

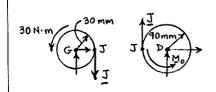
Eq. (2): 
$$F_{AD} = -\frac{2}{15.6} (86 \cos 0 - 42 \sin 0) = -11.03 \text{ kips} \qquad F_{AD} = 11.03 \text{ kips} \qquad C \blacktriangleleft$$

Eq. (3): 
$$F_{EF} = \frac{2}{16.7126} (120 \sin 0 - 28 \cos 0) = -3.35 \text{ kips}$$
  $F_{EF} = 3.35 \text{ kips}$   $C \blacktriangleleft$ 



The gears D and G are rigidly attached to shafts that are held by frictionless bearings. If  $r_D = 90$  mm and  $r_G = 30$  mm, determine (a) the couple  $\mathbf{M}_0$  that must be applied for equilibrium, (b) the reactions at A and B.

# SOLUTION



(a) Projections on yz plane

Free body: Gear G

+)
$$\Sigma M_G = 0$$
: 30 N·m- $J(0.03 \text{ m}) = 0$ ;  $J = 1000 \text{ N}$ 

Free body: Gear D

+)
$$\Sigma M_D = 0$$
:  $M_0 - (1000 \text{ N})(0.09 \text{ m}) = 0$ 

$$M_0 = 90 \text{ N} \cdot \text{m}$$
  $M_0 = (90.0 \text{ N} \cdot \text{m})i$ 

(b) Gear G and axle FH

+)
$$\Sigma M_F = 0$$
:  $H(0.3 \text{ m}) - (1000 \text{ N})(0.18 \text{ m}) = 0$ 

$$H = 600 \text{ N}$$

$$+ \sum F_v = 0$$
:  $F + 600 - 1000 = 0$ 

$$F = 400 \text{ N}$$

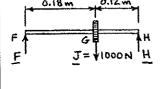
Gear D and axle CE

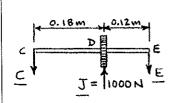
+)
$$\Sigma M_C = 0$$
: (1000 N)(0.18 m) –  $E(0.3 \text{ m}) = 0$ 

$$E = 600 \text{ N}$$

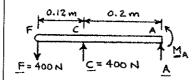
$$+\sum F_{\nu} = 0$$
:  $1000 - C - 600 = 0$ 

$$C = 400 \text{ N}$$





# PROBLEM 6.159 (Continued)



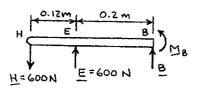
Free body: Bracket AE

$$+\int \Sigma F_y = 0$$
:  $A - 400 + 400 = 0$ 

A = 0

+)
$$\Sigma M_A = 0$$
:  $M_A + (400 \text{ N})(0.32 \text{ m}) - (400 \text{ N})(0.2 \text{ m}) = 0$ 

$$M_A = -(48.0 \text{ N} \cdot \text{m})i$$



Free body: Bracket BH

$$+ \sum F_y = 0$$
:  $B - 600 + 600 = 0$ 

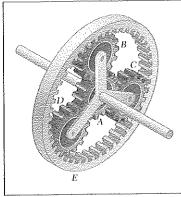
 $\mathbf{B} = 0$ 

+)
$$\Sigma M_B = 0$$
:  $M_B + (600 \text{ N})(0.32 \text{ m}) - (600 \text{ N})(0.2 \text{ m}) = 0$ 

 $M_A = -48 \text{ N} \cdot \text{m}$ 

$$M_B = -72 \text{ N} \cdot \text{m}$$

$$\mathbf{M}_B = -(72.0 \text{ N} \cdot \text{m})\mathbf{i} \blacktriangleleft$$



In the planetary gear system shown, the radius of the central gear A is a = 18 mm, the radius of each planetary gear is b, and the radius of the outer gear E is (a + 2b). A clockwise couple of magnitude  $M_A = 10 \text{ N} \cdot \text{m}$  is applied to the central gear A and a counterclockwise couple of magnitude  $M_S = 50 \text{ N} \cdot \text{m}$  is applied to the spider BCD. If the system is to be in equilibrium, determine (a) the required radius b of the planetary gears, (b) the magnitude  $M_E$  of the couple that must be applied to the outer gear E.

### **SOLUTION**

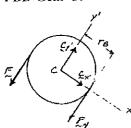
FBD Central Gear:

By symmetry:  $F_1 = F_2 = F_3 = F$ 

$$\sum M_A = 0$$
:  $3(r_A F) - 10 \text{ N} \cdot \text{m} = 0$ ,  $F = \frac{10}{3r_A} \text{ N} \cdot \text{m}$ 

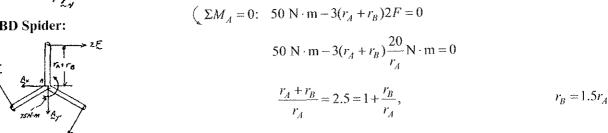
FBD Gear C:

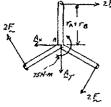
$$\sum M_C = 0$$
:  $r_B(F - F_4) = 0$ ,  $F_4 = F$   
 $\sum F_{x'} = 0$ :  $C_{x'} = 0$   
 $\sum F_{y'} = 0$ :  $C_{y'} - 2F = 0$ ,  $C_{y'} = 2F$ 



Gears B and D are analogous, each having a central force of 2F

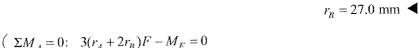
FBD Spider:

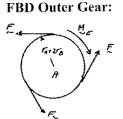




Since  $r_A = 18 \text{ mm}$ ,

(a)



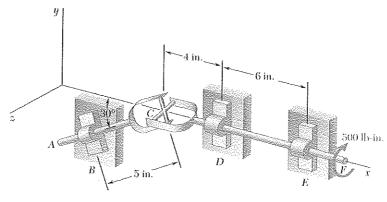


$$\sum M_A = 0$$
:  $3(r_A + 2r_B)F - M_E = 0$   
  $3(18 \text{ mm} + 54 \text{ mm}) \frac{10 \text{ N} \cdot \text{m}}{54 \text{ mm}} - M_E = 0$ 

 $\mathbf{M}_E = 40.0 \,\mathrm{N \cdot m}$ (b)

#### **PROBLEM 6.161\***

Two shafts AC and CF, which lie in the vertical xy plane, are connected by a universal joint at C. The bearings at B and D do not exert any axial force. A couple of magnitude 500 lb  $\cdot$  in. (clockwise when viewed from the positive x axis) is applied to shaft CF at F. At a time when the arm of the crosspiece attached to shaft CF is horizontal, determine (a) the magnitude of the couple that must be applied to shaft AC at A to maintain equilibrium, (b) the reactions at B, D, and E. (Hint: The sum of the couples exerted on the crosspiece must be zero.)



#### **SOLUTION**

We recall from Figure 4.10, that a universal joint exerts on members it connects a force of unknown direction and a couple about an axis perpendicular to the crosspiece.

Free body: Shaft DF

$$\Sigma M_x = 0$$
:  $M_C \cos 30^\circ - 500 \text{ lb} \cdot \text{in} = 0$ 

 $M_C = 577.35 \text{ lb} \cdot \text{in}.$ 

Free body: Shaft BC

We use here x', y', z with x' along BC

$$\Sigma M_C = 0$$
:  $-M_R i' - (577.35 \text{ lb} \cdot \text{in})i' + (-5 \text{ in})i' \times (B_v j' + B_z k) = 0$ 

# PROBLEM 6.161\* (Continued)

Equate coefficients of unit vectors to zero:

i: 
$$M_A - 577.35 \text{ lb} \cdot \text{in} = 0$$

$$M_A = 577.35 \text{ lb} \cdot \text{in}.$$

$$B_z = 0$$

$$M_A = 577 \text{ lb} \cdot \text{in.} \blacktriangleleft$$

$$B_z = 0$$

$$B_y = 0$$

$$B = 0$$

$$\mathbf{B} = 0$$

$$\Sigma \mathbf{F} = 0$$

$$\Sigma \mathbf{F} = 0$$
:  $B + C = 0$ , since  $B = 0$ ,

$$C = 0$$

Return to free body of shaft DF

$$\Sigma \mathbf{M}_{D} = 0 \qquad \text{(Note that } C = 0 \text{ and } M_{C} = 577.35 \text{ lb} \cdot \text{in.)}$$

$$(577.35 \text{ lb} \cdot \text{in.)} (\cos 30^{\circ} \mathbf{i} + \sin 30^{\circ} \mathbf{j}) - (500 \text{ lb} \cdot \text{in.}) \mathbf{i}$$

$$+ (6 \text{ in.)} \mathbf{i} \times (E_{x} \mathbf{i} + E_{y} \mathbf{j} + E_{z} \mathbf{k}) = 0$$

$$(500 \text{ lb} \cdot \text{in.}) \mathbf{i} + (288.68 \text{ lb} \cdot \text{in.}) \mathbf{j} - (500 \text{ lb} \cdot \text{in.}) \mathbf{i}$$

$$+ (6 \text{ in.}) E_{y} \mathbf{k} - (6 \text{ in.}) E_{z} \mathbf{j} = 0$$

Equate coefficients of unit vectors to zero:

$$288.68 \text{ lb} \cdot \text{in.} - (6 \text{ in.})E_z = 0$$
  $E_z = 48.1 \text{ lb}$ 

$$E_v = 0$$

$$\Sigma \mathbf{F} = 0: \quad \mathbf{C} + \mathbf{D} + \mathbf{E} = 0$$

$$0 + D_y \mathbf{j} + D_z \mathbf{k} + E_x \mathbf{i} + (48.1 \text{ lb}) \mathbf{k} = 0$$

$$E_x = 0$$

$$D_y = 0$$

$$D_z + 48.1 \text{ lb} = 0$$
  $D_z = -48.1 \text{ lb}$ 

Reactions are:

$$\mathbf{B} = 0$$

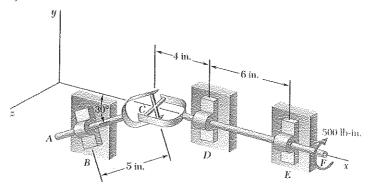
$$\mathbf{D} = -(48.11b)\mathbf{k}$$

$$E = (48.1 lb)k$$

#### **PROBLEM 6.162\***

Solve Problem 6.161 assuming that the arm of the crosspiece attached to shaft CF is vertical.

**PROBLEM 6.161** Two shafts AC and CF, which lie in the vertical xy plane, are connected by a universal joint at C. The bearings at B and D do not exert any axial force. A couple of magnitude 500 lb·in. (clockwise when viewed from the positive x axis) is applied to shaft CF at F. At a time when the arm of the crosspiece attached to shaft CF is horizontal, determine (a) the magnitude of the couple that must be applied to shaft AC at A to maintain equilibrium, (b) the reactions at B, D, and E. (Hint: The sum of the couples exerted on the crosspiece must be zero.)

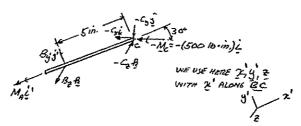


#### SOLUTION

Free body: Shaft DF

$$\Sigma M_x = 0$$
:  $M_C - 500 \text{ lb} \cdot \text{in.} = 0$   
 $M_C = 500 \text{ lb} \cdot \text{in.}$ 

Free body: Shaft BC



We resolve  $-(520 \text{ lb} \cdot \text{in.})i$  into components along x' and y' axes:

$$-\mathbf{M}_{C} = -(500 \text{ lb} \cdot \text{in.})(\cos 30^{\circ} \mathbf{i}' + \sin 30^{\circ} \mathbf{j}')$$

$$\Sigma \mathbf{M}_{C} = 0: \quad M_{A} \mathbf{i}' - (500 \text{ lb} \cdot \text{in.})(\cos 30^{\circ} \mathbf{i}' + \sin 30^{\circ} \mathbf{j}') + (5 \text{ in.}) \mathbf{i}' \times (B_{y'} \mathbf{j}' + B_{z} \mathbf{k}) = 0$$

$$M_{A} \mathbf{i}' - (433 \text{ lb} \cdot \text{in.}) \mathbf{i}' - (250 \text{ lb} \cdot \text{in.}) \mathbf{j} + (5 \text{ in.}) B_{y'} \mathbf{k} - (5 \text{ in.}) B_{z} \mathbf{j}' = 0$$

# PROBLEM 6.162\* (Continued)

Equate to zero coefficients of unit vectors:

i': 
$$M_A - 433 \text{ lb} \cdot \text{in.} = 0$$

$$M_A = 433 \text{ lb} \cdot \text{in.}$$

$$j'$$
:  $-250 \text{ lb} \cdot \text{in.} - (5 \text{ in.})B_z = 0$ 

$$B_z = -50 \text{ lb}$$

$$\mathbf{k} \colon \mathbf{B}_{\mathbf{v}'} = 0$$

Reactions at B:

$$\mathbf{B} = -(50 \text{ lb})\mathbf{k}$$

$$\Sigma \mathbf{F} = 0$$
:  $\mathbf{B} - \mathbf{C} = 0$ 

$$-(50 \text{ lb})\mathbf{k} - C = 0$$

$$C = -(50 \text{ lb})k$$

Return to free body of shaft DF:

$$\Sigma \mathbf{M}_D = 0$$
:  $(6 \text{ in.})\mathbf{i} \times (E_x \mathbf{i} + E_y \mathbf{j} + E_z \mathbf{k}) - (4 \text{ in.})\mathbf{i} \times (-50 \text{ lb})\mathbf{k}$ 

$$-(500 \text{ lb in.})\mathbf{i} + (500 \text{ lb in.})\mathbf{i} = 0$$

$$(6 \text{ in.})E_v \mathbf{k} - (6 \text{ in.})E_z \mathbf{j} - (200 \text{ lb} \cdot \text{in.})\mathbf{j} = 0$$

**k**: 
$$E_y = 0$$

**j**: 
$$-(6 \text{ in.})E_z - 200 \text{ lb} \cdot \text{in.} = 0$$

$$E_z = -33.3 \text{ lb}$$

$$\Sigma F = 0$$
:  $\mathbf{C} + \mathbf{D} + \mathbf{E} = 0$ 

$$-(50 \text{ lb})\mathbf{k} + D_y \mathbf{j} + D_z \mathbf{k} + E_x \mathbf{i} - (33.3 \text{ lb})\mathbf{k} = 0$$

i: 
$$E_x = 0$$

**k**: 
$$-50 \text{ lb} - 33.3 \text{ lb} + D_z = 0$$

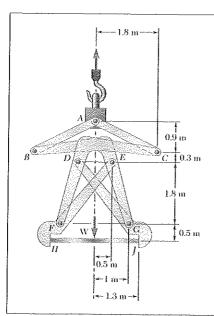
$$D_z = 83.3 \text{ lb}$$

Reactions are:

$$B = -(50 \text{ lb})k$$

$$D = (83.3 \text{ lb})k$$

$$E = -(33.3 \text{ lb})k$$



# **PROBLEM 6.163\***

The large mechanical tongs shown are used to grab and lift a thick 7500-kg steel slab *HJ*. Knowing that slipping does not occur between the tong grips and the slab at *H* and *J*, determine the components of all forces acting on member *EFH*. (*Hint:* Consider the symmetry of the tongs to establish relationships between the components of the force acting at *E* on *EFH* and the components of the force acting at *D* on *CDF*.)

#### **SOLUTION**

Free body: Pin A

$$T = W = mg = (7500 \text{ kg})(9.81 \text{ m/s}^2) = 73.575 \text{ kN}$$

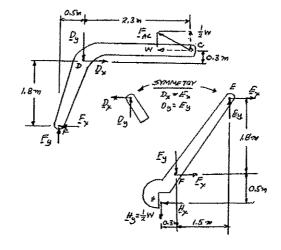
 $\Sigma F_x = 0$ :  $(F_{AB})_x = (F_{AC})_x$ 

$$\Sigma F_y = 0$$
:  $(F_{AB})_y = (F_{AC})_y = \frac{1}{2}W$ 

FAC A FAC

Also:

$$(F_{AC})_x = 2(F_{AC})_y = W$$



Free body: Member CDF

+)
$$\Sigma M_D = 0$$
:  $W(0.3) + \frac{1}{2}W(2.3) - F_x(1.8) - F_y(0.5 \text{ m}) = 0$   
 $1.8F_x + 0.5F_y = 1.45W$  (1)

or

# PROBLEM 6.163\* (Continued)

$$\stackrel{+}{\longrightarrow} \Sigma F_x = 0$$
:  $D_x - F_x - W = 0$ 

or

$$E_x - F_x = W \tag{2}$$

$$+ \sum F_y = 0$$
:  $F_y - D_y + \frac{1}{2}W = 0$ 

or

$$E_y - F_y = \frac{1}{2}W\tag{3}$$

Free body: Member EFH

+)
$$\Sigma M_E = 0$$
:  $F_x(1.8) + F_y(1.5) - H_x(2.3) + \frac{1}{2}W(1.8 \text{ m}) = 0$ 

or:

$$1.8F_x + 1.5F_y = 2.3H_x - 0.9W (4)$$

$$\stackrel{+}{\longrightarrow} \Sigma F_x = 0$$
:  $E_x + F_x - H_x = 0$ 

or

$$E_x + F_x = H_x \tag{5}$$

Subtract (2) from (5):

$$2F_{x} = H_{x} - W \tag{6}$$

Subtract (4) from  $3 \times (1)$ :

$$3.6F_{\rm x} = 5.25W - 2.3H_{\rm x} \tag{7}$$

Add (7) to  $2.3 \times (6)$ :

$$8.2F_x = 2.95W$$

$$F_{x} = 0.35976W \tag{8}$$

Substitute from (8) into (1):

$$(1.8)(0.35976W) + 0.5F_v = 1.45W$$

$$0.5F_y = 1.45W - 0.64756W = 0.80244W$$

$$F_{y} = 1.6049W \tag{9}$$

Substitute from (8) into (2):

$$E_{y} - 0.35976W = W; E_{y} = 1.35976W$$

Substitute from (9) into (3):

$$E_y - 1.6049W = \frac{1}{2}W$$
  $E_y = 2.1049W$ 

From (5):

$$H_x = E_x + F_x = 1.35976W + 0.35976W = 1.71952W$$

Recall that:

$$H_y = \frac{1}{2}W$$

# PROBLEM 6.163\* (Continued)

Since all expressions obtained are positive, all forces are directed as shown on the free-body diagrams.

Substitute

$$W = 73.575 \text{ kN}$$

$$\mathbf{E}_x = 100.0 \text{ kN} \longrightarrow$$

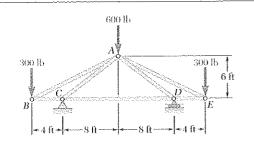
$$\mathbf{E}_y = 154.9 \,\mathrm{kN} \uparrow \blacktriangleleft$$

$$\mathbf{F}_x = 26.5 \text{ kN} \longrightarrow$$

$$\mathbf{F}_{y} = 118.1 \,\mathrm{kN} \,\mathsf{\downarrow} \,\blacktriangleleft$$

$$H_x = 126.5 \text{ kN} -$$

 $H_y = 36.8 \text{ kN}$ 



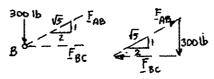
Using the method of joints, determine the force in each member of the truss shown. State whether each member is in tension or compression.

# SOLUTION

Free body: Truss.

From the symmetry of the truss and loading, we find

$$C = D = 600 \text{ lb}$$

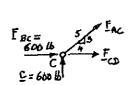


Free body: Joint B

$$\frac{F_{AB}}{\sqrt{5}} = \frac{F_{BC}}{2} = \frac{300 \text{ lb}}{1}$$

 $F_{AB} = 671 \, \text{lb}$  T  $F_{BC} = 600 \, \text{lb}$   $C \blacktriangleleft$ 

Free body: Joint C



+ 
$$\Sigma F_y = 0$$
:  $\frac{3}{5} F_{AC} + 600 \text{ lb} = 0$ 

$$F_{AC} = -1000 \text{ lb}$$

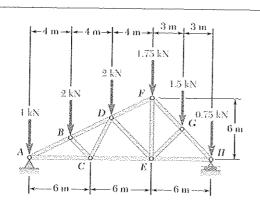
$$F_{AC} = 1000 \, \text{lb} \quad C \blacktriangleleft$$

$$\pm \Sigma F_x = 0$$
:  $\frac{4}{5}$ (-1000 lb) + 600 lb +  $F_{CD}$  = 0  $F_{CD}$  = 200 lb  $T$  ◀

$$F_{CD} = 200 \text{ lb}$$
  $T$ 

From symmetry:

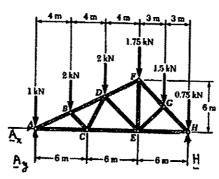
$$F_{AD} = F_{AC} = 1000 \text{ lb}$$
 C,  $F_{AE} = F_{AB} = 671 \text{ lb}$  T,  $F_{DE} = F_{BC} = 600 \text{ lb}$  C



Using the method of joints, determine the force in each member of the double-pitch roof truss shown. State whether each member is in tension or compression.

#### **SOLUTION**

Free body: Truss



+)
$$\Sigma M_A = 0$$
:  $H(18 \text{ m}) - (2 \text{ kN})(4 \text{ m}) - (2 \text{ kN})(8 \text{ m}) - (1.75 \text{ kN})(12 \text{ m})$   
-  $(1.5 \text{ kN})(15 \text{ m}) - (0.75 \text{ kN})(18 \text{ m}) = 0$ 

 $H = 4.50 \text{ kN}^{+}$ 

$$\Sigma F_x = 0$$
:  $A_x = 0$ 

$$\Sigma F_y = 0$$
:  $A_y + H - 9 = 0$ 

$$A_y = 9 - 4.50,$$

 $A_y = 4.50 \text{ kN}$ 

Free body: Joint A

$$\frac{F_{AB}}{\sqrt{5}} = \frac{F_{AC}}{2} = \frac{3.50 \text{ kN}}{1}$$
 $F_{AB} = 7.8262 \text{ kN} \quad C$ 

$$F_{AB} = 7.83 \text{ kN}$$
 C

$$F_{AC} = 7.00 \text{ kN}$$
 T

# PROBLEM 6.165 (Continued)

Free body: Joint B

$$\pm \Sigma F_x = 0$$
:  $\frac{2}{\sqrt{5}} F_{BD} + \frac{2}{\sqrt{5}} (7.8262 \text{ kN}) + \frac{1}{\sqrt{2}} F_{BC} = 0$ 

$$F_{BD} + 0.79057 F_{BC} = -7.8262 \text{ kN} \tag{1}$$

$$+\int \Sigma F_y = 0$$
:  $\frac{1}{\sqrt{5}} F_{BD} + \frac{1}{\sqrt{5}} (7.8262 \text{ kN}) - \frac{1}{\sqrt{2}} F_{BC} - 2 \text{ kN} = 0$ 

$$F_{RO} - 1.58114 F_{RC} = -3.3541 \tag{2}$$

Multiply (1) by 2 and add (2):

$$3F_{BD} = -19.0065$$
  
 $F_{BD} = -6.3355 \text{ kN}$ 

 $F_{BD} = 6.34 \text{ kN} \ C \blacktriangleleft$ 

Subtract (2) from (1):

$$2.37111F_{BC} = -4.4721$$
  
 $F_{BC} = -1.8861 \text{ kN}$ 

 $F_{RC} = 1.886 \, \text{kN} \cdot C \blacktriangleleft$ 

Free body: Joint C

+ 
$$\Sigma F_y = 0$$
:  $\frac{2}{\sqrt{5}} F_{CD} - \frac{1}{\sqrt{2}} (1.8861 \text{ kN}) = 0$   
 $F_{CD} = +1.4911 \text{ kN}$ 

 $F_{CD} = 1.491 \,\text{kN}$  T

$$F_{CD} = +1.4911 \,\text{kN} \qquad F_{CD} = 1.4911 \,\text$$

 $F_{CE} = 5.00 \, \text{kN} \cdot T$ 

Free body: Joint D

$$\pm \Sigma F_x = 0$$
:  $\frac{2}{\sqrt{5}} F_{DF} + \frac{1}{\sqrt{2}} F_{DE} + \frac{2}{\sqrt{5}} (6.3355 \text{ kN}) - \frac{1}{\sqrt{5}} (1.4911 \text{ kN}) = 0$ 

or

$$F_{DF} + 0.79057 F_{DE} = -5.5900 \,\text{kN} \tag{1}$$

+ 
$$\Sigma F_y = 0$$
:  $\frac{1}{\sqrt{5}} F_{DF} - \frac{1}{\sqrt{2}} F_{DE} + \frac{1}{\sqrt{5}} (6.3355 \text{ kN}) - \frac{2}{\sqrt{5}} (1.4911 \text{ kN}) - 2 \text{ kN} = 0$ 

$$F_{DF} - 0.79057 F_{DE} = -1.1188 \text{ kN (2)}$$

Add (1) and (2):

$$2F_{DF} = -6.7088 \text{ kN}$$
  
 $F_{DF} = -3.3544 \text{ kN}$ 

$$F_{\rm DE} = 3.35 \, \rm kN \ C$$

Subtract (2) from (1):

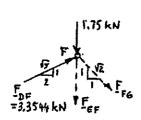
$$1.58114F_{DE} = -4.4712 \text{ kN}$$

$$F_{\text{max}} = 2.83 \text{ kN} \cdot C \blacktriangleleft$$

 $F_{DF} = 2.83 \text{ kN} \cdot C \blacktriangleleft$  $F_{DE} = -2.8278 \text{ kN}$ 

# PROBLEM 6.165 (Continued)

Free body: Joint F



$$\pm \Sigma F_x = 0$$
:  $\frac{1}{\sqrt{2}} F_{FG} + \frac{2}{\sqrt{5}} (3.3544 \text{ kN}) = 0$ 

$$F_{FG} = -4.243 \text{ kN}$$

$$F_{FG} = 4.24 \text{ kN}$$
 C

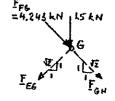
$$F_{FG} = -4.243 \text{ kN}$$
  $F_{FG} = 4.24 \text{ kN}$   
+  $\sum F_y = 0$ :  $-F_{EF} - 1.75 \text{ kN} + \frac{1}{\sqrt{5}} (3.3544 \text{ kN}) - \frac{1}{\sqrt{2}} (-4.243 \text{ kN}) = 0$ 

$$F_{EF} = 2.750 \text{ kN}$$

$$F_{EF} = 2.75 \text{ kN}$$
  $T \blacktriangleleft$ 

Free body: Joint G

$$\pm \Sigma F_x = 0$$
:  $\frac{1}{\sqrt{2}} F_{GH} - \frac{1}{\sqrt{2}} F_{EG} + \frac{1}{\sqrt{2}} (4.243 \text{ kN}) = 0$ 



(1)

or:

$$F_{GH} - F_{EG} = -4.243 \text{ kN}$$

+ 
$$\Sigma F_y = 0$$
:  $-\frac{1}{\sqrt{2}} F_{GH} - \frac{1}{\sqrt{2}} F_{EG} - \frac{1}{\sqrt{2}} (4.243 \text{ kN}) - 1.5 \text{ kN} = 0$ 

or:

$$F_{GH} + F_{EG} = -6.364 \text{ kN}$$
 (2)

Add (1) and (2):

$$2F_{GH} = -10.607$$

$$F_{GH} = -5.303$$

$$F_{CH} = 5.30 \,\text{kN} \cdot C \,\blacktriangleleft$$

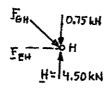
Subtract (1) from (2):

$$2F_{EG} = -2.121 \,\mathrm{kN}$$

$$F_{EG} = -1.0605 \text{ kN}$$

$$F_{VG} = 1.061 \,\text{kN}$$
 C

Free body: Joint H



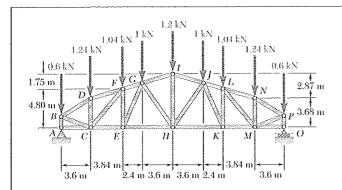
$$\frac{F_{EH}}{1} = \frac{3.75 \text{ kN}}{1}$$

$$F_{EH} = 3.75 \text{ kN}$$
  $T \blacktriangleleft$ 

We can also write:

$$\frac{F_{GH}}{\sqrt{2}} = \frac{3.75 \text{ kN}}{1}$$

$$F_{GH} = 5.30 \text{ kN}$$
 C (Checks)



The truss shown was designed to support the roof of a food market. For the given loading, determine the force in members FG, EG, and EH.

### **SOLUTION**

Reactions at supports. Because of the symmetry of the loading

$$A_x = 0$$
,  $A_y = O = \frac{1}{2}$  (Total load)

A = O = 4.48 kN

We pass a section through members FG, EG, and EH, and use the Free body shown.

Slope 
$$FG = \text{Slope } FI = \frac{1.75 \text{ m}}{6 \text{ m}}$$

$$E = \frac{5.50 \text{ m}}{2.4}$$
Slope  $EG = \frac{5.50 \text{ m}}{2.4 \text{ m}}$ 

Slope 
$$EG = \frac{5.50 \text{ m}}{2.4 \text{ m}}$$

+) 
$$\Sigma M_E = 0$$
:  $(0.6 \text{ kN})(7.44 \text{ m}) + (1.24 \text{ kN})(3.84 \text{ m})$   
 $-(4.48 \text{ kN})(7.44 \text{ m})$   
 $-\left(\frac{6}{6.25}F_{FG}\right)(4.80 \text{ m}) = 0$ 

$$F_{FG} = 5.23 \text{ kN} \quad C \blacktriangleleft$$

+) 
$$\Sigma M_G = 0$$
:  $F_{EH}$  (5.50 m) + (0.6 kN)(9.84 m)  
+(1.24 kN)(6.24 m) + (1.04 kN)(2.4 m)  
-(4.48 kN)(9.84 m) = 0

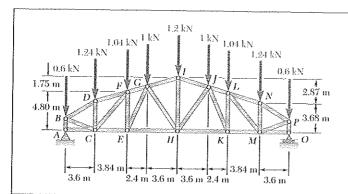
$$F_{EH} = 5.08 \text{ kN}$$
 T

+ 
$$\Sigma F_y = 0$$
:  $\frac{5.50}{6.001} F_{EG} + \frac{1.75}{6.25} (-5.231 \text{ kN}) + 4.48 \text{ kN} - 0.6 \text{ kN} - 1.24 \text{ kN} - 1.04 \text{ kN} = 0$ 

 $F_{FG} = -5.231 \text{ kN}$ 

$$F_{EG} = -0.1476 \text{ kN}$$

$$F_{EG} = 0.1476 \text{ kN}$$
 C



The truss shown was designed to support the roof of a food market. For the given loading, determine the force in members *KM*, *LM*, and *LN*.

# SOLUTION

Because of symmetry of loading,  $\mathbf{O} = \frac{1}{2}(\text{Load})$ 

O = 4.48 kN

We pass a section through KM, LM, LN, and use free body shown

$$+(4.48 \text{ kN} - 0.6 \text{ kN})(3.6 \text{ m}) = 0$$
  
 $F_{LN} = -3.954 \text{ kN}$ 

$$F_{tN} = 3.95 \text{ kN} \ C \blacktriangleleft$$

+)
$$\Sigma M_L = 0$$
:  $-F_{KM}$  (4.80 m) - (1.24 kN)(3.84 m)  
+ (4.48 kN - 0.6 kN)(7.44 m) = 0

+) $\Sigma M_M = 0$ :  $\left(\frac{3.84}{4}F_{LN}\right)(3.68 \text{ m})$ 

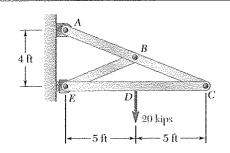
$$F_{KM} = +5.022 \text{ kN}$$

$$F_{KM} = 5.02 \text{ kN}$$
  $T \blacktriangleleft$ 

$$+\Sigma F_y = 0$$
:  $\frac{4.80}{6.147}F_{LM} + \frac{1.12}{4}(-3.954 \text{ kN}) - 1.24 \text{ kN} - 0.6 \text{ kN} + 4.48 \text{ kN} = 0$ 

$$F_{LM} = -1.963 \text{ kN}$$

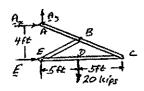
$$F_{LM} = 1.963 \text{ kN} \quad C \blacktriangleleft$$



For the frame and loading shown, determine the components of all forces acting on member ABC.

#### **SOLUTION**

Free body: Entire frame



+)
$$\Sigma M_E = 0$$
:  $-A_x(4) - (20 \text{ kips})(5) = 0$ 

$$A_r = -25 \text{ kips}, \qquad A_x = 25.0 \text{ kips} \longleftarrow \blacktriangleleft$$

$$+ \sum F_y = 0$$
:  $A_y - 20 \text{ kips} = 0$ 

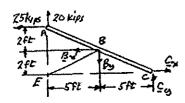
$$A_v = 20 \text{ kips}$$

$$A_y = 20.0 \text{ kips}$$

Free body: Member ABC

*Note:* BE is a two-force member, thus **B** is directed along line BE and  $B_y = \frac{2}{5}B_x$ 

+) $\Sigma M_C = 0$ : (25 kips)(4 ft) - (20 kips)(10 ft) +  $B_x$ (2 ft) +  $B_y$ (5 ft) = 0  $-100 \text{ kip} \cdot \text{ft} + B_x(2 \text{ ft}) + \frac{2}{5}B_x(5 \text{ ft}) = 0$ 



$$B_r = 25 \text{ kips}$$

$$\mathbf{B}_{x} = 25.0 \text{ kips} \blacktriangleleft$$

$$B_y = \frac{2}{5}(B_x) = \frac{2}{5}(25) = 10 \text{ kips}$$
  $\mathbf{B}_y = 10.00 \text{ kips} \downarrow \blacktriangleleft$ 

$$\mathbf{B}_y = 10.00 \text{ kips} \downarrow \blacktriangleleft$$

$$\pm \Sigma F_y = 0$$
:  $C_y - 25 \text{ kips} - 25 \text{ kips} = 0$ 

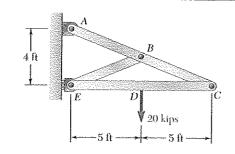
$$C_x = 50 \text{ kips}$$

$$C_x = 50 \text{ kips}$$
  $C_x = 50.0 \text{ kips} \longrightarrow \blacktriangleleft$ 

$$+ \sum F_y = 0$$
:  $C_y + 20 \text{ kips} - 10 \text{ kips} = 0$ 

$$C_y = -10 \text{ kips}$$

$$C_y = 10.00 \text{ kips}$$

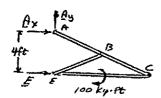


Solve Problem 6.168 assuming that the 20-kip load is replaced by a clockwise couple of magnitude 100 kip · ft applied to member EDC at Point D.

PROBLEM 6.168 For the frame and loading shown, determine the components of all forces acting on member ABC.

# **SOLUTION**

Free body: Entire frame



$$+ \sum F_{\nu} = 0: \quad A_{\nu} = 0$$

+) 
$$\Sigma M_E = 0$$
:  $-A_x(4 \text{ ft}) - 100 \text{ kip} \cdot \text{ft} = 0$ 

$$A_x = -25 \text{ kips}$$

$$A_x = 25.0 \text{ kips} -$$

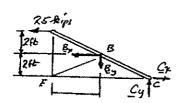
$$A = 25.0 \text{ kips} \blacktriangleleft \blacksquare$$

Free Body: Member ABC

*Note:* BE is a two-force member, thus **B** is directed along line BE and  $B_y = \frac{2}{5}B_x$ 

+) 
$$\Sigma M_C = 0$$
:  $(25 \text{ kips})(4 \text{ ft}) + B_x(2 \text{ ft}) + B_y(5 \text{ ft}) = 0$   

$$100 \text{ kip} \cdot \text{ft} + B_x(2 \text{ ft}) + \frac{2}{5}B_x(5 \text{ ft}) = 0$$



$$B_{\rm v} = -25 \text{ kips}$$

$$\mathbf{B}_x = 25.0 \text{ kips} \longrightarrow \blacktriangleleft$$

$$B_y = \frac{2}{5}B_x = \frac{2}{5}(-25) = -10 \text{ kips};$$
  $\mathbf{B}_y = 10.00 \text{ kips}$ 

$$\mathbf{B}_y = 10.00 \text{ kips}$$

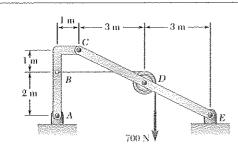
$$\pm \Sigma F_x = 0$$
:  $-25 \text{ kips} + 25 \text{ kips} + C_x = 0$   $C_x = 0$ 

$$+\int \Sigma F_{\nu} = 0$$
:  $+10 \text{ kips} + C_{\nu} = 0$ 

$$C_y = -10 \text{ kips}$$

$$C_v = 10 \text{ kips}$$

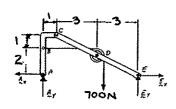
$$C = 10.00 \text{ kips}$$



Knowing that the pulley has a radius of 0.5 m, determine the components of the reactions at A and E.

#### SOLUTION

#### FBD Frame:



$$\sum M_A = 0$$
:  $(7 \text{ m})E_y - (4.5 \text{ m})(700 \text{ N}) = 0$   $E_y = 450 \text{ N}$ 

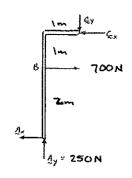
$$\mathbf{E}_{_{\mathrm{P}}} = 450 \,\mathrm{N}^{\dagger} \,\blacktriangleleft$$

$$\Sigma F_{\nu} = 0$$
:  $A_{\nu} - 700 \text{ N} + 450 \text{ N} = 0$ 

$$A_v = 250 \text{ N}^{\dagger}$$

$$-- \Sigma F_x = 0: \quad A_x - E_x = 0 \qquad A_x = E_x$$

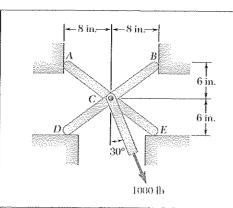
Dimensions in m



$$\sum M_C = 0$$
:  $(1 \text{ m})(700 \text{ N}) - (1 \text{ m})(250 \text{ N}) - (3 \text{ m}) A_x = 0$ 

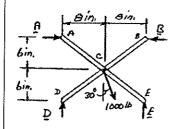
$$A_x = 150.0 \text{ N} - \blacktriangleleft$$

so 
$$\mathbf{E}_x = 150.0 \text{ N} \longrightarrow \blacktriangleleft$$



For the frame and loading shown, determine the reactions at A, B, D, and E. Assume that the surface at each support is frictionless.

### SOLUTION



### Free body: Entire frame

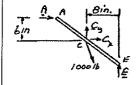
## 
$$\Sigma F_x = 0$$
:  $A - B + (1000 \text{ lb}) \sin 30^\circ = 0$ 

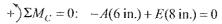
$$A - B + 500 = 0$$

$$+ | \Sigma F_y = 0$$
:  $D + E - (1000 \text{ lb}) \cos 30^\circ = 0$ 

$$D + E - 866.03 = 0$$
(2)

# Free body: Member ACE





$$E = \frac{3}{4}A\tag{3}$$

(4)

### Free body: Member BCD

+) 
$$\Sigma M_C = 0$$
:  $-D(8 \text{ in.}) + B(6 \text{ in.}) = 0$   
$$D = \frac{3}{4}B$$

Substitute E and D from (3) and (4) into (2):

$$-\frac{3}{4}A + \frac{3}{4}B - 866.06 = 0$$

$$A + B - 1154.71 = 0$$
(5)

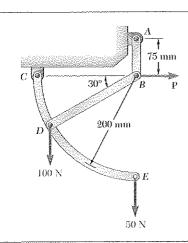
$$(1) A - B + 500 = 0 (6)$$

(5)+(6) 
$$2A-654.71=0$$
  $A=327.4 \text{ lb}$   $A=327 \text{ lb}$ 

(5) - (6) 
$$2B - 1654.71 = 0$$
  $B = 827.4 \text{ lb}$   $\mathbf{B} = 827 \text{ lb} \leftarrow \blacksquare$ 

(4) 
$$D = \frac{3}{4}(827.4)$$
  $D = 620.5 \text{ lb}$   $\mathbf{D} = 621 \text{ lb}$ 

(3) 
$$E = \frac{3}{4}(327.4)$$
  $E = 245.5 \text{ lb}$   $\mathbf{E} = 246 \text{ lb}^{\dagger} \blacktriangleleft$ 



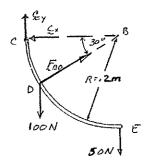
For the system and loading shown, determine (a) the force P required for equilibrium, (b) the corresponding force in member BD, (c) the corresponding reaction at C.

#### **SOLUTION**

**Member FBDs:** 

FBD 1:

I:



$$\sum M_C = 0$$
:  $R(F_{BD} \sin 30^\circ) - [R(1 - \cos 30^\circ)](100 \text{ N}) - R(50 \text{ N}) = 0$ 

$$F_{RD} = 126.795 \text{ N}$$

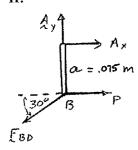
 $\Sigma F_y = 0$ :  $C_y + (126.795 \text{ N}) \sin 30^\circ - 100 \text{ N} - 50 \text{ N} = 0$ 

$$F_{BD} = 126.795 \text{ N}$$
 (b)  $F_{BD} = 126.8 \text{ N}$  T

$$\rightarrow \Sigma F_x = 0$$
:  $-C_x + (126.795 \text{ N})\cos 30^\circ = 0$   $C_x = 109.808 \text{ N} \leftarrow$ 

$$C_v = 86.603 \text{ N}^{\dagger}$$
 (c) so  $C = 139.8 \text{ N} \ge 38.3^{\circ} \blacktriangleleft$ 

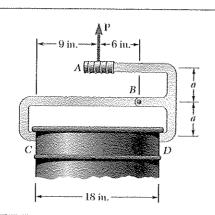
H:



#### FBD II:

$$\sum M_A = 0$$
:  $aP - a[(126.795 \text{ N})\cos 30^\circ] = 0$  (a)  $P = 109.8 \text{ N} \rightarrow \blacksquare$ 

(a) 
$$P = 109.8 N \rightarrow$$



A small barrel weighing 60 lb is lifted by a pair of tongs as shown. Knowing that a = 5 in., determine the forces exerted at B and D on tong ABD.

#### **SOLUTION**

We note that BC is a two-force member.

Free body: Tong ABD

$$\frac{B_x}{15} = \frac{B_y}{5} \quad B_x = 3B_y$$

+)
$$\Sigma M_D = 0$$
:  $B_y(3 \text{ in.}) + 3B_y(5 \text{ in.}) - (60 \text{ lb})(9 \text{ in.}) = 0$ 



$$B_x = 3B_y$$
:  $B_x = 90 \text{ lb}$ 

$$\pm \Sigma F_x = 0$$
:  $-90 \text{ lb} + D_x = 0$ 

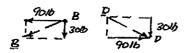
+ 
$$\Sigma F_y = 0$$
: 60 lb - 30 lb -  $D_y = 0$ 

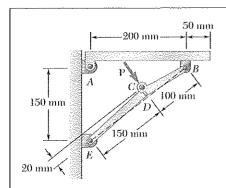
$$\mathbf{D}_x = 90 \text{ lb} \longrightarrow$$

$$\mathbf{D}_{y} = 30 \text{ lb}$$

$$B = 94.9 \text{ lb} > 18.43^{\circ} \blacktriangleleft$$

$$D = 94.9 \text{ lb} \le 18.43^{\circ} \blacktriangleleft$$

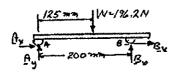




A 20-kg shelf is held horizontally by a self-locking brace that consists of two Parts EDC and CDB hinged at C and bearing against each other at D. Determine the force  $\mathbf{P}$  required to release the brace.

#### SOLUTION

Free body: Shelf



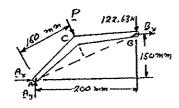
$$W = (20 \text{ kg})(9.81 \text{ m/s}^2) = 196.2 \text{ N}$$

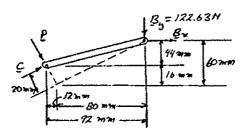
+)
$$\Sigma M_A = 0$$
:  $B_y (200 \text{ mm}) - (196.2 \text{ N})(125 \text{ mm}) = 0$ 

$$B_v = 122.63 \text{ N}$$

Free body: Portion ACB

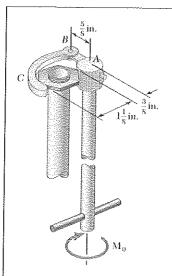
+)
$$\Sigma M_A = 0$$
:  $-B_x (150 \text{ mm}) - P(150 \text{ mm}) - (122.63 \text{ N})(200 \text{ mm}) = 0$   
 $B_x = -163.5 - P$  (1)





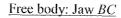
+) 
$$\Sigma M_C = 0$$
: +(122.63 N)(92 mm) +  $B_x$ (94 mm) = 0  
+(122.63 N)(92 mm) + (-163.5 -  $P$ )(44 mm) = 0

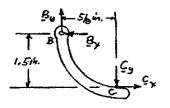
P = 92.9 N P = 92.9 N



The specialized plumbing wrench shown is used in confined areas (e.g., under a basin or sink). It consists essentially of a jaw BC pinned at B to a long rod. Knowing that the forces exerted on the nut are equivalent to a clockwise (when viewed from above) couple of magnitude 135 lb · in., determine (a) the magnitude of the force exerted by pin B on jaw BC, (b) the couple  $M_0$  that is applied to the wrench.

### **SOLUTION**



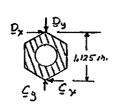


This is a two-force member

$$\frac{C_y}{1.5 \text{ in.}} = \frac{C_x}{\frac{5}{8} \text{ in.}} \quad C_y = 2.4 C_x$$

$$\Sigma F_x = 0: \quad B_x = C_x$$
(1)

$$\Sigma F_y = 0$$
:  $B_y = C_y = 2.4 C_x$  (2)



<u>Free body: Nut</u>  $\Sigma F_x = 0$ :  $C_x = D_x$ 

$$\Sigma M = 135 \text{ lb} \cdot \text{in}.$$

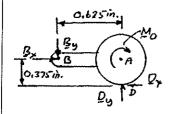
$$C_x(1.125 \text{ in.}) = 135 \text{ lb} \cdot \text{in.}$$

$$C_x = 120 \text{ lb}$$

(a) Eq. (1): 
$$B_r = C_r = 120 \text{ lb}$$

Eq. (2): 
$$B_y = C_y = 2.4(120 \text{ lb}) = 288 \text{ lb}$$

$$B = \left(B_x^2 + B_y^2\right)^{1/2} = (120^2 + 288^2)^{1/2}$$
 B = 312 lb



(b) Free body: Rod

+)
$$\Sigma M_D = 0$$
:  $-M_0 + B_y (0.625 \text{ in.}) - B_x (0.375 \text{ in.}) = 0$   
- $M_0 + (288)(0.625) - (120)(0.375) = 0$ 

 $M_0 = 135.0 \text{ lb} \cdot \text{in.}$ 

